

Does CCS reduce power generation flexibility? A dynamic study of combined cycles with post-combustion CO₂ capture

Jairo Rúa^a, Mai Bui^{b,c}, Lars O. Nord^a, Niall Mac Dowell^{b,c,*}

^a*Department of Energy and Process Engineering, Norwegian University of Science and Technology, Trondheim, Norway*

^b*Centre for Process Systems Engineering, Imperial College London, London, UK*

^c*Centre for Environmental Policy, Imperial College London, London, UK*

Abstract

To date, the deployment, integration, and utilisation of intermittent renewable energy sources, such as wind and solar power, in the global energy system has been the cornerstone of efforts to combat climate change. At the same time, it is recognised that renewable power represents only one element of the portfolio of technologies that will be required to deliver a technically feasible and financially viable energy system. In this context, carbon capture and storage (CCS) is understood to play a uniquely important role, providing significant value through flexible operation. It is therefore of vital importance that CCS technology can operate synergistically with intermittent renewable power sources, and consequently ensuring that CCS does not inhibit the flexible and dispatchable nature of thermal power plants. This work analyses the intrinsic dynamic performance of the power and CO₂ capture plants independently and as an integrated system. Since the power plant represents the fast dynamics of the system and the steam extraction is the main point of integration between the CO₂ capture and power plants, disturbances with fast dynamics are imposed on the steam extraction valve during steady state and dynamic operation of a natural gas combined cycle (NGCC) to study the effects of the integration on power generation capacity. The results demonstrate that the integration of liquid-absorbent based post-combustion CO₂ capture has negligible impact on the power generation dynamics of the NGCC.

Keywords: combined cycle gas turbine (CCGT), post-combustion CO₂ capture, amine absorption process, monoethanolamine (MEA), flexible operation, dynamic operation, dynamic modelling

1. Introduction

Climate change mitigation is one of the greatest challenges in the 21st century. Anthropogenic greenhouse gas emissions since the industrial revolution have resulted in increasing temperatures and changes in natural and human ecosystems (IPCC, 2014). Thus, a deep decarbonization of all sectors is necessary to meet the target of not exceeding the 1.5°C temperature increase respect to pre-industrial levels (IPCC, 2018).

Among the different possibilities available to reduce the greenhouse gas emissions, carbon capture and storage (CCS) is a uniquely important technology for mitigating the CO₂ emissions associated with the energy sector and industry (IPCC, 2005, 2014). These two sectors account for more than 50% of the total global greenhouse gas emissions (IPCC, 2014; IEA, 2018a).

Renewable energy sources will also contribute significantly to reducing CO₂ emissions (IEA, 2018b). Future energy systems are expected to be characterised by a high penetration of intermittent renewable sources. This will result in additional costs associated with load balancing, additional firming capacity,

*Corresponding author.

Email addresses: jairo.r.pazos@ntnu.no (Jairo Rúa), m.bui@imperial.ac.uk (Mai Bui), lars.nord@ntnu.no (Lars O. Nord), niall@imperial.ac.uk (Niall Mac Dowell)

28 energy storage, and interconnection capacity
29 (Heuberger et al., 2017a,b).

30 Flexible dispatchable energy generation
31 technologies such as thermal power with CCS
32 offer a cost effective way to balance this inter-
33 mittency (Heuberger et al., 2016; Kondziella
34 and Bruckner, 2016; Montañés et al., 2016;
35 Mac Dowell and Staffell, 2016). Consequently,
36 thermal power plants will be exposed to cy-
37 cling operation and more frequent start-ups
38 and shut-downs (Eser et al., 2017; González-
39 Salazar et al., 2017). Thus, to deploy CCS
40 technology in a power market dominated by
41 the high variability of renewable energy, it is
42 necessary to prove its adequacy for flexible op-
43 eration (Adams and Mac Dowell, 2016).

44 Post-combustion CO₂ capture is arguably
45 the most mature CCS technology (IPCC, 2005;
46 Bui et al., 2018a). Therefore, deep under-
47 standing of the dynamic performance of these
48 capture plants integrated with thermal power
49 plants is essential. Dynamic modelling and
50 simulation remains the primary medium to
51 study the interaction of these systems under
52 transient operation due to the lack of full-
53 scale experience (Bui et al., 2014, 2018a). De-
54 veloping further detailed insight into the pro-
55 cess dynamics could help improve the accu-
56 racy and robustness of dynamic process control
57 and scheduling during flexible operation, plant
58 start-up and shut-down.

59 The development of dynamic CO₂ capture
60 models was extensively reviewed by Bui et al.
61 (2014, 2018b). Whilst the vast majority of re-
62 search on flexible operation of CCS focuses on
63 modelling the dynamics of the capture plant,
64 there are relatively few studies that model
65 the integrated system with a thermal power
66 plant (Lawal et al., 2012; Mac Dowell and
67 Shah, 2013, 2015; Wellner et al., 2016; He
68 and Ricardez-Sandoval, 2016; Mechleri et al.,
69 2017a,b; Garðarsdóttir et al., 2017; Montañés
70 et al., 2017b).

71 Lawal et al. (2012) studied the dynamic in-
72 teraction between a coal-fired power plant and
73 a post-combustion capture plant with MEA,
74 and showed how tight control (*i.e.*, rapidly re-
75 sponds to minimise deviation between the con-
76 trolled variable and its set-point) on the cap-
77 ture plant may interfere with the power output
78 of the power plant. For a similar integrated

79 system, Garðarsdóttir et al. (2017) found that
80 power generation settling times are essentially
81 independent of the integration of the capture
82 plant. However, inadequate control strategies
83 may result in unnecessary longer stabilization
84 times. Both studies concluded that the dy-
85 namics of the capture plant are significantly
86 slower than the power plant, leading to longer
87 settling times in the absence of adequate con-
88 trol structures, which may affect power plant
89 performance. Retrofitted coal power plants ex-
90 hibit the same transient behaviour and the in-
91 tegration with the capture plant acts as steam
92 storage that can be rapidly adjusted to meet
93 peak power demands through the manipulation
94 of the extraction valve (Wellner et al., 2016).
95 Mac Dowell and Shah (2013, 2015), and Mech-
96 leri et al. (2017a,b) also developed integrated
97 systems of coal-fired power plants with post-
98 combustion capture plants to study the eco-
99 nomic performance during flexible operation
100 accounting for variations in the electricity mar-
101 ket, although the dynamic interaction was not
102 studied.

103 Commercial natural gas combined cycles in-
104 tegrated with full-scale post-combustion cap-
105 ture plants show similar transient perfor-
106 mance. He and Ricardez-Sandoval (2016) and
107 Montañés et al. (2017b) proved the faster dy-
108 namics of the power plant compared to the cap-
109 ture plant, which resulted in slow oscillations in
110 the longer time-scales as a consequence of the
111 interaction between both plants. The analysis
112 of varying inputs in open-loop in the capture
113 plant also showed the benefits that may be ob-
114 tained from close-loop control and simultane-
115 ous scheduling of the power and CO₂ capture
116 plant (He and Ricardez-Sandoval, 2016). Fur-
117 ther, evaluation of several control structures in
118 the capture plant showed that different control
119 couplings may lead to distinct long term dy-
120 namics in the low-pressure steam turbine. Nev-
121 ertheless, the short-term transient behaviour of
122 the natural gas combined cycle is not affected
123 as a result of the slow dynamic response of
124 the post-combustion capture plant (Montañés
125 et al., 2017b).

126 These studies on the full-scale transient per-
127 formance of integrated systems showed that
128 slow dynamic interactions between the ther-
129 mal power plant with the post-combustion CO₂

capture plant do not affect notably their power production capacity, albeit the stabilization time is affected by the slow response of the capture plant. However, the dynamics of power generation are determined by the transient behaviour of the steam cycle, that is, by the fast dynamics of the integrated system. The decoupling of power generation capacity from the CCS process has the potential to significantly enhance the economic efficiency and the technical performance. Therefore, rapid dynamic disturbances must be analysed in order to determine whether the CO₂ capture plant limits the electricity production capabilities of the thermal power plant.

The aim of this study is to investigate the extent to which fast disturbances in the steam extraction affect the power generation capability of the integrated system. Building on previous work, a thorough analysis of the dynamics governing the thermal power plant, the post-combustion capture plant and the integrated system is included in [Section 2](#) to understand the physical mechanisms dictating their transient operation. [Section 3](#) describes the modelling of the natural gas combined cycle integrated with the post-combustion CO₂ capture plant and the special power generation characteristic of this type of power generation systems. Results are presented and discussed in [Section 4](#), and the conclusions are presented in [Section 5](#).

2. Dynamic Analysis of Thermal Power Plants Integrated with CCS

Thermal power plants and post-combustion capture plants exhibit distinct dynamic behaviour. This section identifies and evaluates the process and dominant dynamics that significantly influence thermal power plants integrated with post-combustion CO₂ capture plants, including passive elements that contribute to the dynamics but are not the main source.

2.1. Thermal Power Plants

As post-combustion capture plants are a cost effective technology to remove CO₂ from large-emission sources, they are a suitable complement for heavy-duty natural gas combined cycles and coal- and biomass-fired power plants

([IPCC, 2005](#)). Natural gas combined cycles rely on gas turbines to control and produce most of the power and a steam cycle that acts as a passive element, which utilizes the energy contained in the exhaust gas to generate extra power. In contrast, power generation from solid fuels, namely coal and biomass, using sub-critical and supercritical power plant technology, produce electricity solely via the steam cycle, which is driven by the combustion process in the furnace.

[Fig. 1](#) shows the different operation range of each thermal power plant. The minimum load of modern gas turbine is limited to 40% of its full load owing to the combustion stability of the fuel and the associated emissions ([Alobaid et al., 2017](#); [Eser et al., 2017](#)). Therefore, since the gas turbine accounts for a large share of the total power capacity of natural gas combined cycles, this type of power plants cannot reduce its power generation below this limit. Conversely, coal and biomass power plants are not restricted by a gas turbine; and their minimum compliant load is around 25% of their full load ([Hentschel et al., 2016](#)). This broader operation range enhances the utilisation of coal and biomass power plants as spinning reserves.

A common characteristic of all thermal power plants is the heat transfer in the steam generator between the combustion gases and the working fluid of the Rankine cycle. In this equipment, the combustion gases flow in a counter-current or cross-flow manner through several tube bundles where energy is transferred progressively to produce the superheated steam that drives the steam turbines. Steam generators are bulky equipment whose enormous mass of metal stores large amounts of energy due to its heat capacity. This leads to slow responses in the steam cycle and hence the power generation in the steam turbines. Thus, steam generators are the main limitation during the transient operation of thermal power plants and consequently define their dominant dynamics ([Alobaid et al., 2017](#)).

Two different time-scales dictate the dynamic operation of natural gas combined cycles. Modern gas turbines are fast components that can have load ramps up to 15% per minute and whose dominant dynamics are in the order of seconds ([Hentschel et al., 2016](#)). Steam

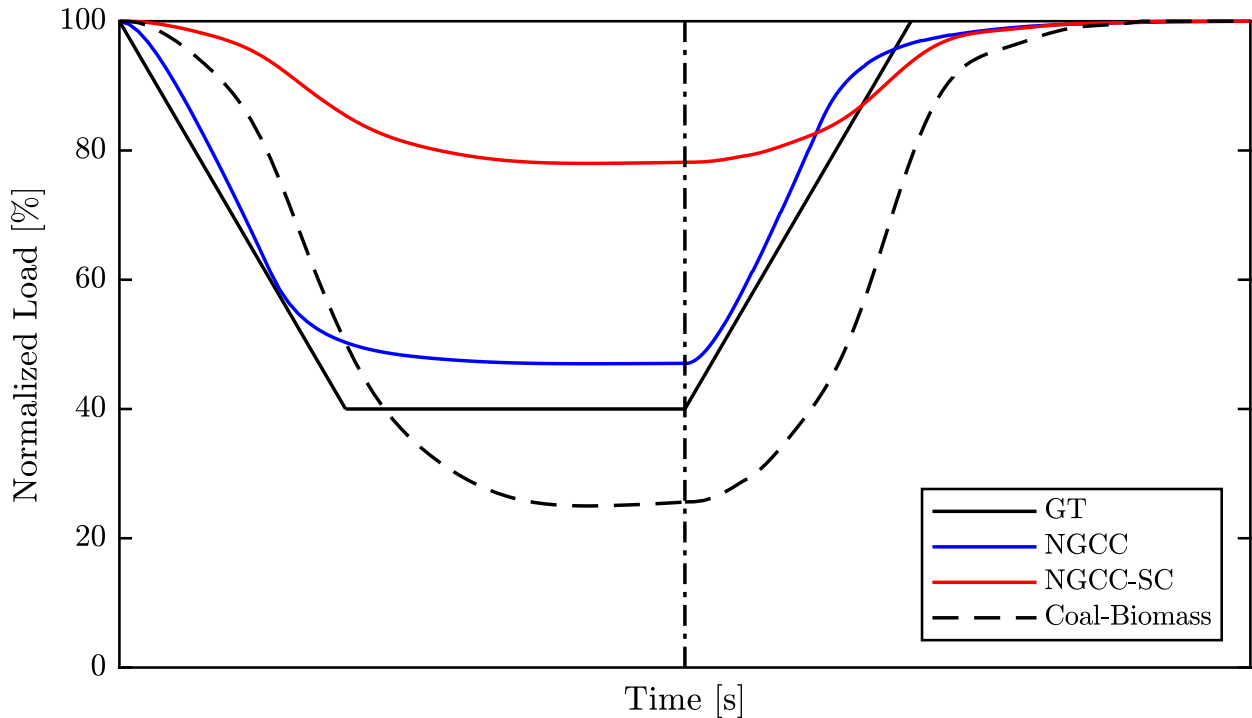


Figure 1: Generic dynamic behaviour of different thermal power plants of similar size. Maximum and minimum loads and power generation shares depend on the power plant design. The vertical line indicates the increasing load dynamic behaviour. The nomenclature is as follows. GT: Gas Turbine, NGCC: Natural Gas Combined Cycle, SC: Steam Cycle.

230 cycles are limited by the heat capacitance of 256
 231 the steam generator and thus their dominant 257
 232 dynamics are on the order of minutes. Fig. 1 258
 233 represents the general dynamic behaviour of a 259
 234 natural gas combined cycle. The gas turbine 260
 235 drives the transient operation of the NGCC by 261
 236 changing its load, whilst the steam cycle deter- 262
 237 mines the time required to reach steady-state 263
 238 (Kehlhofer et al., 2009). Nevertheless, natural 264
 239 gas combined cycles are able to meet the power 265
 240 demand before the steam cycle reaches steady- 266
 241 state by under- or over-shooting the gas turbine 267
 242 (Rúa et al., 2020). This unique ability of the 268
 243 gas turbine compensates for the slow transient 269
 244 performance of the steam cycle, enhancing the 270
 245 adequacy of NGCC for flexible operation.

246 Coal and biomass power plants do not have 271
 247 a gas turbine to control the power generation, 272
 248 thus governor valves are required at the inlet 273
 249 of the steam turbine to guarantee tight power 274
 250 control during transient operation. Fuel con- 275
 251 sumption is adjusted according to power de- 276
 252 mand to regulate the part-load performance, 277
 253 but this strategy cannot be applied in the time- 278
 254 scale of seconds owing to the heat capacitance 279
 255 of the steam generator and the slow response 280

of the steam cycle (see Fig. 1). Consequently, the slow dynamics of the steam cycle dominate the transient operation of coal and biomass power plants, making them slower than modern NGCC and less suitable for flexible operation (Eser et al., 2017; González-Salazar et al., 2017).

2.2. Post-Combustion Capture Plants

Capture plants are passive systems whose operation is determined by the conditions of the gas to be treated and the steam available for the reboiler. From a dynamic operation perspective, the gas is a disturbance to which the capture plant must adapt to, whereas the steam is considered a manipulated variable. The stripper condenser pressure is also a boundary condition of the capture plant, however, this is considered constant as it is rarely modified during dynamic operation.

In a post-combustion capture plant, the fastest units are the rotating machinery (*i.e.*, blowers, compressors and pumps), as they have almost negligible dynamics with time constants in the order of a few seconds. Thus, solvent flow rates stabilize within a few minutes, de-

281 pending on the size of the plant and the mag- 332
282 nitude of the flow change (Flø et al., 2016; 333
283 Montañés et al., 2018). Heat exchangers and 334
284 piping lead to transport delays that do not af- 335
285 fect the nature of the dynamics. Conversely, 336
286 large vessels such as absorber and stripper 337
287 sumps, reboiler hotwells or buffer tanks in- 338
288 troduce significant inertia, which buffers and 339
289 smooths the overall dynamic behaviour of the 340
290 capture plant (Flø et al., 2016). Liquid hold-up 341
291 in the absorber and stripper also contributes to 342
292 this buffering effect, however, the effect on the 343
293 solvent flow rate dynamics is small relative to 344
294 that of sumps, storage tanks, *etc.* Therefore, 345
295 the dynamics of the post-combustion capture 346
296 plant are not governed by the mass balance 347
297 but by the total volume of solvent, the volu- 348
298 metric capacity of the plant, and the solvent 349
299 circulation time. 350

300 Chemical and thermal equilibrium in the ab- 351
301 sorber and stripper columns also affect the 352
302 transient behaviour but has a minor influence 353
303 on the stabilisation time of the capture plant. 354
304 (Flø et al., 2016; Tait et al., 2016; Montañés 355
305 et al., 2017a, 2018). During open-loop oper- 356
306 ation, changes in flue gas flow rate primarily 357
307 impacts the absorption section, affecting the 358
308 CO₂ capture rate and shifting the temperature 359
309 profile as a result of the difference in released 360
310 energy from the exothermic chemical reactions 361
311 (Kvamsdal and Rochelle, 2008; Bui et al., 2016; 362
312 Tait et al., 2016; Montañés et al., 2018). Both 363
313 changes are dominated by the chemical and 364
314 thermal inertia within the absorber as the sta- 365
315 bilization times of the absorber temperature 366
316 profile and CO₂ capture rate are larger than 367
317 the rise time of the flue gas flow rate (Montañés 368
318 et al., 2018). 369

319 For a given solvent flow rate, moderate 370
320 changes to the exhaust gas flow rate have a mi- 371
321 nor effect on the rich CO₂ loading of the solvent 372
322 (Lawal et al., 2010; Flø et al., 2016; Bui et al., 373
323 2016; Montañés et al., 2017a, 2018; Bui et al., 374
324 2018b). However, sufficiently large variations 375
325 in the feed gas CO₂ concentration or mass flow 376
326 rate may lead to more pronounced effects on 377
327 rich solvent loading. Changes in flue gas flow 378
328 rate only affect the absorption section and the 379
329 solvent loading, but the effect of these changes 380
330 on the overall stabilization time of the entire 381
331 capture plant is essentially negligible.

The steam flow rate to the reboiler is an im-
portant process parameter. Sufficiently large
changes to the steam flow rate will vary the
temperature in the reboiler, and consequently
the operating conditions of the stripper col-
umn. Assuming the other process conditions
remain constant or are not adequately adapted,
this would result in changes to the lean CO₂
loading exiting the stripper. (Lawal et al.,
2010; Garðarsdóttir et al., 2015; Flø et al.,
2016; Montañés et al., 2017a; Bui et al., 2020).
This change in lean loading affects the amount
of CO₂ the solvent can absorb, which in turn
influences the energy released during the ab-
sorption reaction, the absorber column temper-
ature profile and the CO₂ capture rate. These
operation changes are expected to result in dif-
ferent rich loadings, which will affect the strip-
per transient conditions (Lawal et al., 2010; Flø
et al., 2016; Bui et al., 2016; Montañés et al.,
2017a, 2018).

Slow and long dynamics can limit the rate
of transient behaviour and increase solvent cir-
culation time. There is a combination of fac-
tors that contribute to slow dynamics, these in-
clude (i) total volume of solvent stored or held-
up in the capture plant, (ii) size of the vessels
in the system which impacts residence time,
and (iii) transport delay introduced by the heat
exchangers and piping. There is also an ob-
servable inter-column interaction between the
stripper and absorber conditions. In a plant
with slow dynamics (*e.g.*, owing to larger to-
tal liquid hold-up), changes to the solvent flow
rate lead to slow variation of the rich and lean
solvent loading. Thus, the slow interaction be-
tween the absorber and stripper columns due
to the large liquid volumes (*e.g.*, long solvent
circulation time or slow transient behaviour) is
the main bottleneck, slowing the response time
during flexible operation of post-combustion
capture plants.

2.3. Thermal Power Plants Integrated with Post-Combustion Capture Plants

Several process configurations to integrate
the power and capture plants have been stud-
ied (Botero et al., 2009; Lucquiaud et al.,
2009; Jordal et al., 2012; Jonshagen et al.,
2012; Mac Dowell and Shah, 2014), with
steam extraction from the crossover between

382 the intermediate- and low-pressure (IP-LP) 433
383 steam turbines being the preferred option 434
384 (Lawal et al., 2012; Montañés et al., 2017b; 435
385 Garðarsdóttir et al., 2017). In this integra- 436
386 tion approach the steam extracted from the 437
387 steam turbine may be mixed with low-pressure 438
388 superheated steam in NGCC, and tempera- 439
389 ture is controlled by evaporative spray cooling 440
390 with pressurized water from the intermediate- 441
391 pressure economizer (Montañés et al., 2017b). 442
392 In contrast, temperature control in coal or 443
393 biomass power plants is achieved by using feed- 444
394 water downstream the condenser (Fernandez 445
395 et al., 2016; Garðarsdóttir et al., 2017). 446

396 Steam availability at the IP-LP crossover 447
397 does not limit the dynamic operation of inte- 448
398 grated system. This is largely due to the steam 449
399 requirements of the CO₂ capture plant being 450
400 small compared the large amount of steam pro- 451
401 duced in the Rankine cycle of the thermal 452
402 power plant (Jordal et al., 2012; Rezazadeh 453
403 et al., 2015). As a result, steam can always 454
404 be extracted by modifying the opening of the 455
405 steam extraction valve. This equipment can 456
406 move from fully open to fully closed in seconds 457
407 and thus their dynamics are negligible com- 458
408 pared to those governing the thermal power 459
409 plant and post-combustion capture plant. 460

410 System integration also includes the cooling 461
411 and compression of the exhaust gas leaving the 462
412 heat-recovery steam generation. From the per- 463
413 spective of dynamic operability, treatment of 464
414 this flue gas is not a major concern due to 465
415 the fast the dynamics of the blowers utilized to 466
416 overcome the absorber column pressure drop, 467
417 and hence do not limit the capture plant pro- 468
418 cess dynamics. The direct contact cooler only 469
419 introduces time delays. Ideally, the equipment 470
420 integrating the thermal power plant with the
421 post-combustion capture plant should not slow 471
422 the overall transient operation of the integrated
423 system. However, this coupling may lead to in- 472
424 teractions between both plants that can affect 473
425 their dynamics. 474

426 As different time-scales govern the dynamic 475
427 operation of thermal power plants and post- 476
428 combustion capture plants, system integration 477
429 must consider the distinctively different pro- 478
430 cess dynamics. Whilst heat capacitance in the 479
431 steam generator limits the transient behaviour 480
432 of thermal power plants to an order of minutes, 481

typically 10-20 minutes for power plants of sev-
eral hundred MW, the large solvent volumes
and long circulation time in the CO₂ capture
plant might lead to stabilization times in the
order of hours (Lawal et al., 2012; Montañés
et al., 2017b; Garðarsdóttir et al., 2017).

The power demand defines the operation of
the power plant and hence the mass flow rate
of the exhaust gas. Whereas changes in the flue
gas conditions do not affect the performance of
the thermal power plant, such changes are a
disturbance for the capture plant, which must
adapt its operation to meet the CO₂ capture
targets. This may lead to different steam ex-
traction rates that also modify the operating
conditions in the power plant. If steam ex-
traction variation occurs at a slow dynam-
ics scale, *i.e.*, the time-scale defined by the
capture plant, small fluctuations and longer
stabilization times are obtained in the power
generation of the low-pressure steam turbine
(Lawal et al., 2012; Garðarsdóttir et al., 2017;
Montañés et al., 2017b). However, this type of
interaction between both plants is not critical
as the thermal power plants are faster than the
slow-dynamic time-scales of the CO₂ capture
plant. Furthermore, steam extraction does not
significantly influence the load of the power
plant. On the contrary, steam extraction in the
fast dynamic time-scale occurs simultaneously
with the change of power plant load and may
lead to dynamic interactions that compromise
the power generation capacity of the system.
Therefore, it is important to address this pos-
sible issue by studying the dynamic interaction
between the thermal power plant and the CCS
system in the fast dynamics time-scale, which
are addressed in Sections 3 and 4.

3. Dynamic Modelling

In this study, a physics-based model of a
615 MW NGCC integrated with a 30 wt%
MEA-based post-combustion capture process
was used to study the dynamic interaction of
NGCC integrated with absorption CO₂ cap-
ture Montañés et al. (2017b). Triple pres-
sure steam cycles with reheat are the most ef-
ficient and common configuration of modern
natural gas combined cycles (Alobaid et al.,
2017; Kehlhofer et al., 2009). GT PRO (Ther-

482 moflow, 2014) was utilized to design the natu- 533
483 ral gas combined cycle as it provides detailed 534
484 information about the geometry of the plant. 535
485 Full-physics dynamic modelling was carried out 536
486 in the Modelica-based (Modelica Association, 537
487 2019) software Dymola (Dassault Systemes, 538
488 2016) using the specialized Thermal Power li- 539
489 brary (Modelon, 2015). 540

490 Full-scale post-combustion capture plants 541
491 are designed based on the flue gas CO₂ con- 542
492 centration and conditions (i.e., flow rate, tem- 543
493 perature, pressure), the required CO₂ capture 544
494 rate, the maximum pressure drops in the ab- 545
495 sorber and stripper columns, column flooding 546
496 limits and a reasonable balance between capi- 547
497 tal and operational costs (Jordal et al., 2012; 548
498 Dutta et al., 2017). For the natural gas com- 549
499 bined cycle considered in this work, a capture 550
500 plant with two absorber columns in parallel 551
501 and one stripper for a nominal 90% CO₂ cap- 552
502 ture rate was found to meet these requirements 553
503 (Montañés et al., 2017b). A dual absorber pro- 554
504 cess topology was selected due to the limits in 555
505 column sizing and construction (Dutta et al., 556
506 2017). 557

507 Integration of the power and capture plants 558
508 was achieved by extracting steam from the 559
509 crossover between the intermediate- and low- 560
510 pressure steam turbines (see Section 2.3). 561
511 Thus, the low-pressure section of the steam 562
512 turbine was designed for nominal conditions 563
513 where steam is extracted to achieve a 90% 564
514 capture rate. Fig. 2 represents the layout of 565
515 the natural gas combined cycle integrated with 566
516 the post-combustion capture plant. Details 567
517 on the design data, performance indicators, 568
518 modelling assumptions and validation results 569
519 are presented in the work by Montañés et al. 570
520 (2017b). 571

521 The design of the power plant steam cycle 572
522 includes the extraction of steam for the CO₂ 573
523 capture plant. Consequently, the power gen- 574
524 eration distribution between the different tur- 575
525 bines in this power plant differs from mod- 576
526 ern NGCC without a capture plant. Fig. 3 577
527 represents the power generation distribution 578
528 at different gas turbine loads. The gas tur- 579
529 bine produces the majority of the power as 580
530 in any combined cycle without steam extrac- 581
531 tion, however, the contribution to the over-
532 all power generation of the low-pressure sec-

tion of the steam turbine is halved due to the
steam extraction (Jordal et al., 2012; Reza-
zadeh et al., 2015). Therefore, the contribu-
tion of the low pressure section in electric-
ity production and in the steam cycle dimin-
ishes as a result of the integration with the
post-combustion capture system. The high-
and intermediate-pressure steam turbines con-
tribute similarly as in NGCCs without steam
extraction. This leads to larger power genera-
tion from the intermediate-pressure section be-
cause of the similar inlet temperature owing to
the reheating and its larger pressure ratio.

4. Results and Discussion: Dynamics of a NGCC with CO₂ Capture

The dynamics of the natural gas combined cycle occur in shorter time scales compared to the overall transient operation of the integrated system. Thus, to study whether the steam extraction coupling affects the power generation capacity in different dynamic operation scenarios, the variations in the opening of the extraction valve must be faster than the dominant dynamics of the thermal power part (see Section 2). A damping sine signal was hence superimposed on the extraction valve opening to ensure fast dynamics in the interface between the NGCC and the capture plant (Ljung, 1987). This signal was characterized by an offset of 0.69 and an amplitude of 0.29, with a natural and damping frequencies of 0.01 and 0.001 Hz, respectively. These values ensure that variations in the steam extraction occur faster than the dominant dynamics of the NGCC. Albeit highly oscillating valve movements do not occur in practice during open loop operation (i.e., no feedback control), these values generate a signal that provides sufficient variation in steam extraction from the IP-LP crossover valve. This will give insight into the transient effects of variations in steam extraction on power generation.

Two different scenarios were considered to analyse the integration effect on the power generation during both steady-state and transient operation of the power plant. In the case where the NGCC is at steady-state, the damping sine in the valve opening drives the dynamics of the system. In contrast, when there is a

582 load change in the power plant, the dynam- 600
 583 ics are dictated by simultaneous changes oc- 601
 584 ccurring at the gas turbine and the steam ex-
 585 traction. Varying gas turbine loads directly af- 602
 586 fect power production. Secondly, steam cycle 603
 587 performance is influenced by change in exhaust 604
 588 gas conditions (*e.g.*, temperature and mass flow 605
 589 rate), and variations in steam extraction. 606

590 As optimal operation of the integrated system 607
 591 is not the main objective of this work, 608
 592 the NGCC only had a regulatory control layer, 609
 593 which regulates the steam temperature at the 610
 594 steam turbine inlet, levels in drums and con- 611
 595 denser, and the pressures in the deaerator and 612
 596 the low-pressure drum. In the post-combustion
 597 capture plant, the levels in the large vessels 613
 598 were exclusively controlled, fixing the solvent 614
 599 flow rate to nominal conditions. Details of the 615

implementation of this control structure are in-
 cluded in the work by [Montañés et al. \(2017b\)](#).

4.1. Effect of Steam Extraction During Steady State Operation of the NGCC

In this study, a sinusoidal signal was imposed in the steam extraction valve during steady state operation of the NGCC to observe the effect of disturbances in the interface of the integrated system. The power generation distribution was analysed in the NGCC, whilst key performance indicators of the capture plant such the carbon capture ratio and the reboiler temperature were investigated.

4.1.1. Power Generation Performance

Several part-loads during steady state operation are considered in order to cover a wide op-

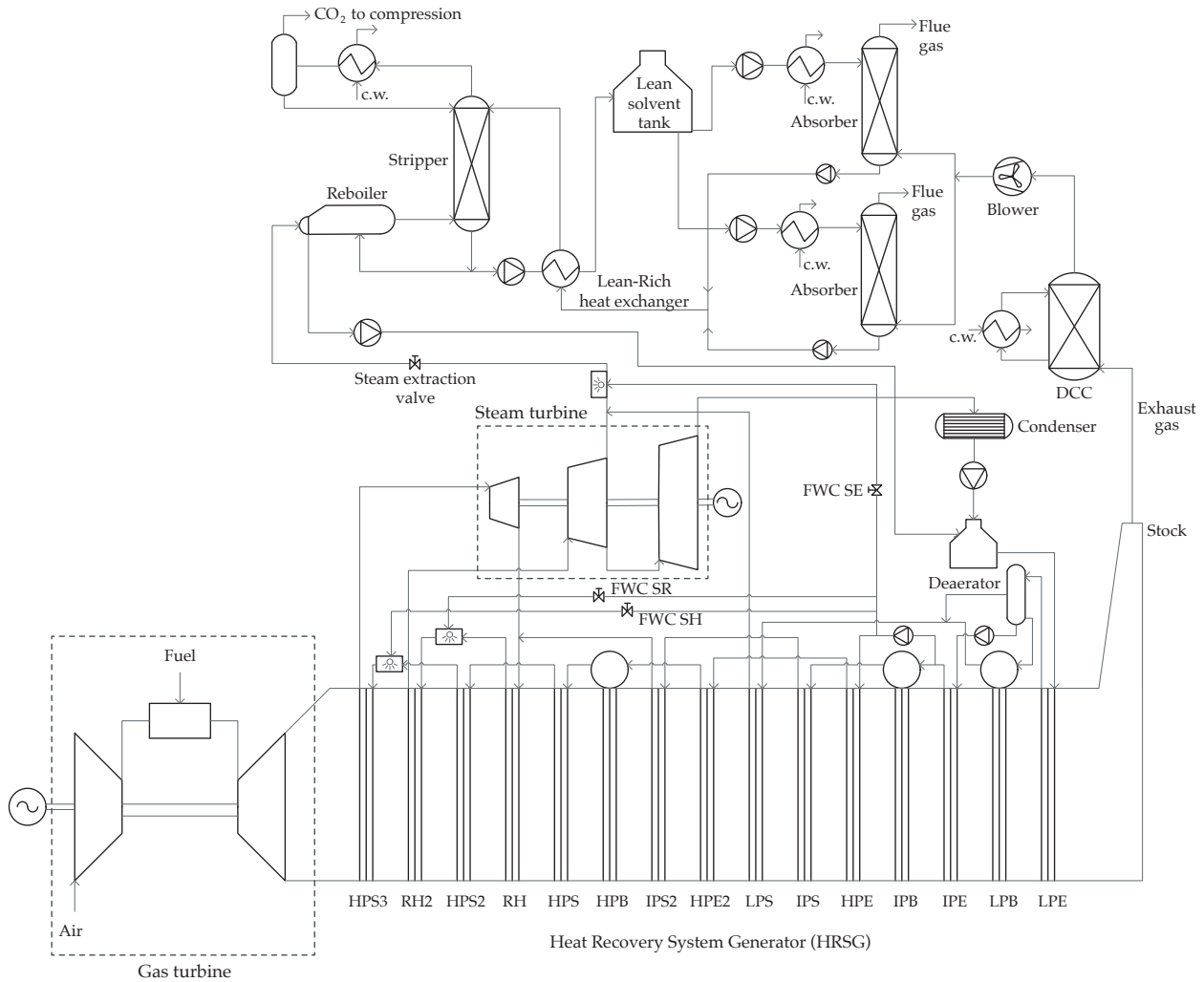


Figure 2: Process diagram of the natural gas combined cycle integrated with the post-combustion capture plant. The nomenclature is as follows. E: Economizer, B: Boiler, S: Superheater, R: Reheater P: Pressure, L: Low, I: Intermediate, H: High, FWC: Feed-water cooling, RS: Reheated steam, SS: Superheated steam, SE: steam extraction, DCC: Direct contact cooler.

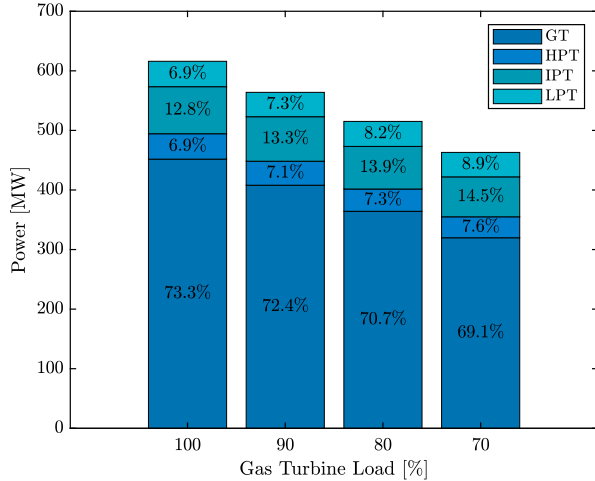


Figure 3: Power distribution of the natural gas combined cycle with CCS at different gas turbine loads.

616 erational range of the NGCC integrated with
 617 post-combustion CO₂ capture. Fig. 4 repre-
 618 sents the variation in mechanical power pro-
 619 duction in the NGCC and the different steam
 620 turbine sections due to variation in steam ex-
 621 traction from the IP-LP crossover. The open-
 622 ing of the steam extraction valve defines the
 623 mass flow rate of working fluid available for
 624 expansion, which appears to have the greatest
 625 impact on the low pressure section in Fig. 4d.
 626 The valve opening also alters the intermediate
 627 and low pressure sections of the steam cycle,
 628 leading to deviations in power generation by
 629 the intermediate-pressure section of the steam
 630 turbine, albeit to a lesser extent compared to
 631 the low-pressure counterpart.

632 The variation in power generation by the
 633 intermediate- and low-pressure steam turbines
 634 has a negligible impact on the total power pro-
 635 duced by the NGCC. The reasons for this ef-
 636 fect is the gas turbine generates most of the
 637 total power and the average contribution from
 638 the IP and LP steam turbine sections is 20%
 639 (see Fig. 3). Therefore, the variations induced
 640 by the steam extraction valve in the NGCC
 641 power generation during steady-state operation
 642 are negligible and can be easily compensated by
 643 the power controllers included in the gas tur-
 644 bine. Fig. 4a demonstrates how the variation
 645 in steam extraction only creates a small distur-
 646 bances in the total power generation.

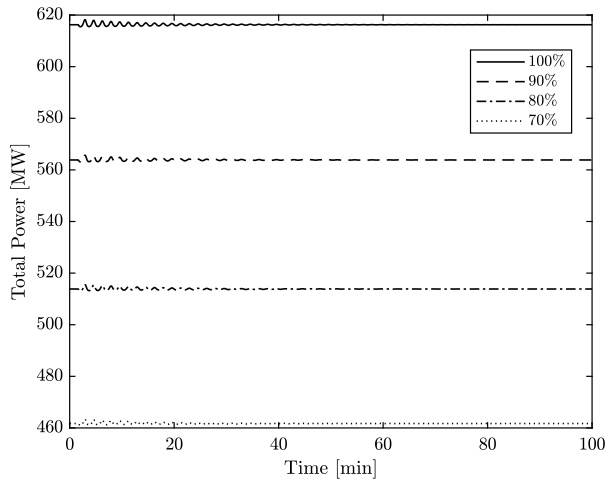
4.1.2. CO₂ Capture Performance

647 Steam extraction dictates the steam flow
 648 rate to the reboiler of the post-combustion cap-
 649 ture plant, thereby influencing the CO₂ cap-
 650 ture performance. Fig. 5 illustrates the effect
 651 of steam flow rate on reboiler temperature, lean
 652 loading and CO₂ capture rate. The steam flow
 653 rate has an insignificant effect on the transient
 654 behaviour of the reboiler temperature, where
 655 variation is less than 0.2 °C (shown in Fig. 5d).
 656 Therefore, the operating conditions within the
 657 stripper column are relatively unaffected and
 658 the solvent lean loading (Fig. 5b) only devi-
 659 ates slightly from its steady-state value. This
 660 results in almost constant CO₂ capture ratios,
 661 defined as the ratio of CO₂ product over CO₂
 662 in the feed flue gas (see Fig. 5a).

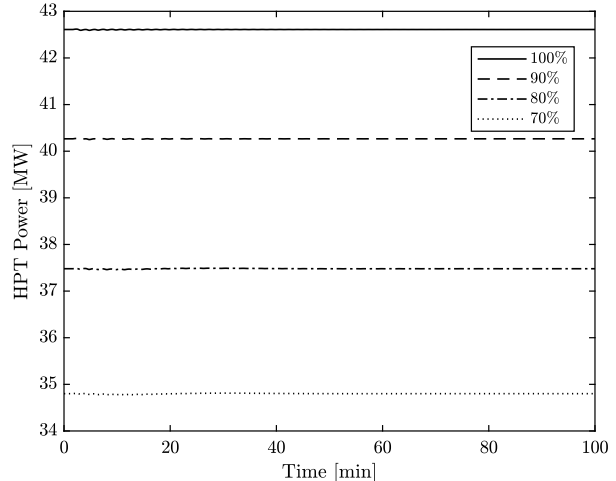
664 Fig. 5 shows how the effect of large fluctua-
 665 tions in steam mass flow rate (Fig. 5c) is damp-
 666 ened in the CO₂ capture system (described in
 667 Section 2.2). The dampening effect observed in
 668 these results are in line with previous dynamic
 669 operation studies discussed in Section 2.2. As
 670 steam flow rate fluctuates, the transfer of heat
 671 is limited by the heat capacitance of the equip-
 672 ment and fluid. Consequently, the change in
 673 reboiler temperature is dampened (Fig. 5d),
 674 that is, very little fluctuation observed. Hence,
 675 there is minor variation in the degree of solvent
 676 regeneration, which leads to limited change in
 677 lean loading (Fig. 5b). This contributes to the
 678 “smoothing” of the CO₂ capture ratio trend
 679 (Fig. 5a). Similarly, the volume of solvent hold-
 680 up in the plant (buffer/storage tanks, column
 681 sumps) also has a role in buffering variations in
 682 the system. Therefore, having large liquid ves-
 683 sels that limit the transient behaviour during
 684 slow disturbances are advantageous during fast
 685 disturbances as they buffer the dynamics and
 686 prevent departure from steady state set-points
 687 of the CO₂ capture process. However, this only
 688 occurs if the initial and final state of the dis-
 689 turbed variable are similar, otherwise the time
 690 required to reach a new steady-state is dictated
 691 by the large liquid hold-ups and the transport
 692 delays in the capture plant.

4.2. Effect of Steam Extraction During Dy- namic Operation of the NGCC

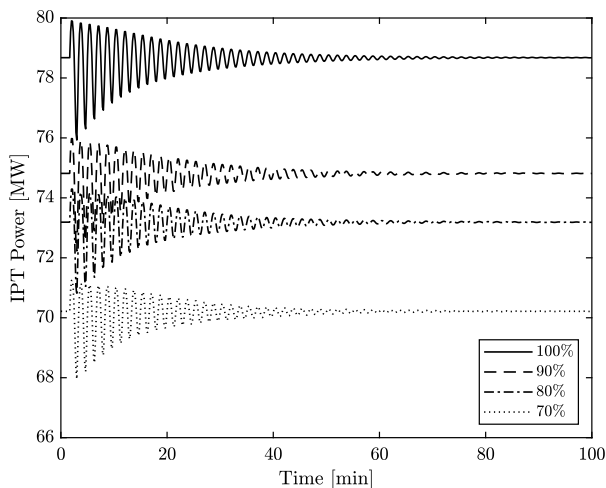
693 In this case, the disturbance in the steam
 694 extraction valve was imposed simultaneously
 695
 696



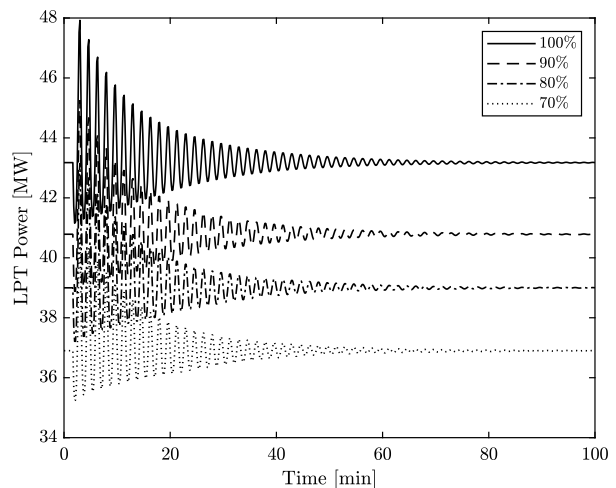
(a) Total power generation of the natural gas combined cycle.



(b) Power generation of the high-pressure steam turbine.



(c) Power generation of the intermediate-pressure steam turbine.



(d) Power generation of the low-pressure steam turbine.

Figure 4: Variation in power generation in the natural gas combined cycle and the steam turbine sections (HP, IP and LP) due to the fluctuation in the steam extracted from the IP-LP crossover at different gas turbine loads.

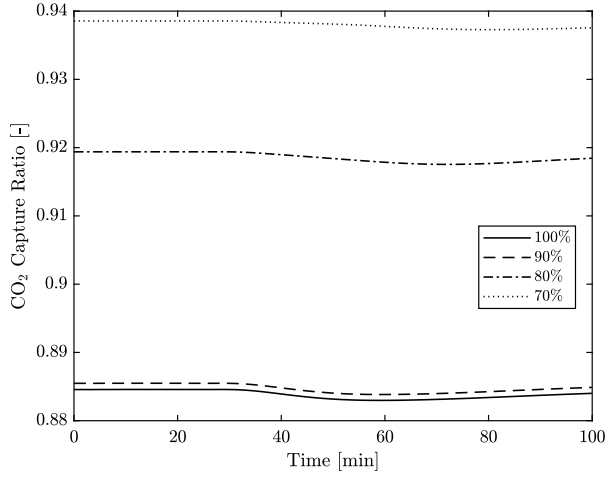
697 with a change of load in the gas turbine. The
 698 same parameters, i.e. power generation distri-
 699 bution and key performance indicators, were
 700 analysed in the NGCC and CO₂ capture plant,
 701 respectively.

702 4.2.1. Power Generation Performance

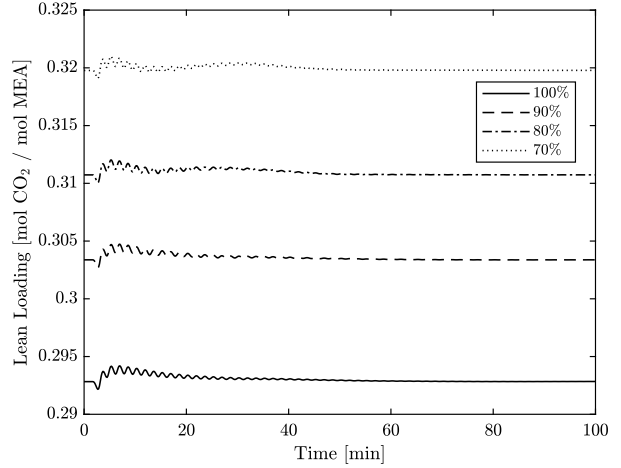
703 The damping sine signal in the steam ex-
 704 traction valve was implemented during a gas
 705 turbine load change from 100% to 70%. This
 706 demonstrates the effect of fast variations in the
 707 steam extraction during transient operation of
 708 the NGCC. Fig. 6 represents the power gen-
 709 eration profile of the overall power plant and
 710 of each section of the steam turbine. Fig. 7
 711 shows key process variables of the CO₂ cap-

712 ture plant during the transient operation of
 713 the power plant with varying steam extrac-
 714 tion. Figs. 6 and 7 show performance with
 715 fast dynamic fluctuations in the steam extrac-
 716 tion valve (black line), and without fluctua-
 717 tions (red line).

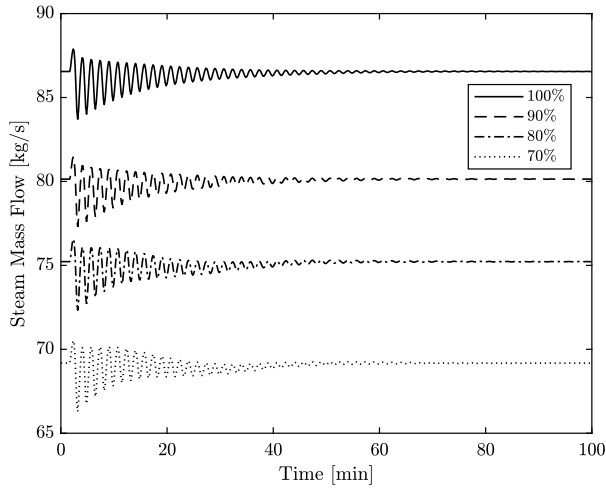
718 During transient operation, the change in
 719 gas turbine load dictates power generation
 720 (Fig. 6a). This is because the oscillations gen-
 721 erated by the steam extraction valve have a
 722 negligible effect on power generation in NGCC
 723 plants. This occurs regardless of the fluctua-
 724 tions in the IP and LP steam turbines, repre-
 725 sented in Figs. 6c and 6d respectively, due to
 726 the low contribution of these units to the to-
 727 tal power production (see Fig. 3). As steam



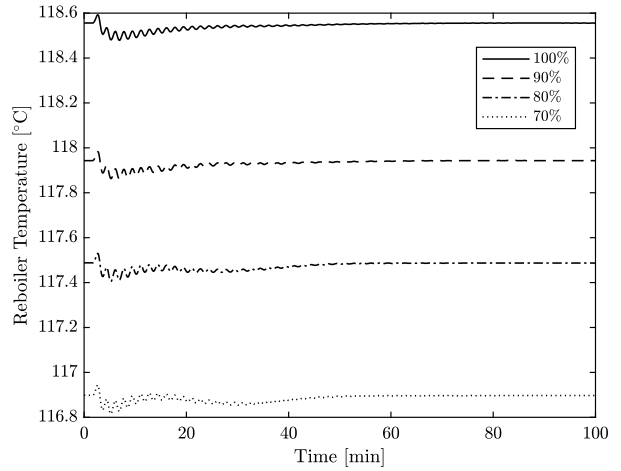
(a) CO₂ capture ratio.



(b) Lean solvent CO₂ concentration.



(c) Steam mass flow rate.



(d) Reboiler temperature.

Figure 5: Dynamic behaviour of key process variables in the post-combustion capture plant during steady-state operation of the natural gas combined cycle. Transient operation is driven by opening variations of the steam extraction valve.

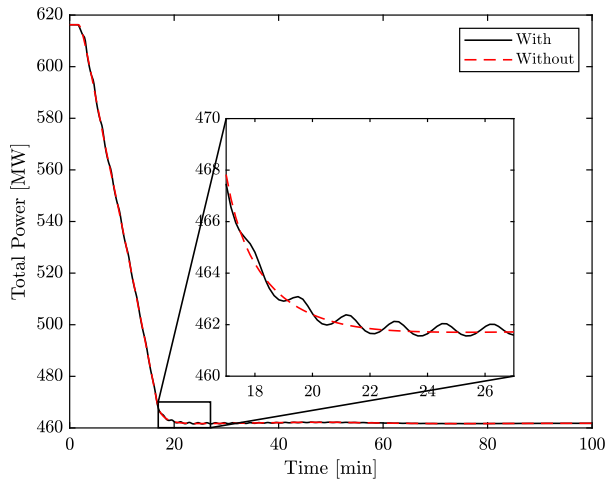
728 extraction does not have a notable effect on 744
 729 the total power generation, the NGCC power 745
 730 plant may operate independently of the cap- 746
 731 ture plant without any penalty on its dynamic 747
 732 performance. 748

733 4.2.2. CO₂ Capture Performance 750

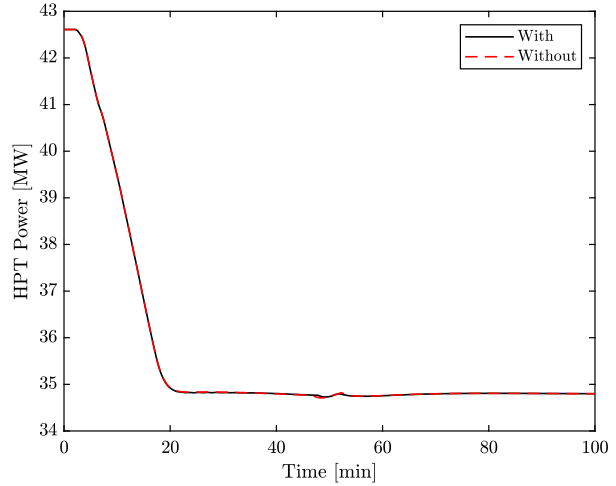
734 The transient behaviour of the CO₂ capture 751
 735 process is governed by the variation in flue gas 752
 736 conditions due to changes in gas turbine load 753
 737 and the steam flow rate fed to the reboiler, 754
 738 which depends on the steam availability in the 755
 739 power plant and the opening of the steam ex- 756
 740 traction valve. The gas turbine load deter- 757
 741 mines steam availability for extraction at the 758
 742 IP-LP crossover valve, and hence dictates the 759
 743 dynamic performance of the reboiler and strip- 760

per. The scenario without fast dynamic fluctu-
 ations in steam extraction is represented by the
 red line in Fig. 7, whereas the behaviour with
 fast valve fluctuations is shown by the black
 line.

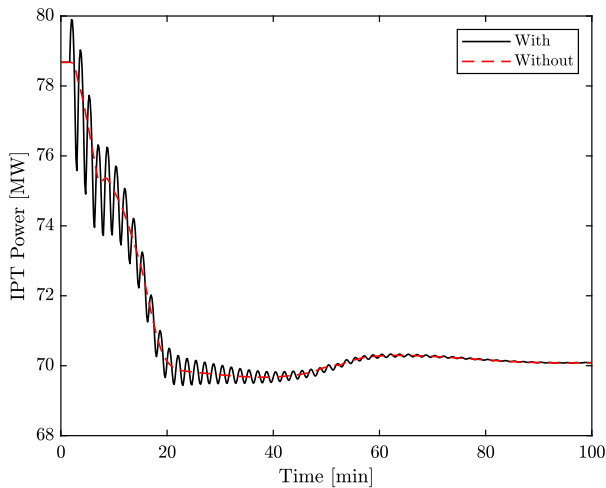
For a given steam extraction valve open-
 ing, the decrease in steam availability that
 arises from the change in gas turbine load re-
 sults in less steam extraction (Fig. 7c), which
 leads to more pronounced variations in the re-
 boiler temperature and lean loading. Unlike
 the fast disturbances of imposed fluctuations in
 the opening of the steam extraction valve, the
 gas turbine load change disturbance is slower.
 There is sufficient time for heat transfer from
 the steam to the reboiler fluid, thus reboiler
 temperature follows the same trajectory as the



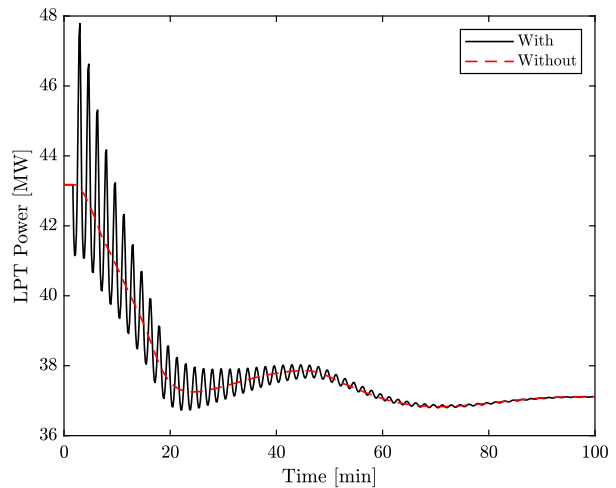
(a) Total power generation of the natural gas combined cycle.



(b) Power generation of the high-pressure steam turbine.



(c) Power generation of the intermediate-pressure steam turbine.



(d) Power generation of the low-pressure steam turbine.

Figure 6: Power generation dynamic behaviour during a gas turbine load change from 100% to 70% with and without fast dynamic fluctuations in the steam extraction valve.

761 steam flow rate. Similarly, as the reboiler tem- 777
 762 perature dictates the degree of solvent regen- 778
 763 eration, lean loading has the same trend. The 779
 764 variation in reboiler temperature and lean load- 780
 765 ing have an apparent effect on the CO₂ capture 781
 766 ratio. 782

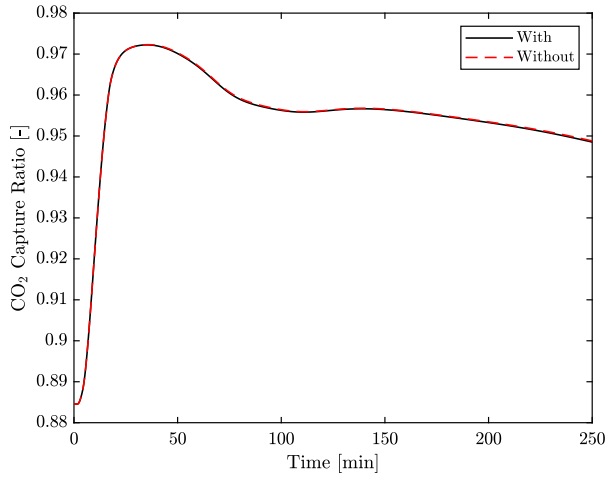
767 In contrast, fast dynamic fluctuations in the 783
 768 steam extraction do not disrupt the transient 784
 769 behaviour of the plant as the main process vari- 785
 770 ables follow the same trajectory as in the sce- 786
 771 nario without fluctuations (red and black lines 787
 772 in Fig. 7). Thus, steam availability in the 788
 773 steam cycle has a more pronounced affect on 789
 774 the dynamic response of the CO₂ capture plant 790
 775 than the opening of the steam extraction valve.

776 Similar to the steady-state operation results,

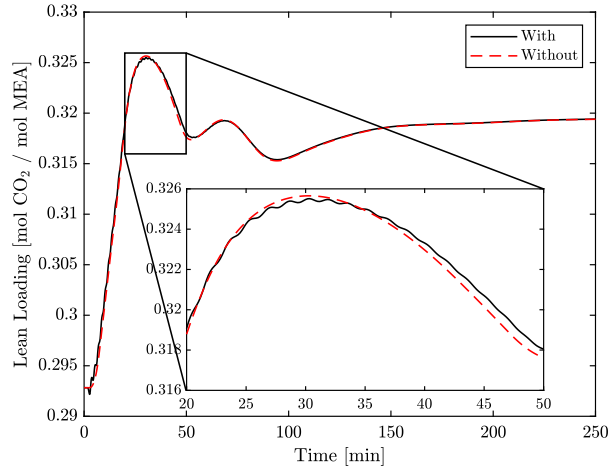
a smoothing effect of the fast fluctuations in 777
 steam extraction was observed during dynamic 778
 operation. Due to the fluctuations in the steam 779
 valve being so rapid, which subsequently re- 780
 sults in equally rapid steam flow rate fluctu- 781
 ations, there is insufficient time for heat to 782
 transfer from the steam to the reboiler fluid. 783
 Thus, the reboiler temperature, lean loading 784
 and CO₂ capture rate are practically the same 785
 with and without steam valve fluctuations. 786

787 5. Conclusions

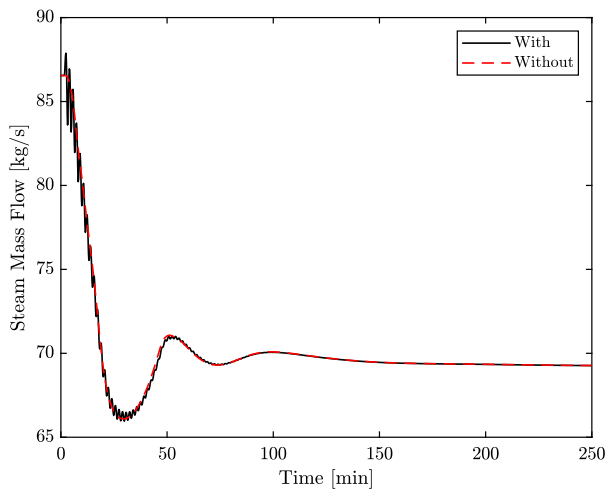
788 There are essentially two ways to integrate 789
 789 post-combustion CO₂ capture with thermal 790
 790 power plants. The first simply connects the



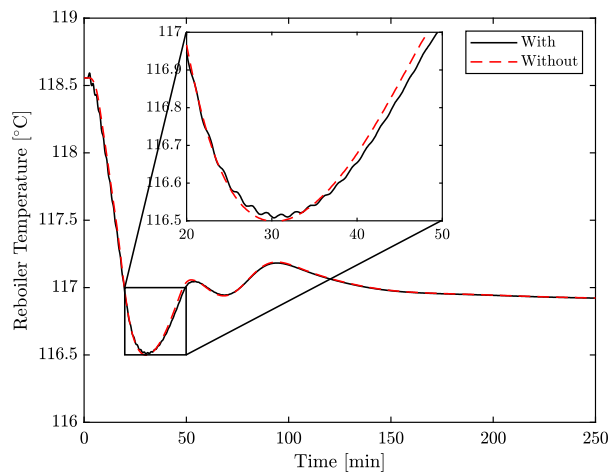
(a) CO₂ capture ratio.



(b) Lean solvent CO₂ concentration.



(c) Steam mass flow rate.



(d) Reboiler temperature.

Figure 7: Dynamic behaviour of key process variables in the post-combustion capture plant during a gas turbine load change from 100% to 70% with and without fast dynamic fluctuations in the steam extraction valve.

791 exhaust gas with the capture process, and the 809
 792 energy required for solvent regeneration is supplied 810
 793 externally. Whilst this does present the 811
 794 challenge of mitigating any emissions associated 812
 795 with providing that energy, it does entirely 813
 796 avoid imposing constraints on the operability 814
 797 of the power plant - this form of CCS is 815
 798 an entirely “end of pipe” solution. The second, 816
 799 more commonly discussed, option involves the 817
 800 extraction of steam from between the intermediate 818
 801 and low pressure steam turbines. This 819
 802 avoids the challenge of having to mitigate additional 820
 803 emissions, but has led to concerns as to the effect 821
 804 this strategy might have on the operability of the 822
 805 power plant, since these two plants operate in two 823
 806 different transient time-scales. This work seeks to 824
 807 address this challenge by analysing the effect of 825
 808 disturbances 826

on power generation capacity, specifically disturbances 810
 with faster dynamics than the dominant dynamics of the 811
 power plant. 812

Transient power generation was assessed 813
 during steady-state and dynamic operation of the 814
 power plant by modifying the valve opening for 815
 steam extraction in the short time-scales defined by 816
 the power plant. Since the gas turbine generates 817
 most of the total power, fluctuations in the steam 818
 extraction valve have no impact on the power 819
 generation capacity. In steady-state power plant 820
 operation, the total power generation remains 821
 unaltered with small fluctuations around the 822
 steady-state value that are easily compensated for 823
 with small adjustments in the gas turbine. During 824
 transient operation of the power plant, the change 825
 of load in the gas turbine drives the dynamic 826
 behaviour of the

827 NGCC. Hence, disturbances in steam extrac- 878
828 tion can be regarded as noise around the tran- 879
829 sient value dictated by the gas turbine load. 880

830 Different behaviour may be expected from 881
831 power plants where the steam cycle gener- 882
832 ates most of the power, *e.g.* coal-fired power 883
833 plants. Valve opening fluctuations in this type 884
834 of power plants might lead to larger variations 885
835 in the total power generation since larger steam 886
836 mass flow rates are required in the steam tur- 887
837 bine. Therefore, steam extraction from power 888
838 plants dominated by the steam cycle perfor- 889
839 mance has greater influence on power genera- 890
840 tion, and may add value to the flexible perfor- 891
841 mance of NGCCs. However, this behaviour is 892
842 yet to be demonstrated by dynamic studies. 893

843 The transient behaviour of the capture plant 894
844 was similar to the power plant since its dynam- 895
845 ics is dominated by the operating conditions in 896
846 the gas turbine and steam cycle. A change in 897
847 gas turbine load results in different flue gas flow 898
848 rate and steam availability, thereby influenc- 899
849 ing the performance of the capture plant. The 900
850 varying steam extraction only leads to small 901
851 fluctuations, with the trends following the same 902
852 trajectory as the profile of the scenario varying 903
853 valve opening variations. These small fluctu- 904
854 ations disturb the process and are smoothed 905
855 along the capture plant. This effect is demon- 906
856 strated by the disturbance starting as signif- 907
857 icant fluctuations in steam flow rate, which 908
858 dampen to become smaller fluctuations in re- 909
859 boiler temperature and lean loading, then fi- 910
860 nally resulting in a smooth CO₂ capture ra- 911
861 tio profile. The dampening effect is attributed 912
862 to the heat capacitance of the system and the 913
863 buffering of the disturbance in the large liq- 914
864 uid hold-ups. Thus, the large vessels of the 915
865 capture plant are advantageous for small, fast 916
866 variations as they buffer disturbances, avoiding 917
867 departure from steady-state conditions. This 918
868 phenomena occurs at both steady and dynamic 919
869 operation of the NGCC. 920

870 These results highlight the benefits and dis- 920
871 advantages of having large liquid hold-ups in 921
872 the capture plant. Large storage vessels allow 922
873 the buffering of the fast variations in the pro- 923
874 cess variables. However, these vessels also lead 924
875 to slow transients, increasing the time to reach 925
876 a new steady state to several hours, which will 926
877 potentially limit the flexibility of the capture 927

plant. This suggests that the post-combustion 878
capture plants can be operated optimally and 879
independently of the power plant. Imposing 880
tight controls on specific variables to minimize 881
the difference between a value and its set-point 882
a could limit the flexibility of the integrated 883
system. Instead, the capture plant should 884
aim at finding a new optimal operation point 885
given the boundary conditions imposed by the 886
power plant. This is because any changes in 887
steam extraction to achieve optimal operat- 888
ing conditions would not affect power plant 889
performance, as shown in this work. There- 890
fore, the decarbonisation of an NGCC via post- 891
combustion CO₂ capture does not appear to 892
impose any limitation on the flexibility or op- 893
erability of the underlying power plant in terms 894
of power generation. 895

Therefore, one key research challenge is to 900
develop control strategies and operation proto- 901
cols that enable optimal operation of the cap- 902
ture plant that is essentially independent from 903
the operation of the power plant rather than 904
load following mode with fixed capture ratios 905
(Sahraei and Ricardez-Sandoval, 2014; Bankole 906
et al., 2018; Decardi-Nelson et al., 2018). This 907
may lead to improvements in the financial vi- 908
ability of the CCS project as steam extraction 909
fluctuations have no impact on power genera- 910
tion. The development of process control struc- 911
tures designed for flexible operation and dy- 912
namic conditions will be an important area of 913
future research (Åkesson et al., 2012; Hauger 914
et al., 2019). Finally, the development of reli- 915
able start-up and shut-down protocols for CCS- 916
equipped power plants so as to avoid increasing 917
the carbon intensity of these assets is a priority. 918

919 Acknowledgements

This work has been financially supported by 920
the Department of Energy and Process Engi- 921
neering at the Norwegian University of Sci- 922
ence and Technology - NTNU, and the fund- 923
ing from the Research Councils UK (RCUK) 924
under grants EP/M001369/1 (MESMERISE- 925
CCS), EP/M015351/1 (Opening New Fuels for 926
UK Generation), EP/N024567/1 (CCSInSup- 927
ply), and NE/P019900/1 (GGR Opt). The au-
thors also thank Dr. Rubén Mocholí Montañés
for providing the dynamic model of the power
plant and for his valuable advice.

928 **References**

- 929 Adams, T., Mac Dowell, N., 2016. Off-design point
930 modelling of a 420 MW CCGT power plant inte-
931 grated with an amine-based post-combustion CO₂
932 capture and compression process. *Applied energy*
933 178, 681–702.
- 934 Åkesson, J., Laird, C. D., Lavedan, G., Prölb, K.,
935 Tummescheit, H., Velut, S., Zhu, Y., 2012. Nonlinear
936 model predictive control of a CO₂ post-combustion
937 absorption unit. *Chemical Engineering & Technology*
938 35 (3), 445–454.
- 939 Alobaid, F., Mertens, N., Starkloff, R., Lanz, T.,
940 Heinze, C., Epple, B., 2017. Progress in dynamic sim-
941 ulation of thermal power plants. *Progress in Energy*
942 *and Combustion Science* 59, 79–162.
- 943 Bankole, T., Jones, D., Bhattacharyya, D., Turton, R.,
944 Zitney, S. E., 2018. Optimal scheduling and its Lyapunov
945 stability for advanced load-following energy
946 plants with CO₂ capture. *Computers & Chemical*
947 *Engineering* 109, 30–47.
- 948 Botero, C., Finkenrath, M., Bartlett, M., Chu, R., Choi,
949 G., Chinn, D., 2009. Redesign, optimization, and
950 economic evaluation of a natural gas combined cycle
951 with the best integrated technology CO₂ capture.
952 *Energy Procedia* 1 (1), 3835–3842.
- 953 Bui, M., Adjiman, C. S., Bardow, A., Anthony, E. J.,
954 Boston, A., Brown, S., Fennell, P. S., Fuss, S.,
955 Galindo, A., Hackett, L. A., et al., 2018a. Carbon
956 capture and storage (CCS): the way forward. *Energy*
957 *& Environmental Science* 11 (5), 1062–1176.
- 958 Bui, M., Flø, N. E., de Cazenove, T., Mac Dowell, N.,
959 2020. Demonstrating flexible operation of the Tech-
960 nology Centre Mongstad (TCM) CO₂ capture plant.
961 *International Journal of Greenhouse Gas Control* 93,
962 102879.
- 963 Bui, M., Gunawan, I., Verheyen, V., Feron, P., Meule-
964 man, E., 2016. Flexible operation of CSIRO’s post-
965 combustion CO₂ capture pilot plant at the AGL Loy
966 Yang power station. *International Journal of Green-*
967 *house Gas Control* 48, 188–203.
- 968 Bui, M., Gunawan, I., Verheyen, V., Feron, P., Meule-
969 man, E., Adeloju, S., 2014. Dynamic modelling and
970 optimisation of flexible operation in post-combustion
971 CO₂ capture plants A review. *Computers & Chem-*
972 *ical Engineering* 61, 245–265.
- 973 Bui, M., Tait, P., Lucquiaud, M., Mac Dowell, N.,
974 2018b. Dynamic operation and modelling of amine-
975 based CO₂ capture at pilot scale. *International Jour-*
976 *nal of Greenhouse Gas Control* 79, 134–153.
- 977 Dassault Systemes, 2016. [https://www.3ds.com/
978 products-services/catia/products/dymola/](https://www.3ds.com/products-services/catia/products/dymola/).
- 979 Decardi-Nelson, B., Liu, S., Liu, J., 2018. Im-
980 proving Flexibility and Energy Efficiency of Post-
981 Combustion CO₂ Capture Plants Using Economic
982 Model Predictive Control. *Processes* 6 (9), 135.
- 983 Dutta, R., Nord, L. O., Bolland, O., 2017. Selection
984 and design of post-combustion CO₂ capture process
985 for 600 MW natural gas fueled thermal power plant
986 based on operability. *Energy* 121, 643–656.
- 987 Eser, P., Chokani, N., Abhari, R., 2017. Operational
988 and financial performance of fossil fuel power plants
989 within a high renewable energy mix. *Journal of the*
990 *Global Power and Propulsion Society* 1, 16–27.
- 991 Fernandez, E. S., del Rio, M. S., Chalmers, H.,
992 Khakharia, P., Goetheer, E. L., Gibbins, J., Luc-
993 quiaud, M., 2016. Operational flexibility options in
994 power plants with integrated post-combustion cap-
995 ture. *International Journal of Greenhouse Gas Con-*
996 *trol* 48, 275–289.
- Flø, N. E., Kvamsdal, H. M., Hillestad, M., Mejdell, T.,
2016. Dominating dynamics of the post-combustion
CO₂ absorption process. *Computers & Chemical*
Engineering 86, 171–183.
- Garðarsdóttir, S. Ó., Montanes, R. M., Normann, F.,
Nord, L. O., Johnsson, F., 2017. Effects of CO₂-
absorption control strategies on the dynamic perfor-
mance of a supercritical pulverized-coal-fired power
plant. *Industrial & Engineering Chemistry Research*
56 (15), 4415–4430.
- Garðarsdóttir, S. Ó., Normann, F., Andersson, K.,
Prölb, K., Emilsdóttir, S., Johnsson, F., 2015. Post-
combustion CO₂ capture applied to a state-of-the-art
coal-fired power plant: The influence of dynamic pro-
cess conditions. *International Journal of Greenhouse*
Gas Control 33, 51–62.
- González-Salazar, M. A., Kirsten, T., Prchlik, L., 2017.
Review of the operational flexibility and emissions
of gas-and coal-fired power plants in a future with
growing renewables. *Renewable and Sustainable En-*
ergy Reviews 82, 1497–1513.
- Hauger, S. O., Flø, N. E., Kvamsdal, H., Gjertsen,
F., Mejdell, T., Hillestad, M., 2019. Demonstra-
tion of non-linear model predictive control of post-
combustion CO₂ capture processes. *Computers &*
Chemical Engineering 123, 184–195.
- He, Z., Ricardez-Sandoval, L. A., 2016. Dynamic mod-
elling of a commercial-scale CO₂ capture plant inte-
grated with a natural gas combined cycle (NGCC)
power plant. *International Journal of Greenhouse*
Gas Control 55, 23–35.
- Hentschel, J., Spliethoff, H., et al., 2016. A parametric
approach for the valuation of power plant flexibility
options. *Energy Reports* 2, 40–47.
- Heuberger, C. F., Rubin, E. S., Staffell, I., Shah,
N., Mac Dowell, N., 2017a. Power capacity ex-
pansion planning considering endogenous technology
cost learning. *Applied Energy* 204, 831–845.
- Heuberger, C. F., Staffell, I., Shah, N., Mac Dowell, N.,
2016. Quantifying the value of CCS for the future
electricity system. *Energy & Environmental Science*
9 (8), 2497–2510.
- Heuberger, C. F., Staffell, I., Shah, N., Mac Dowell, N.,
2017b. A systems approach to quantifying the value
of power generation and energy storage technologies
in future electricity networks. *Computers & Chem-*
ical Engineering 107, 247–256.
- IEA, 2018a. CO₂ Emissions from Fuel Combustion
2018. [https://www.iea.org/statistics/
co2emissions/](https://www.iea.org/statistics/co2emissions/).
- IEA, 2018b. World Energy Outlook 2018: Executive
Summary. [https://webstore.iea.org/download/
summary/190?fileName=English-WE0-2018-ES.
pdf](https://webstore.iea.org/download/summary/190?fileName=English-WE0-2018-ES.pdf).
- IPCC, 2005. IPCC Special Report on Carbon Dioxide
Capture and Storage. Prepared by Working
Group III of the Intergovernmental Panel on Climate

- 1054 Change [Metz, B., O. Davidson, H. C. de Coninck, 1117
1055 M. Loos, and L. A. Meyer (eds.)]. Cambridge Uni- 1118
1056 versity Press, Cambridge, United Kingdom and New 1119
1057 York, NY, USA. 1120
- 1058 IPCC, 2014. Climate Change 2014: Synthesis Report. 1121
1059 Contribution of Working Groups I, II and III to the 1122
1060 Fifth Assessment Report of the Intergovernmental 1123
1061 Panel on Climate Change [Core Writing Team, R.K. 1124
1062 Pachauri and L.A. Meyer (eds.)], IPCC, Geneva, 1125
1063 Switzerland. 1126
- 1064 IPCC, 2018. Summary for Policymakers. In: Global 1127
1065 warming of 1.5°C. An IPCC Special Report on 1128
1066 the impacts of global warming of 1.5°C above pre- 1129
1067 industrial levels and related global greenhouse gas 1130
1068 emission pathways, in the context of strengthening 1131
1069 the global response to the threat of climate change, 1132
1070 sustainable development, and efforts to eradicate 1133
1071 poverty. [V. Masson-Delmotte, P. Zhai, H. O. Prt- 1134
1072 ner, D. Roberts, J. Skea, P. R. Shukla, A. Pirani, W. 1135
1073 Moufouma-Okia, C. Pan, R. Pidcock, S. Connors, J. 1136
1074 B. R. Matthews, Y. Chen, X. Zhou, M. I. Gomis, 1137
1075 E. Lonnoy, T. Maycock, M. Tignor, T. Waterfield 1138
1076 (eds.)]. World Meteorological Organization, Geneva, 1139
1077 Switzerland. 1140
- 1078 Jonshagen, K., Sammak, M., Genrup, M., 2012. Post- 1141
1079 combustion CO₂ capture for combined cycles uti- 1142
1080 lizing hot-water absorbent regeneration. Journal of 1143
1081 Engineering for Gas Turbines and Power 134 (1), 1144
1082 011702. 1145
- 1083 Jordal, K., Ystad, P. A. M., Anantharaman, R., 1146
1084 Chikukwa, A., Bolland, O., 2012. Design-point and 1147
1085 part-load considerations for natural gas combined cy- 1148
1086 cle plants with post combustion capture. Interna- 1149
1087 tional Journal of Greenhouse Gas Control 11, 271–1150
1088 282. 1151
- 1089 Kehlhofer, R., Hannemann, F., Rukes, B., Stirnimann, 1152
1090 F., 2009. Combined-Cycle Gas & Steam Turbine 1153
1091 Power Plants. Pennwell Books. 1154
- 1092 Kondziella, H., Bruckner, T., 2016. Flexibility require- 1155
1093 ments of renewable energy based electricity systems– 1156
1094 A review of research results and methodologies. Re- 1157
1095 newable and Sustainable Energy Reviews 53, 10–22. 1158
- 1096 Kvamsdal, H. M., Rochelle, G. T., 2008. Effects of the 1159
1097 temperature bulge in CO₂ absorption from flue gas 1160
1098 by aqueous monoethanolamine. Industrial & Engi- 1161
1099 neering Chemistry Research 47 (3), 867–875. 1162
- 1100 Lawal, A., Wang, M., Stephenson, P., Koumpouras, G., 1163
1101 Yeung, H., 2010. Dynamic modelling and analysis 1164
1102 of post-combustion CO₂ chemical absorption process 1165
1103 for coal-fired power plants. Fuel 89 (10), 2791–2801. 1166
- 1104 Lawal, A., Wang, M., Stephenson, P., Obi, O., 2012. 1167
1105 Demonstrating full-scale post-combustion CO₂ cap- 1168
1106 ture for coal-fired power plants through dynamic 1169
1107 modelling and simulation. Fuel 101, 115–128. 1170
- 1108 Ljung, L., 1987. System identification: theory for the 1171
1109 user. Prentice-hall. 1172
- 1110 Lucquiaud, M., Chalmers, H., Gibbins, J., 2009. 1173
1111 Capture-ready supercritical coal-fired power plants 1174
1112 and flexible post-combustion CO₂ capture. Energy 1175
1113 Procedia 1 (1), 1411–1418. 1176
- 1114 Mac Dowell, N., Shah, N., 2013. Identification of the 1177
1115 cost-optimal degree of CO₂ capture: An optimisation 1178
1116 study using dynamic process models. International 1179
1180 Journal of Greenhouse Gas Control 13, 44–58.
- Mac Dowell, N., Shah, N., 2014. Dynamic modelling and analysis of a coal-fired power plant integrated with a novel split-flow configuration post-combustion CO₂ capture process. International Journal of Greenhouse Gas Control 27, 103–119.
- Mac Dowell, N., Shah, N., 2015. The multi-period optimisation of an amine-based CO₂ capture process integrated with a super-critical coal-fired power station for flexible operation. Computers & Chemical Engineering 74, 169–183.
- Mac Dowell, N., Staffell, I., 2016. The role of flexible CCS in the UK’s future energy system. International Journal of Greenhouse Gas Control 48, 327–344.
- Mechleri, E., Fennell, P. S., Mac Dowell, N., 2017a. Optimisation and evaluation of flexible operation strategies for coal-and gas-ccs power stations with a multi-period design approach. International Journal of Greenhouse Gas Control 59, 24–39.
- Mechleri, E., Lawal, A., Ramos, A., Davison, J., Mac Dowell, N., 2017b. Process control strategies for flexible operation of post-combustion CO₂ capture plants. International Journal of Greenhouse Gas Control 57, 14–25.
- Modelica Association, 2019. <https://www.modelica.org/>.
- Modelon, 2015. Thermal Power Library. <https://www.modelon.com/library/thermal-power-library/>.
- Montañés, R., Flø, N., Nord, L., 2017a. Dynamic process model validation and control of the amine plant at CO₂ Technology Centre Mongstad. Energies 10 (10), 1527.
- Montañés, R. M., Flø, N. E., Nord, L. O., 2018. Experimental results of transient testing at the amine plant at Technology Centre Mongstad: Open-loop responses and performance of decentralized control structures for load changes. International Journal of Greenhouse Gas Control 73, 42–59.
- Montañés, R. M., Gardarsdóttir, S. Ó., Normann, F., Johnsson, F., Nord, L. O., 2017b. Demonstrating load-change transient performance of a commercial-scale natural gas combined cycle power plant with post-combustion CO₂ capture. International Journal of Greenhouse Gas Control 63, 158–174.
- Montañés, R. M., Korpås, M., Nord, L. O., Jaehnert, S., 2016. Identifying operational requirements for flexible CCS power plant in future energy systems. Energy Procedia 86, 22–31.
- Rezazadeh, F., Gale, W. F., Hughes, K. J., Pourkashanian, M., 2015. Performance viability of a natural gas fired combined cycle power plant integrated with post-combustion CO₂ capture at part-load and temporary non-capture operations. International Journal of Greenhouse Gas Control 39, 397–406.
- Rúa, J., Agromayor, R., Hillestad, M., Nord, L. O., 2020. Optimal dynamic operation of natural gas combined cycles accounting for stresses in thick-walled components. Applied Thermal Engineering Accepted.
- Sahraei, M. H., Ricardez-Sandoval, L. A., 2014. Controllability and optimal scheduling of a CO₂ capture plant using model predictive control. International Journal of Greenhouse Gas Control 30, 58–71.

1180 Tait, P., Buschle, B., Ausner, I., Valluri, P., Wehrli,
1181 M., Lucquiaud, M., 2016. A pilot-scale study of dy-
1182 namic response scenarios for the flexible operation of
1183 post-combustion CO₂ capture. *International Journal*
1184 *of Greenhouse Gas Control* 48, 216–233.
1185 Thermoflow, 2014. GT Pro 24.0. Thermoflow Inc.
1186 Wellner, K., Marx-Schubach, T., Schmitz, G., 2016. Dy-
1187 namic behavior of coal-fired power plants with post-
1188 combustion CO₂ capture. *Industrial & Engineering*
1189 *Chemistry Research* 55 (46), 12038–12045.