Moment resisting on-site splice of large glulam elements by use of mechanically
 coupled long threaded rods
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# 9 Abstract

10 Large spans of modern timber bridges can be achieved by use of glulam arches with network hanger configuration. Since 11 transportation and production limit the length of timber elements, the glulam arches must be spliced on bridge site. 12 However, it is difficult to obtain practical moment resisting on-site splicing of massive glulam elements featuring 13 flexural rigidity by the available timber splicing techniques. Consequently, the arches are often designed as trusses 14 containing a large number of connections, which are costly and present a risk of decay development. In the present paper, 15 a novel splicing technique suitable for large massive timber sections is presented. The flexural rigidity of the joint is 16 obtained by the utilisation of long threaded rods having large withdrawal stiffness. Fast and easy on-site assembly is 17 facilitated by mechanical coupling of the rods. The rods are oriented with a small inclination to grain, which prevents 18 potential development of shrinkage cracks along the rods. Experimental and numerical methods were used to investigate 19 the flexural joint characteristics. The joint prototypes featured large rotational stiffness without initial slip. As a basis for 20 practical joint design, analytical relations are proposed for estimating the rotational stiffness, the moment capacity and 21 the capacity under combined bending and normal force.

22 Keywords: long threaded rod, timber splice joint, glulam, 3D finite element model, rotational stiffness, moment capacity

# 23 1. Introduction

24 Feasibility studies of glulam arch bridges with network hanger configuration have shown excellent structural properties

for bridges with massive glulam arches spanning up to 100-120 m [1, 2]. Since the timber arches cannot be produced and

- transported in one piece, the timber elements must be spliced on bridge site. In order to maintain the stability of the
- arches, it is crucial to incorporate flexural rigidity in the splice connections [3-5].

Figure 1 shows the recently erected network arch bridge Steibrua in Norway [6]. With a span of 88 m, the bridge is currently the longest single-span timber road bridge in the world. However, due to the lack of rotationally stiff splicing solution for large timber elements, the arches of Steibrua are formed as hybrid timber-steel trusses. This is probably not the most optimal solution since the trusses contain a large number of connections, which are expensive and vulnerable to decay developments. A more durable and cheaper solution could be achieved by the use of massive glulam arches,

33 necessitating on-site splice joints with sufficient rotational stiffness.



Figure 1: Steibrua, Norway – Network arch bridge with glulam arches [6].

34

The pros and cons of different splice connection techniques in timber engineering are discussed in [7]. Recent research on steel rods glued into timber has demonstrated that connections featuring large stiffness and capacity can be achieved by using high strength epoxy adhesives [8-11]. However, for large joints, multiple rods are necessary, and the brittleness of the adhesives can lead to a progressive failure in a group of rods [12]. Therefore, design provisions for ductile failure are necessary [13-17]. The main shortcoming associated with the application of glued-in rods is the production.

Experience from reviewers of failed joints revealed inadequately mixed and incorrectly applied epoxy on site. Nowadays,
the production is limited to a climate controlled environment with quality control and skilled personnel [12].

42 The difficulties connected to gluing of rods are avoided by using long threaded rods, which are simply driven into pre-43 drilled holes in timber. Large rotational stiffness and moment capacity of spliced timber beams was achieved in [18] by 44 using commonly available long threaded rods (SFS WB-T-20). The rods were inserted parallel to the grain in the 45 opposed parts of timber beams, and the mutual splicing of the rods was carried out by grout-filled steel couplers (similar to systems used for reinforced pre-cast concrete). The parallel to grain orientation of the threaded rods enables effective 46 47 force transfer in the axial direction and allows the utilisation of the high withdrawal stiffness of rods parallel to the grain. 48 On the other hand, the development of shrinkage cracks (in the grain direction) in close proximity to the threaded rods 49 can lead to loss of capacity. In addition, the gluing operation on site implies quality control issues, and curing of the glue 50 affects the final setting time of the joint.

51 In this paper, a novel splicing solution is presented, which overcomes the aforementioned shortcomings by the use of 52 slightly inclined long threaded rods with a metric threaded part at one end. A principle layout of the joint is shown in 53 Figure 2. Inserting the rods with a small inclination to the grain avoids the risk of failure due to the occurrence of 54 shrinkage cracks since the rods cross several "layers" of wood. The mechanical joint of the rods allows easy and fast onsite mounting without the need of special tools. In order to transmit the normal force acting in the arch, mutual contact of 55 the mating timber end faces is assured by tightening the rods in the couplers. The shear force can be transmitted through 56 57 shear keys. A reliable prediction of the structural properties and ductile behaviour is achieved by design provisions 58 enforcing a failure mode driven by yielding of the steel rods.



59

60 Figure 2: Principle layout of the splice joint with inclined mechanically coupled long threaded rods.

61

The key prerequisite regarding splicing of massive glulam arches is a sufficient and predictable rotational stiffness of the splice joints. Therefore, the main objective of the present work is to determine the flexural characteristics of the proposed splicing technique by the use of experimental testing on full-scale prototype joints and numerical models. In order to allow for practical design of the joint, analytical relations are here proposed for the determination of the rotational stiffness, the moment capacity and the combined capacity for bending moment and normal force.

67 2. Materials and methods

# 68 2.1. Analytical prediction of flexural joint characteristics

# 69 **2.1.1. Rotational stiffness**

An analytical model for the determination of the rotational stiffness of a splice joint using long threaded rods inserted

parallel to the grain was derived in [18]. However, due to the inclination of the rods in the proposed splicing solution, a

12 lateral force component is present at the rod-ends. A modification of the model accounting for the lateral deformations of

the rods is presented in the following, with input parameters specified in the Appendix. The model parameters are shown in Figure 3. Here, *h* and *b* are the height and the width of the cross-section respectively,  $a_0$  is the height of wood in compression,  $a_i$  is a coordinate along z-axis of the i-th rod row determined from the upper edge of wood in compression (with reference to Figure 3,  $a_i$  is negative for the rods in compression),  $h_i$  is the height of timber end faces in mutual contact,  $\theta$  represents the relative rotation of the end timber faces,  $\gamma$  is the rod-to-grain angle,  $\sigma_x$  is the longitudinal compression stress in wood,  $K_{si}$  is the stiffness of the i-th rod row,  $u_i$  is the horizontal displacement at the i-th rod row,  $z_i = a_i - a_0$  is the z-coordinate of the the i-th rod row, and  $F_i$  is the force in the i-th rod row found by [18]:

80 
$$F_i = K_{si} \cdot u_i = K_{si} \cdot \theta \cdot z_i$$



81

82 Figure 3: Analytical model nomenclature.

The analytical model presented in [18] is based on the assumption that the relative rotation of the end faces of the splice connection caused by the action of bending moment is approximated by a relative rotation of the end sections of a beam portion of length  $2l_c$ . The flexural stiffness is thus governed by the deformation of the wooden part in compression and the elongation and the contraction of springs representing the axial stiffness of the steel rods.

87 The position of the neutral axis is obtained by requiring no resulting axial force [18]:

88
$$a_{0} = \frac{-\sum_{i=1}^{n} K_{si} + \sqrt{\left(\sum_{i=1}^{n} K_{si}\right)^{2} + \frac{E \cdot b}{l_{c}} \cdot \sum_{i=1}^{n} K_{si} \cdot a_{i}}}{\frac{E \cdot b}{2 \cdot l_{c}}}$$
(1)

where *E* is the elastic modulus of timber parallel to the grain,  $l_c$  represents an equivalent length of the compression (crushing) zone at the mutual contact of the wooden parts ( $l_c$  is assumed to be of equal size on both sides of the contact 91 interface of the timber parts), and *n* is the number of the rod rows. The rotational stiffness of the connection  $k_{\theta}$  is

92 determined by [18]:

93 
$$k_{\theta} = \sum_{i=1}^{n} K_{si} \cdot z_{i}^{2} + \frac{E \cdot b \cdot a_{0}^{3}}{6 \cdot l_{c}}$$
(2)

94 More information and expressions for the determination of  $K_{si}$  and  $l_c$  are given in the Appendix.

#### 95 2.1.2. Moment capacity

The moment capacity of the splice joint,  $M_u$ , is estimated under the assumption of elastic distribution of forces until an ultimate force is reached in either of the rods or in the timber (bilinear approximation) by:

98 
$$M_{u} = \min \begin{cases} F_{u,i} \cdot z_{eq,i} \\ M_{u,i} \end{cases}$$
(3)

where  $M_{u,t}$  is the moment capacity corresponding to the compression strength of timber,  $F_{u,i}$  is the ultimate force in the ith rod row and  $z_{eq,i}$  is the equivalent lever arm of the i-th rod row given by:

101 
$$z_{eq,i} = \frac{M}{F_i} = \frac{k_{\theta}}{K_{si} \cdot z_i}$$
(4)

102 The moment capacity corresponding to the compression strength of timber  $M_{u,t}$  (with a linear distribution of compression 103 stresses as illustrated in Figure 3) is determined by:

104 
$$M_{u,t} = \frac{2 \cdot k_{\theta} \cdot l_c \cdot f_{c,0}}{E \cdot a_0}$$
(5)

105 where  $f_{c,0}$  is the timber strength in compression parallel to the grain.

106 The ultimate force in the i-th rod row,  $F_{u,i}$ , is obtained by:

107 
$$F_{u,i} = n_r \cdot \cos \gamma \cdot \min(R_{axu}, R_u)$$
(6)

108 where  $n_r$  is the number of the rods in one row,  $R_{axu}$  is the ultimate withdrawal strength of the rods and  $R_u$  is the tensile

- 109 strength of the rods. See the Appendix for more details and determination of  $R_{axu}$ .
- 110

#### 111 2.1.3. Capacity under combined action of bending moment and normal force

112 The interaction of bending moment and normal force acting in the splice joint was studied by the use of numerical

113 models. It is proposed to verify the joint capacity by a modification of the relation provided by Eurocode 5 [19] for

114 combined bending and axial compression:

115 
$$\left(\frac{N}{N_u}\right)^2 + \frac{M}{M_u} \le 1$$
(7)

where *N* and *M* are the normal force and the bending moment acting in the joint, respectively, and  $N_u$  and  $M_u$  are the capacity in axial compression and bending, respectively.

118 The ultimate normal force,  $N_u$ , is determined assuming a ductile post-elastic behaviour in both the rods and the timber in 119 compression by:

120 
$$N_{u} = \sum_{i=1}^{n} F_{u,i} + b \cdot h_{t} \cdot f_{c,y}$$
(8)

- 121 where  $f_{c,y}$  is the asymptotic final compression stress in timber, which is assumed as 0.8  $f_{c,0}$  [20]. The asymptotic
- 122 compression stress  $f_{c,y}$  corresponds to a simplified bilinear elastic plastic stress-strain curve for timber in compression

123 parallel to the grain according to [21], as illustrated in Figure 4.





- 125 Figure 4: Stress-strain relation for timber in axial tension and compression parallel to the grain [21]
- 126 Note that due to different failure modes, the moment capacity by Eq. (5), is based on a linear elastic stress distribution in
- 127 timber, while the compression capacity by Eq. (8) is based on a ductile "plastic" stress distribution. It was observed in
- 128 [18] that an increasing rotation in the joint gives rise to shear stresses and tensile stresses perpendicular to the grain in the
- 129 compression zone, due to change of slope at the rotated end faces. The possible "plastification" of the compression zone

is thus accompanied by occurrence of cracks along the grain near the neutral axis, which limit the bending capacity. Onthe contrary, the failure mode in timber in pure compression is quite ductile [20].

132

#### 133 **2.2. Experimental tests**

## 134 2.2.1. Experimental set-up

135 The prototype beam splices were tested in a four-point bending configuration yielding a pure bending in the splice connection. The experimental set-up and the detailing of the joint are shown in Figure 5 and Figure 7, respectively. The 136 137 relative rotations of the end faces (denoted  $\theta$  in Figure 3) were obtained by linear regression of the horizontal 138 displacements, monitored by the digital image correlation (DIC) system ARAMIS [22] along vertical sections placed at 139 both spliced beams in a distance of 130 mm from the axis of symmetry of the joint (this corresponds to a distance of 140 approximately 10-20 mm from the outmost edge of the slots for the rod couplers). The measurements by DIC were 141 validated by use of additionally applied transducers. The transducers were located on one side of the beam (confer Figure 142 6), and the other side of the beam was monitored by DIC. In total, two rotational transducers (denoted as T1 and T2 in Figure 6) and five displacement transducers (denoted as T3 - T7) were placed across the end faces of the splice joint. 143 144 Linear approximation between the relative displacements obtained from the displacement transducers at the compression 145 and the tension side, respectively, was used to determine the relative rotations of the end faces. The load was applied 146 according to the loading procedure given in EN 26891:1991 [23].



Figure 5: Experimental set-up: (a) the spliced beam, (b) technical specification of the joint. Measures given in mm.



148

149 Figure 6: Schematic visualization of transducer locations.

150 The experimental programme is summarised in Table 1. The purpose of the experiments was to investigate the structural

151 performance for different geometrical configurations, making available validation cases for numerical studies. Therefore,

152 one test for each configuration was performed. Four geometrical configurations were tested, in which number of rods, 153 effective length of rods,  $l_{ef}$  (the length of rod screwed in the timber), and detailing in the rod couplers were varied. The 154 threaded rods, inserted in the opposed parts of the beams with a constant rod-to-grain angle of 5 degrees, were connected 155 in the rod couplers. Purpose-made fitted washers were used to distribute the force from the rods onto the surface of the 156 couplers. The detailing at the compression couplers was varied in the different configurations. The configurations C1 and C3 contained, besides the external fitted washers, standard M20 washers at the inner side (see Figure 7c), while in C2 157 158 and C4, the fitted internal washers were used (see Figure 7d and Figure 7e). The timber end faces of the spliced parts of 159 the beams were brought in direct mutual contact provided by tightening the nuts. It should be noted that the level of prestress at the end grains is difficult to quantify due to unequal planeness of the surfaces. The nuts were tightened until the 160 161 end faces were in a full mutual contact with no clearances. Since no initial slip was observed in any of the tested 162 specimens, this procedure seems to provide a sufficient contact of the end faces.



Figure 7: Detailing of the joint: (a) the assembled joint (C1), (b) detail at the tension side (C3). Details at the compression side: (c) C3 under assembly, (d) C2 after testing, (e) C4.

### 163 Table 1: Experimental programme

$n_r$	$l_{ef}$ [mm]	DA1	DA2	Notation
2	1850	no	no	C1-2-1850-0-0
2	1200	yes	no	C2-2-1200-1-0
3	1200	no	no	C3-3-1200-0-0
3	1000	yes	yes	C4-3-1000-1-1
	$ \begin{array}{c} n_r \\ 2 \\ 2 \\ 3 \\ 3 \end{array} $	$\begin{array}{c c c c c c c c c c c c c c c c c c c $	$\begin{array}{c c c c c c c c c c c c c c c c c c c $	$\begin{array}{c c c c c c c c c c c c c c c c c c c $

Design Alternative:

DA1 - Application of internal fitted washers in compression couplers

DA2 - Anchorage of compression couplers

- Lateral displacements of the prototype rod couplers at the tension side were prevented by the use of screws inserted in pairs underneath the couplers at each side of the connection. The compression rod couplers were anchored to the timber by self-tapping screws in C4 (see Figure 7e). Perpendicular-to-grain reinforcement was applied by the use of self-tapping screws (STS). Three STS were used in the C1 and C2 configurations, while in the C3 and C4, four STS were used at each
- 169 side of the timber beams.

#### 170 **2.2.2. Dimensions and materials**

- 171 The timber beams were made of glulam strength class GL30c [24] and had 215 mm width and 585 mm height. The
- beams were fabricated with 33 mm thick lamellas of Scots pine. The timber specimens were conditioned at the standard
- 173 environment of 20° C and 65 % relative humidity, resulting in approximately 12 % moisture content.
- The purpose-made threaded rods allow fast mounting using rod couplers. The geometry of the rods is shown in Figure 8, and the average outer and inner diameters were d = 22.4 mm and  $d_1 = 16.9$  mm, respectively. The initial 90 mm long part of the rods was manufactured with an M20 metric threaded part. The declared strength class is 8.8. In order to verify the strength of the rods, three tensile tests of the rods were carried out. The obtained mean capacity was 207.6 kN (COV = 0.01), which corresponds to a mean strength of 925 MPa if the average inner diameter is assumed.



Figure 8: Purpose-made long threaded rods used in the investigation (left) and geometry of the wood thread (right). Measures given in mm.

179 The rod couplers were manufactured from steel \$355. The outer and inner diameters were 170 mm and 110 mm,

respectively, and the width was 60 mm. The 28 mm wide slot in the couplers allowed the application of the couplers after the timber parts were brought together. It should be noted that the couplers were generally-suited prototypes designed for the purpose of the experimental investigation. The fitted washers were made of steel S355 and had a width of 60 mm and a height of 70 mm. The thickness was 15 mm at the axis of symmetry.

184

## 185 2.3. Numerical model

- 186 Numerical simulations were carried out by ABAQUS [25]. The geometrical layout of the numerical model corresponds
- 187 to the experimental investigation shown in Figure 7. The splice joint details of the model are shown in Figure 9. Given

188 the symmetry of the problem, only half of the beam was modelled. Transverse displacements of the beam and the rod

189 couplers were prevented. Loading was applied through a displacement rate at the top of the beam, corresponding to the

190 experiments (4-point bending).



Figure 9: Visualisation of the numerical model: (a) detail at the splice joint, (b) steel parts of the joint, (c) detail at the compression coupler, (d) detail at the tension coupler.

191 Eight-node brick elements with reduced integration and hour-glass control (C3D8R) were used in the models. A

192 sensitivity study was carried out in order to determine a satisfactory mesh size. The mesh was denser in the zones

surrounding the rods.

194 The threaded rods and their interaction with timber were based on numerical models presented in [18, 26]. The effective

length of the rods was 1200 mm, and the inner and the outer diameters of wood threads were 16.9 mm and 22.4 mm,

196 respectively (see Figure 8). Contact properties between rods and timber utilised a "hard" contact behaviour in the normal

direction and isotropic tangential behaviour with a coefficient of friction of 0.2, which was based on the study in [27].

198 The anchorage screws of the rod couplers (see Figure 9b) were modelled without threads (by the outer diameter) and

199 with the interaction with the timber realised through a tie-constraint. This simplified approach was shown to be suitable

200 for self-tapping screws in [28].

The relative slip between the rods and the couplers is rigidly constrained for all degrees of freedom, while the relative displacements between the couplers and the anchoring screws are prevented only in the vertical direction (along z-axis with reference to Figure 9).

- 204 The stiffness at the interface of two mutually compressed mating timber end faces is affected by end grain effects. These
- 205 end grain effects are described in [18] by introducing a "crushing zone" in the vicinity of the timber end faces
- characterized by a crushing modulus,  $E_{cr}$ , and a crushing length,  $l_{cr}$ . The contact stiffness between the end timber faces in
- 207 numerical models can thus be modelled by defining a linear elastic contact stiffness at the timber end face with a
- 208 magnitude of  $E_{cr}/l_{cr}$ . Based on the experimental results in [18], the stiffness is here assumed as  $E_{cr}/l_{cr} = 914$  MPa / 3
- 209 mm = 304 MPa/mm (see further description in the Appendix).
- 210 Material properties of wood are summarised in Table 2. Here, E is the modulus of elasticity, G is the shear modulus, v is
- 211 the Poisson's ratio and  $\sigma_y$  represents the yield strength. The longitudinal direction (L) is the grain direction, and no
- 212 distinction is made between tangential (T) and radial (R) directions.

	[MPa]	[MPa]	[MPa]	[MPa]	[-]	[-]
Elastic	$E_L^{(a)}$	$E_R = E_T^{(a)}$	$G_{LR} = G_{LT}^{(b)}$	$G_{RT}{}^{(\mathrm{b})}$	$v_{LR} = v_{LT}^{(b)}$	$v_{TR}^{(b)}$
	13000	400	600	30	0.6	0.315
Yield	$\sigma_{yL}{}^{(\mathrm{b})}$	$\sigma_{yR} = \sigma_{yT}^{(b)}$	$\sigma_{yLR} = \sigma_{yLT}^{(b)}$	$\sigma_{yRT}^{(b)}$		
	23	2.4	3	0.9	_	

213 Table 2: Material properties of wood used in numerical simulations

<sup>(a)</sup> Manufacturer: Moelven industrier ASA, class L40 (GL30c)

<sup>(b)</sup> Estimations based on [29]

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215 Hill's yield surface [30] was applied to wood together with a linear isotropic hardening and associated flow rule. It 216 should be noted that the material model gives no distinction between positive and negative stresses. No damage coupling 217 was defined in order to describe brittle failure in tension and shear. However, as long as the post-elastic behaviour of the 218 joint is governed by the plastic deformations of the steel parts and crushing of the compression zone of timber, the model 219 provides a suitable description of the joint behaviour also in the non-linear domain. The uniaxial strength parameters 220 were thus chosen to represent the compression stress states, and the hardening was formulated to fit the experimental 221 results in [29] in compression parallel to the grain, such that plastic strains of 0.0035 correspond to the stress level of 33 222 MPa.

Steel was modelled as isotropic with E = 210 GPa and v = 0.3. The plastic domain was described by von Mises's yield criterion, linear isotropic hardening and associated flow rule. The yield strength and the hardening formulation of the threaded rods were based on the conducted tensile tests giving the following approximate values of plastic strains at the corresponding stress levels: 0 at 758 MPa and 0.031 at 925 MPa. Corresponding properties for structural steel S355 were used as 0 at 355 MPa and 0.25 at 510 MPa.

## 228 **3. Results and discussion**

#### 229 **3.1. Experimental results**

- 230 Experimental results of bending tests of the splice connections are presented in terms of moment vs. rotation in the joint,
- see the plots in Figure 10.

232



Figure 10: Moment-rotation curves based on data from digital image correlation monitored during bending tests.

233

The ultimate moment  $M_u$  and the elastic rotational stiffness  $k_\theta$  are presented in Table 3. The splice efficiency with respect to moment capacity,  $\eta_M$ , is the ratio between the measured ultimate moment and the mean theoretical bending capacity of the unspliced timber cross-section. The efficiency with respect to rotational stiffness,  $\eta_k$ , relates the mid-span deflection of an unspliced beam (obtained with mean values of material characteristics) to that of a beam containing a splice connection.

239

240 Table 3: Experimental results of bending tests of splice connection

Notation	$M_{\mu}$		$k_ heta$		Foilure mode
	[kNm]	$\eta_M{}^{(a)}$	[kNm/rad]	$\eta_k^{(\mathrm{b})}$	Failule mode
C1-2-1850-0-0	161/169	30 %	14863	52 %	Nut/Steel rods
C2-2-1200-1-0	173	30 %	24952	65 %	Steel rods
C3-3-1200-0-0	241	42 %	29167	68 %	Timber compression
C4-3-1000-1-1	230	40 %	35324	72 %	Rods withdrawal

input: GL30c: <sup>(a)</sup>  $f_{m,mean}$  = 41.4 MPa <sup>(b)</sup>  $E_{mean}$  = 13000 MPa,  $G_{mean}$  = 650 MPa

242 The test of C1 initially failed in a brittle manner due to a nut-thread failure. However, the joint was reassembled and by 243 use of two nuts at the tension side, the new test resulted in the ductile tensile failure of the threaded rods. The plot of C1 244 in Figure 10 represents the first test until the nut-thread failure. The rotational stiffness of C1 is low, compared to C2, 245 which also had 2 rods per row. This is due to the use of standard M20 washers at the connection of the compression rods 246 in the couplers, which did not provide sufficient support in order to prevent rotation of the rod-ends in the couplers. A considerably better rotational stiffness was achieved by the use of fitted internal washers in the compression couplers in 247 248 the C2 configuration. It was, however, observed that even the use of internal fitted washers did not prevent the outward 249 bending of the compression rods under increasing loading (see Figure 11e). The force transfer in the compression rods 250 was thus limited, which in turn resulted in increased deformation of the timber in the compression zone. The failure was 251 finally caused by a tensile rupture of the rods (see Figure 11d). As for C1, the standard M20 washers were also used in 252 the C3 configuration. Insufficient rotational restraint of the compression rod-ends led to bending of the rods and 253 increased utilisation of timber in compression (see Figure 11b and Figure 11c). Compression of timber is very ductile and the test was stopped after reaching large deformations without any significant decrease in capacity. The bending of the 254 255 compression rods was prevented in the C4 configuration by anchoring the compression couplers and the use of internal fitted washers. However, shorter rods were used in C4 and the ultimate failure was caused by the withdrawal of the rods 256 257 in tension. Note that no initial slips were observed in any of the performed tests.

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Figure 11: Joint details during and after tests: (a) C2 during test, (b) compression couplers of C3 after test, (c) failure at compression zone of C3, (d) tensile rupture of tension rods of C2, (e) compression rods of C2 after testing.

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The experimental results demonstrate large rotational stiffness of the timber splice joints by the use of long threaded rods. In agreement with pure withdrawal tests of threaded rods [26], the relation between the effective length and the withdrawal stiffness is non-linear. This implies that increasing the effective length of the rods more than necessary in order to obtain the steel failure has a negligible effect on the withdrawal stiffness. Tensile ruptures of the rods were encountered for rods of 1200 mm effective length in the current investigation. In order to fully utilise the large withdrawal stiffness of the rods, slips between the rods and the couplers should be minimised. In addition, bending of the compression rods can be avoided by anchoring the couplers to the timber.

### 267 **3.2.** Analytical prediction of capacity and stiffness compared to experimental and numerical results

- 268 The moment capacity  $M_u$  and the rotational stiffness  $k_\theta$  of the joint obtained from experiments, numerical models and the 269 proposed analytical model are summarised in Table 4.
- 270 Two sets of results are shown for the analytical model. The first set, denoted as Anal.A, was obtained by the use of the
- 271 withdrawal stiffness of the threaded rods,  $K_w$ , obtained from numerical models. In the second set, Anal.B,  $K_w$  was
- obtained by the model proposed in [26] with the assumption of the so-called pull-shear boundary conditions. While the
- set of results Anal.A gives a good agreement between the analytical and the experimental results in terms of rotational
- 274 stiffness, the set of results Anal.B gives conservative estimates of the rotational stiffness of the joint.
- 275 The rotational stiffness predicted by the numerical models was in a good agreement with the experimental results.

277 Table 4: Experimental, numerical and analytical results

$n_r$	$k_{\theta}$ [kNm/rad]				$M_u$ [kNm]		
	Exp.	Anal.A	Anal.B	Num.	Exp.	Anal.A	Num.
2	24952 (C2)	23227	18780	23755	169/173	188	190
3	35324 (C4)	34044	27513	36522	241/230*	284/243*	286

Input geom.: b = 215 mm,  $h_i = 325$  mm,  $a_i = 415$  mm for lower and -90 mm for upper rods,

 $d = 22.4 \text{ mm}, d_l = 16.9 \text{ mm}, l_f = 45 \text{ mm}, {}^*l_{ef} = 1000 \text{ mm}, \text{ otherwise } l_{ef} = 1200 \text{ mm}$ 

278

## 279 Material characteristics used as an input to the analytical models (mean values):

$K_w$	243 kN/mm	Withdrawal stiffness of threaded rods obtained by numerical model
$K_w$	176 kN/mm in Anal.B	Withdrawal stiffness of threaded rods obtained by model in [26]
Ε	13 000 MPa	MOE of timber parallel to the grain (see Table 2)
$E_s$	210 GPa	MOE of steel
$E_{cr}$	914 MPa	Crushing modulus [18] (see Appendix)
$l_{cr}$	3 mm	Crushing length [18] (see Appendix)
$k_t$	710 MPa	Foundation modulus of timber transverse to the grain [31, 32] (see Appendix)
$k_l$	1300 MPa	Foundation modulus of timber longitudinal to the grain [31, 32] (see Appendix)
$K_{co}$	450 kN/mm	Stiffness of the rod coupler obtained by numerical model (see Appendix)
$R_u$	207.6 kN	Tensile capacity of the threaded rods obtained by tensile tests (see 2.2.2)
<i>R</i> <sub>ax</sub>	Var. for var. rod lengths	Withdrawal strength of the threaded rods under pure axial loading by model in [33]

280

281 The experimentally obtained ultimate moments were affected by bending of the compression rods in the configurations 282 C1, C2 and C3 due to insufficient stiffness between the rods and the couplers, while the withdrawal of the tension rods 283 limited the capacity in the C4 configuration. The analytical and numerical models assume no slip between the rods and the couplers, and the predicted moment capacity is, therefore, higher compared to the experimental results. A good 284 285 mutual agreement is obtained for the moment capacity predicted by the analytical and the numerical models. The 286 ultimate load in the numerical model was governed by a combination of a full utilization of the capacity of the rods in 287 tension and bending of the compression rods together with a "plastification" of the timber compression zone. Note that 288 the numerical models were not formulated to predict the withdrawal failure of the rods, which was the failure mode in the 289 C4 configuration. The moment capacity predicted by the numerical and the analytical models can thus be interpreted as a 290 maximum moment capacity of the joint, assuring that the outward bending of the compression rods is prevented. 291 The moment-rotation curves obtained by the numerical simulations are presented together with the experimental results 292 (for configurations containing the internal fitted washers) in Figure 12. The numerical simulations were performed both 293 with the compression rod couplers anchored to the timber, i.e. in correspondence with the results in Table 4 (curves

denoted as num.Alt.I) and without anchoring the compression couplers (curves denoted as num.Alt.II). The numerical

295 results show that, by anchoring the compression couplers, the moment capacity is increased by 5 %, while the anchorage 296 has a negligible effect on the rotational stiffness of the joint. When comparing the numerical and the experimental results 297 in the case of 2 rods per row, it is observed that the moment capacity of the joint can be enhanced by 5 % if the slip 298 between the rods and the couplers is entirely prevented and by an additional 5 % by anchoring the couplers. In the case of 299 3 rods per row, the couplers were anchored in the experimental setup (C4) and the failure was caused by the withdrawal 300 of the tension rods. It is seen that by providing a sufficient effective length of the rods, an increase in moment capacity by 301 24 % can be achieved. With reference to Table 3, the joint efficiency, in terms of capacity, is thus improved from 40 % to 302 50 % in the configuration with 3 rods per row.



Figure 12: Moment-rotation curves obtained by the numerical models compared to the experimental results.

303

As discussed in Section 2.2, the rod couplers used in the current investigation were generally-suited prototypes not optimised for the particular joint layout. A further optimisation of the design will probably allow a reduction of the size of the couplers, leading to smaller openings in the timber beams. If the height of the openings is reduced from the current 130 mm to 80 mm, the analytical model shows an increase of the rotational stiffness and the moment capacity by 7 % and 1%, respectively.

309

#### 310 **3.3.** Combined action of bending moment and normal force

The capacity under the combined action of bending moment and normal force was studied by use of numerical models for the configuration of 2 rods per row. The results of the analyses together with the analytical solutions are shown in Figure 13 by the interaction diagrams of bending moment M and normal force N. In this study, two sets of analyses by numerical models were performed. In the first set, the rod couplers were anchored to the timber members both at the tension and the compression side (results denoted as Num.Anch.). In the second set, no interaction between the couplers and the timber was applied (results denoted as Num.Unanch.). The analytical solution was obtained by the use of Eq. (7). In the case of the "anchored" set of analyses,  $N_u$  and  $M_u$  were obtained by Eqs. (8) and (3), respectively. If the rod couplers are not anchored to the timber, the rods can be bended and the force transfer through the rods is limited. The analytical solution shown in Figure 13 for the case of "unanchored" couplers was thus obtained for  $N_u$  determined by Eq. (8), disregarding the contribution from the rods, but assuming the maximum compression strength of timber  $f_{c,0}$ .  $M_u$  was obtained by the use of Eq. (3) with disregard of the compression rods. The input parameters to the analytical models are summarised in Section 3.2 ( $K_w$  from numerical models was used herein).



Figure 13: Interaction M-N diagram for joint with 2 rods per row.

The ultimate load modes in the numerical models differ in dependence on the ratio of M/N. For higher ratios of M/N, the ultimate load is governed by a full utilisation of the axial capacity of the rods in tension. For lower ratios of M/N, the ultimate load is governed by bending of the rod in compression together with full utilization of timber in compression. It should be noted that the numerical models were not formulated to predict the brittle failures in timber in direction perpendicular to the grain and the shear failures. Therefore, the obtained results should be interpreted with caution since they likely represent an upper bound. Nevertheless, Figure 13 indicates that the proposed simple analytical interaction model enables a reasonable prediction of the joint capacity under combined action of bending moment and normal force.

330

## 331 **4. Concluding remarks**

332 The present investigation shows that splicing of massive glulam sections can effectively be achieved by use of inclined333 long threaded rods. The experimentally investigated prototype joint featured large rotational stiffness without any initial

334 slip. It was shown that the boundary conditions at the rod-ends affect both the rotational stiffness and the moment 335 capacity considerably. The reported joint moment capacity appear to be sufficient for the intended joint application in 336 timber network arch bridges, for which the design is governed by a normal force [4]. In addition, the location of the joint 337 can be optimized with respect to extreme values of bending moments in the arch.

For practical joint design, analytical relations validated by experiments and numerical models are proposed for the determination of the rotational stiffness and moment capacity. In addition, a simple model for predicting the capacity under combined action of bending moment and normal force is proposed and compared to the results obtained by numerical models. Note that the formulation of the numerical models did not allow capturing all failure modes. Therefore, care should be taken when interpreting the numerical results.

The rod inclination to grain prevents the loss of capacity in case of shrinkage cracks in close proximity to the threaded rods. In the current investigation, the rod-to-grain angle was 5 degrees, which is considered appropriate in order to bridge possible shrinkage cracks, as well as to maintain the large withdrawal stiffness and capacity of the rods. By providing sufficient effective length, the failure mode is ductile, due to yielding of the rods. As shown in [34], under the combined action of axial and lateral loads, both the capacity and the initial stiffness of the threaded rods are reduced for both increased rod-to-grain and load-to-rod angles.

349 In timber network arch bridges, the bending moment acting in the joint will change orientations for different traffic load 350 positions and different directions of wind loads. In addition, a considerable normal force will act in the connection, 351 transmitted through both the contact of the timber end faces and the rods. Hence, the layout of the connection must be 352 symmetric (i.e. the rod detailing at both sides of the connection must allow the transfer of both tension and compression) 353 and the timber end faces must be in mutual contact provided by tightening of the rods during assembly. It should be 354 noted that the scope of the present work did not cover the fatigue resistance. Yet, the preliminary results of fatigue tests 355 of axially loaded threaded rods indicate that the fatigue resistance of timber is large compared to steel, and the fatigue 356 capacity of joints with this type of connector is thus governed by the resistance of the steel components [35, 36].

The proposed splicing technique utilises long threaded rods embedded with a small inclination to the grain. The hygroscopic deformations in timber normal grain are thus not restrained, resulting in a favourable solution regarding the moisture induced stresses in the joint. However, self-tapping screws embedded perpendicular to the grain were used to reinforce the joint and to anchor the couplers to the timber, which may initiate moisture induced cracks [37, 38]. The self-tapping screws used in the current investigation were not optimised and further investigations should be carried out to eliminate the possible effect on the development of the moisture induced cracks.

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367 Appendix

#### 368

### 369 Input parameters to the expressions presented in Sections 2.1.1 and 2.1.2

370 The equivalent length of the timber compression (crushing) zone at each side of the contact interface,  $l_c$ , is given by [18]:

$$l_c = 0.85 \cdot h_t + l_{cr} \cdot \frac{E}{E_{cr}}$$
(9)

where  $h_t$  is the height of timber end faces in mutual contact (see Figure 3), and  $E_{cr}$  and  $l_{cr}$  are the crushing modulus and the crushing length, respectively.

374 The first term in Eq. (9) characterizes the compliance of the wood in compression and approximately defines a local area, 375 in close proximity to the splice joint, for which the Navier's hypothesis is violated and where the stress state is a 376 combination of compression stresses parallel to the grain, tensile stresses perpendicular to the grain and shear stresses. 377 The second term characterizes the deformation at the end grains of two mutually compressed mating timber end faces 378 and it is derived by introducing a crushing zone characterized by a crushing modulus,  $E_{cr}$ , and a crushing length,  $l_{cr}$ . The 379 parameters of the crushing zone were obtained experimentally. Based on the experimental results performed for glulam 380 class GL30c, it is suggested in [18] to use  $l_{cr} = 3$  mm with the corresponding  $E_{cr} = 914$  MPa, yielding the stiffness of the 381 crushing zone  $E_{cr} / l_{cr} \approx 300 \text{ N} / \text{mm}^3$ .

382 The stiffness of the i-th rod row,  $K_{si}$ , given by Eq. (10), is obtained as a system of three springs in series representing: 1) 383 the stiffnesses of the threaded rods in the direction of the applied force at each side of the connection denoted as  $K_{p}$ , and, 384 2) the stiffness of the rod coupler denoted as  $K_{co}$ . The number of steel rods in one row equals  $n_r$ .

$$K_{si} = n_r \cdot \frac{K_p \cdot K_{co}}{2 \cdot K_{co} + K_p}$$
(10)

The stiffness of the threaded rods in the direction of the applied force (parallel to the grain),  $K_p$ , is affected by the rod-tograin and load-to-rod angles, and by the boundary conditions at the rod-end. These effects were studied and analytical 388 relations for the stiffness prediction were derived in [34]. In the following, the relations associated with the actual 389 geometrical layout are presented.

390 The rod stiffness in the direction of the applied force  $K_p$  is found as the interaction of the axial rod stiffness  $K_{ax}$  and the 391 lateral rod stiffness  $K_v$  at the rod-end by [34]:

392 
$$K_{p} = \frac{K_{ax} \cdot K_{v}}{K_{ax} \cdot \sin^{2}(\gamma) + K_{v} \cdot \cos^{2}(\gamma)}$$
(11)

393 If the displacements transverse to the load direction are prevented (by anchoring the coupler to the timber), the rod
394 stiffness is determined by [34]:

395 
$$K_p = K_{ax} \cdot \cos^2(\gamma) + K_v \cdot \sin^2(\gamma)$$
(12)

396 The stiffness in the axial direction  $K_{ax}$  of the rod is found by [34]:

$$K_{ax} = \frac{K_w \cdot K_{ax,f}}{K_w + K_{ax,f}}$$
(13)

where  $K_w$  is the withdrawal axial stiffness of the threaded rod. The axial stiffness of the free part of the rod not embedded in timber is determined as  $K_{ax,f} = E_s \cdot A_s / l_f$ , where  $l_f$  is the length of the free part of the rod between the timber member and the rod fastening in the coupler,  $E_s$  is the elastic modulus of steel and  $A_s$  is the cross-sectional area of the rods determined with the core diameter of the rods  $d_l$ .

402 Under the assumption of a rotational restraint at the rod-end implied by a connection in the couplers, the lateral stiffness 403 at the rod-end,  $K_{\nu}$ , is found by [34]:

404 
$$K_{\nu} = \frac{12kEI(4EI\lambda^{3} + kl_{f})}{48(EI)^{2}\lambda^{4} + 8kEI\lambda l_{f}(2l_{f}^{2}\lambda^{2} + 3l_{f}\lambda + 3) + k^{2}l_{f}^{4}}$$
(14)

405 where *EI* is the flexural stiffness of the rods obtained as  $EI = E_s \cdot \pi \cdot d_1^4 / 64$ , the parameter  $\lambda$  is found as  $\lambda = \sqrt[4]{k / (4EI)}$ 406 and *k* is the foundation modulus of timber obtained by the interaction of foundation moduli of timber longitudinal to the 407 grain,  $k_l$ , and transverse to the grain,  $k_l$ , as [34]:

408 
$$k = \frac{k_l \cdot k_t}{k_l \cdot \cos^2(\gamma) + k_t \cdot \sin^2(\gamma)}$$
(15)

409 The withdrawal strength of the rods is reduced compared to pure axial loading due to the inclination of the rods and the

410 imposed rotational restraint at the rod-end (the rods are loaded by a combination of a lateral force and a bending

411 moment). Similar to the model proposed in [18], the ultimate withdrawal strength of the rods  $R_{axu}$  is found by a linear

412 reduction of the effective length of the threaded rods:

$$R_{axu} = R_{ax} \cdot \left(\frac{l_{ef} - l_x}{l_{ef}}\right)$$
(16)

414 where  $R_{ax}$  is the withdrawal strength of the threaded rods under pure axial loading ( $R_{ax}$  can be determined for the

415 effective length of the threaded rods, *l<sub>ef</sub>*, by the use of the model proposed in [33] with the assumption of the so-called

416 pull-shear boundary conditions), and  $l_x$  is a free length of the rod that is not considered to contribute to the withdrawal

417 capacity.

413

418 The free length of the rod,  $l_x$ , can be determined in the same manner as in [18] by assuming an interaction of lateral force

419 and bending moment and respecting the actual boundary conditions at the rod-end. However, the exact analytical solution

420 becomes unhandy to use and the following conservative simplification is thus proposed:

421 
$$l_x = \pi \cdot d_1 \cdot \sqrt[4]{\frac{\pi \cdot E_s}{k}}$$
(17)

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