

Master's thesis

NTNU
Norwegian University of Science and Technology
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Department of Mechanical and Industrial Engineering

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Enhancing the Analysis Possibilities of Fedem by Performing Model Reduction in ANSYS

Master's thesis in Produktutvikling og produksjon

Supervisor: Terje Rølvåg

June 2019



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Abstract

When performing dynamic analyses, model reduction is a method used that will reduce the computational cost and storage requirements in finite element analysis programs. The dynamic analysis software Fedem uses the same model reduction techniques as the finite element analysis software ANSYS. As ANSYS has a wider range of element types available for analyses, Fedem would benefit from utilizing this when performing dynamic analyses.

To enable this integration between Fedem and ANSYS, the ANSYS ACT extension "ModRed" has been developed. It offers the possibility of calculating and exporting the matrices needed by Fedem for performing dynamic analyses. In order to test performance of the extension, CMS model reduction of the same models has been performed in both ANSYS and Fedem for investigating the equality of the resulting matrices.

The results show that the full mass matrices of both systems are nearly identical, but the reduced mass matrices show poor similarity, with 10% to 15% difference from the Fedem reference matrix. The calculated gravity vectors give very varied equality to the used Fedem reference; from being close to identical to having a 36.5% difference in the worse test case.

As an issue in partitioning ANSYS matrices has been discovered, and since this issue is believed to be a cause for the varied results, it is too early to make a conclusion on the accuracy acquired from the extension. More testing using a wider range of test models and element types, as well as resolving the issue of matrix partitioning is needed. Despite varying results in testing, the ModRed extension has proven to be a valuable and easy-to-use method for enhancing the analysis capabilities of Fedem.

Sammendrag

Ved utførelse av dynamisk analyse er modellreduksjon en metode ofte tatt i bruk. Denne metoden vil redusere utregningstiden og lagringsbehovene for elementmetode-program. Dynamisk analyse-programvaren Fedem benytter de samme teknikkene for modellreduksjon som det som blir brukt i elementmetode-programmet ANSYS. Ettersom ANSYS har et større utvalg av elementtyper tilgjengelig, vil det kunne være nyttig for Fedem å utnytte dette i dynamiske analyser.

ANSYS ACT-programtillegget "ModRed" er blitt utviklet for å gjøre en slik integrering mellom ANSYS og Fedem mulig. Programtillegget gjør det mulig å regne ut og eksportere de nødvendige matrisene som skal til for at Fedem skal kunne utføre dynamiske analyser. For å teste ytelsen til programtillegget, har en CMS-modellreduksjon av identiske modeller blitt utført i både ANSYS og Fedem, og de resulterende matrisene er blitt sammenliknet.

Resultatene viser at de fulle massematrisene fra begge systemer er så godt som identiske. Det viser seg imidlertid at de reduserte massematrisene kun i liten grad er like, med 10% til 15% forskjell fra Fedems referansematrix. De utregnede gravitasjonsvektorene viser svært varierende likhet til referansen fra Fedem; fra å være tilnærmet identiske, til å være 36.5% forskjellige i tilfellet med størst forskjell.

Ettersom et problem med partisjonering av ANSYS-matriser har blitt oppdaget, og siden dette antas å være en årsak til de varierende resultatene, er det for tidlig å fastslå nøyaktigheten man kan oppnå ved å bruke programtillegget. Ytterligere testing med bruk av et større utvalg test-modeller og element-typer, samt å rette opp i matrise-partisjonerings-problemet er nødvendig. Til tross for varierende resultater ved testing, har ModRed-programtillegget vist seg å være en verdifull metode som det er lett å ta i bruk for å utvide Fedems muligheter for analyse.

Preface

This master's thesis is the product of work done during the spring term in 2019 as the final part of a five-year master's degree in mechanical engineering. The thesis is written at the Norwegian University of Science and Technology (NTNU) in Trondheim, at the Department of Mechanical and Industrial Engineering.

The project was chosen based on my own interest in both mechanical engineering and software development. As this project would allow me to explore both fields, the potential learning outcome was a key driver. I would like to thank my supervisor, Professor Terje Rølvåg from the Department of Mechanical and Industrial Engineering for introducing me to the project and providing constructive feedback. In addition, I would like to thank Knut Morten Okstad from Fedem Technology for valuable support on the Fedem software.

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Adrian Opheim

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Abbreviations

ACT	=	ANSYS Customization Toolkit
APDL	=	ANSYS Parametric Design Language
CAD	=	Computer Aided Design
CAE	=	Computer Aided Engineering
CMS	=	Component Mode Synthesis
DOF	=	Degree of Freedom
FE	=	Finite Element
FEM	=	Finite Element Method
GUI	=	Graphical User Interface
WBEX	=	WorkBench Extension
XML	=	Extensible Markup Language

Chapter 1

Introduction

When performing dynamic analyses of structures, similar repeated matrix calculations must be performed for every time step in the analysis. Dynamic analyses are therefore generally much more expensive in terms of computational time and storage space than performing static analyses. In order to reduce the cost of performing dynamic analyses, model reduction methods that reduce the number of degrees of freedom in the system may be applied. Guyan reduction and the component mode synthesis method are examples of such methods. The finite element software ANSYS have both these methods implemented in its APDL solver. In order to access these methods through the ANSYS Mechanical software, an ANSYS ACT extension must be developed, as model reduction techniques are currently not available through ANSYS Mechanical.

The dynamic analysis software Fedem uses the same model reduction techniques as ANSYS. Due to the extensive finite element library available, it is of interest to perform model reduction in ANSYS. The reduced models may thereafter be imported to Fedem for further dynamic analyses. This thesis will present such an extension that enables this integration.

Chapter 2

Theory

2.1 The Finite Element Method in Linear Dynamics

When analyzing continuous problems, very few of them have an exact, closed-form solution, implicating that discretization techniques should be applied in order to make an approximated system. The behavior of the discretized system could then be computed and used as an approximation for the continuous system. The finite element method is an discretization method that has become the preferred computational method for analyzing structures. Dividing the a structure into a finite number of elements, then assembling them while keeping continuity in the displacements of connected elements for representing the global displacements is the core principle of the finite element method.

For illustrating the principle of the finite element method, we consider a bar of length L in axial deformation subjected to axial loads P_1 and P_2 . The bar is divided into N finite elements of length l , as shown in Figure 2.1.

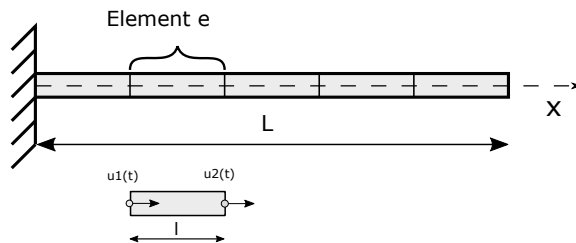


Figure 2.1: Bar in axial deformation

For each element, the displacement field $u(x, t)$ is linearly interpolated by

$$u(x, t) = u_1(t)\phi_1(x) + u_2(t)\phi_2(x) \tag{2.1}$$

where $\phi_1(x)$, $\phi_2(x)$ are so-called shape functions of the element. The shape functions are chosen in such a way that they fulfill the boundary conditions

$$u(0, t) = u_1(t) \quad u(l, t) = u_2(t)$$

u_1 and u_2 are axial displacements at the element ends, referred to as nodes.

For the bar in axial deformation, the shape functions are linear, given as

$$\phi_1(x) = 1 - \frac{x}{l} \quad \phi_2(x) = \frac{x}{l} \quad (2.2)$$

(2.1) can be written in matrix form

$$u(x, t) = \mathbf{F}_e(x) \mathbf{q}_e(t) \quad (2.3)$$

where x must be within the element e , and

$$\begin{aligned} \mathbf{F}_e(x) &= [\phi_1(x) \quad \phi_2(x)] \\ \mathbf{q}_e(t) &= [u_1(t) \quad u_2(t)]^T \end{aligned} \quad (2.4)$$

Using this, we can compute the elementary mass and stiffness matrices \mathbf{M}_e and \mathbf{K}_e given by

$$\begin{aligned} \mathbf{M}_e &= \int_0^l m \mathbf{F}_e^T \mathbf{F}_e dx \\ \mathbf{K}_e &= \int_0^l EA \frac{d\mathbf{F}_e^T}{dx} \frac{d\mathbf{F}_e}{dx} dx \end{aligned} \quad (2.5)$$

For the axial bar, we obtain the matrices

$$\mathbf{K}_e = \frac{EA}{l} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \quad \mathbf{M}_e = \frac{ml}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix} \quad (2.6)$$

The force element vector \mathbf{g}_e is equal to to the end loads of the element:

$$\mathbf{g}_e^{(1)} = \begin{bmatrix} P_1 \\ P_2 \end{bmatrix} \quad (2.7)$$

2.1.1 Assembling elements to the global structure

For expressing dynamic equilibrium for whole bar, we need to know the topology of every element e : Where is it located with respect to the global model and with respect to the other elements of the structure? This is solved by a topology matrix \mathbf{L}_e . It is constructed so that

$$\mathbf{q}^T = \mathbf{L}_e \mathbf{q} \quad (2.8)$$

where \mathbf{q} is a matrix containing all $(N + 1)$ nodal displacements:

$$\mathbf{q} = [u_0 \quad u_1 \quad u_2 \cdots u_N]^T$$

For instance, element 1 and 2 of Figure 2.1 have the topology matrix

$$\begin{aligned} \mathbf{L}_1 &= \begin{bmatrix} 0 & 0 & 0 & 0 & \cdots & 0 \\ 0 & 1 & 0 & 0 & \cdots & 0 \end{bmatrix} \\ \mathbf{L}_2 &= \begin{bmatrix} 0 & 1 & 0 & 0 & \cdots & 0 \\ 0 & 0 & 1 & 0 & \cdots & 0 \end{bmatrix} \end{aligned} \quad (2.9)$$

Using the topology matrix, we can assemble all N elements into a global system, and we can then define

- The mass matrix of the assembled system:

$$\mathbf{M} = \sum_{e=1}^N \mathbf{L}_e^T \mathbf{M}_e \mathbf{L}_e \quad (2.10)$$

- The stiffness matrix of the assembled system:

$$\mathbf{K} = \sum_{e=1}^N \mathbf{L}_e^T \mathbf{K}_e \mathbf{L}_e \quad (2.11)$$

- The load vector of the assembled system:

$$\mathbf{g} = \sum_{e=1}^N \mathbf{L}_e^T \mathbf{g}_e \quad (2.12)$$

This corresponds to an assembly as seen in Figure 2.2. From the figure, one can observe that

- The diagonal terms of the stiffness and mass matrix add up two and two along the diagonal of the matrix.
- The shaded zone correspond to the clamped end of the bar, and could therefore be set to 0.

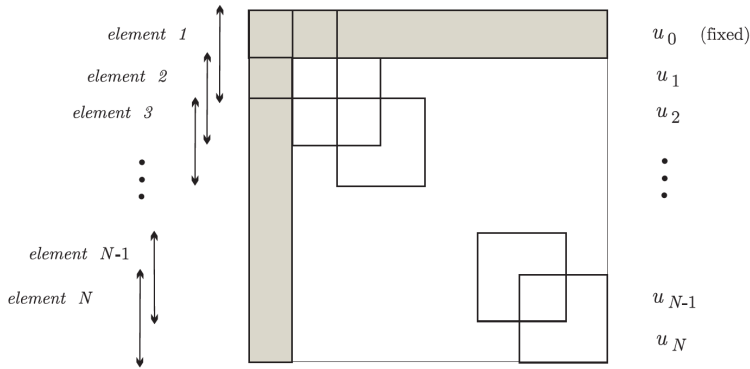


Figure 2.2: Assembly of the global matrices

For the axially loaded bar, the global structural matrices become

$$\mathbf{K} = \frac{EA}{l} \begin{bmatrix} 2 & -1 & & & & \\ -1 & 2 & -1 & & & \mathbf{0} \\ & -1 & 2 & \ddots & & \\ & & \ddots & \ddots & -1 & \\ \mathbf{0} & & & -1 & 2 & -1 \\ & & & & -1 & 1 \end{bmatrix} \quad (2.13)$$

$$\mathbf{M} = \frac{ml}{6} \begin{bmatrix} 4 & 1 & & & & \\ 1 & 4 & 1 & & & \mathbf{0} \\ & 1 & 4 & \ddots & & \\ & & \ddots & \ddots & 1 & \\ \mathbf{0} & & & 1 & 4 & 1 \\ & & & & 1 & 2 \end{bmatrix} \quad (2.14)$$

2.2 Reduction Methods for Dynamic Problems

When a finite element analysis of a static system is performed, static deformations and stress levels in small details of the system is usually of interest. The static FEM models will therefore usually have a highly refined mesh and a corresponding very high number of degrees of freedom. One are able to solve these models fairly efficiently with static solvers, but if one wishes to find the dynamic behavior of the system, the computing time required is often unacceptable. However, the highly refined meshes used in static analyses are often not needed to capture the dynamic behavior of the system. If one were to find the free vibration modes of the system, we know that the first free vibration modes have a rather smooth deformation, meaning that a coarser mesh would be sufficient to capture these

modes. The reduction methods for dynamic problems addresses this problem. Because creating a fine mesh that captures the required static solutions is an important and time-consuming part of FE analysis, the reduction methods do not modify the fine mesh created for static analyses, but instead reduces the size of the dynamic problem to solve.

Model reduction can in general be expressed as

$$\mathbf{v} = \mathbf{H}\mathbf{q} \quad (2.15)$$

where \mathbf{v} is the full set of degrees of freedom for the fine mesh and is of size $(n \times 1)$. \mathbf{H} is the reduction matrix and \mathbf{q} is the reduced set of displacements used for capturing the dynamic behavior and is of size $(m \times 1)$. The aim for the model reduction is to achieve $m \ll n$ without a significant loss in accuracy for the stress results.

The matrix equation that governs system dynamics is expressed as

$$\mathbf{M}\ddot{\mathbf{v}} + \mathbf{K}\mathbf{v} = \mathbf{Q} \quad (2.16)$$

if one neglects damping. When reduction methods are applied to the system equations, the equation is partitioned as follows

$$\begin{bmatrix} \mathbf{M}_{ee} & \mathbf{M}_{ei} \\ \mathbf{M}_{ie} & \mathbf{M}_{ii} \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{v}}_e \\ \ddot{\mathbf{v}}_i \end{bmatrix} + \begin{bmatrix} \mathbf{K}_{ee} & \mathbf{K}_{ei} \\ \mathbf{K}_{ie} & \mathbf{K}_{ii} \end{bmatrix} \begin{bmatrix} \mathbf{v}_e \\ \mathbf{v}_i \end{bmatrix} = \begin{bmatrix} \mathbf{Q}_e \\ \mathbf{Q}_i \end{bmatrix} \quad (2.17)$$

where subscript e represents external nodes and subscript i represents internal nodes. External nodes in the model will typically be chosen to be joints, springs, dampers, external loads, control input, points of interest, etc. There should be as few as possible external nodes to reduce the simulation model as much as possible. As will be shown, the reduction methods will eliminate the internal nodes from the FE model.

From (2.17), the stiffness relation for a sub-structure can be written as

$$\begin{bmatrix} \mathbf{K}_{ee} & \mathbf{K}_{ei} \\ \mathbf{K}_{ie} & \mathbf{K}_{ii} \end{bmatrix} \begin{bmatrix} \mathbf{v}_e \\ \mathbf{v}_i \end{bmatrix} = \begin{bmatrix} \mathbf{Q}_e \\ \mathbf{Q}_i \end{bmatrix} \quad (2.18)$$

Writing out the equations gives

$$\begin{aligned} \mathbf{K}_{ee}\mathbf{v}_e + \mathbf{K}_{ei}\mathbf{v}_i &= \mathbf{Q}_e \\ \mathbf{K}_{ie}\mathbf{v}_e + \mathbf{K}_{ii}\mathbf{v}_i &= \mathbf{Q}_i \end{aligned} \quad (2.19)$$

Solving the second equation for \mathbf{v}_i gives

$$\begin{aligned} \mathbf{v}_i &= \mathbf{K}_{ii}^{-1}\mathbf{Q}_i - \mathbf{K}_{ii}^{-1}\mathbf{K}_{ie}\mathbf{v}_e \\ &= \mathbf{K}_{ii}^{-1}\mathbf{Q}_i + \mathbf{B}\mathbf{v}_e \end{aligned}$$

where \mathbf{B} is denoted as the influence matrix, given by $\mathbf{B} = -\mathbf{K}_{ii}^{-1}\mathbf{K}_{ie}$.

The internal displacements \mathbf{v}_i can then be expressed as

$$\mathbf{v}_i = \mathbf{v}_i^{\text{dyn}} + \mathbf{v}_i^{\text{stat}} \quad (2.20)$$

where $\mathbf{v}_i^{\text{dyn}}$ represents internal displacements with external DOFs fixed, the dynamic part of the internal node's displacements

$$\mathbf{v}_i^{\text{dyn}} = \mathbf{K}_{ii}^{-1} \mathbf{Q}_i \quad (2.21)$$

and $\mathbf{v}_i^{\text{stat}}$ represents internal displacements as a function of external displacements, called the "static" part, because the internal nodes respond quasi-statically to the external nodes' displacements.

$$\mathbf{v}_i^{\text{stat}} = -\mathbf{K}_{ii}^{-1} \mathbf{K}_{ie} \mathbf{v}_e = \mathbf{B} \mathbf{v}_e \quad (2.22)$$

When applying model reduction techniques, two methods are commonly used, namely Guyan reduction and Component Mode Synthesis (CMS). The two methods differ in how they are expressing the internal displacements \mathbf{v}_i . When applying Guyan reduction, the "dynamic" part $\mathbf{v}_i^{\text{dyn}}$ is neglected, and the reduction is based purely on the static response of the internal nodes from displacements at the external nodes, while the Component Mode Synthesis method uses both the dynamic and static part of the internal nodes' displacements

2.2.1 Guyan Reduction

Guyan reduction utilizes the fact that the quasi-static response of the internal nodes is often sufficient for describing the substructure's displacements. The reduction method neglects the dynamic part of the internal node's displacements, thus setting $\mathbf{v}_i^{\text{dyn}} = \mathbf{0}$. It is also assumed that no forces are applied on the internal nodes, $\mathbf{Q}_i = \mathbf{0}$. The reduction is then built by

$$\mathbf{v} = \begin{bmatrix} \mathbf{v}_e \\ \mathbf{v}_i \end{bmatrix} = \mathbf{H}_{\text{Guyan}} \mathbf{v}_e = \begin{bmatrix} \mathbf{I} \\ \mathbf{B} \end{bmatrix} \mathbf{v}_e \quad (2.23)$$

where \mathbf{I} is the identity matrix, and $\mathbf{B} = -\mathbf{K}_{ii}^{-1} \mathbf{K}_{ie}$ is the influence matrix.

The reduced stiffness and mass matrices from the Guyan reduction, respectively $\mathbf{k}_{\text{Guyan}}$ and $\mathbf{m}_{\text{Guyan}}$, are then found by premultiplying (2.16) with $\mathbf{H}_{\text{Guyan}}^T$ and inserting (2.23) for \mathbf{v} . The reduced stiffness and mass matrices are then

$$\begin{aligned} \mathbf{k}_{\text{Guyan}} &= \mathbf{H}_{\text{Guyan}}^T \mathbf{K} \mathbf{H}_{\text{Guyan}} \\ &= \mathbf{K}_{ee} - \mathbf{K}_{ei} \mathbf{K}_{ii}^{-1} \mathbf{K}_{ie} \\ \mathbf{m}_{\text{Guyan}} &= \mathbf{H}_{\text{Guyan}}^T \mathbf{M} \mathbf{H}_{\text{Guyan}} \\ &= \mathbf{M}_{ee} - \mathbf{M}_{ei} \mathbf{K}_{ii}^{-1} \mathbf{K}_{ie} - \mathbf{K}_{ei} \mathbf{K}_{ii}^{-1} \mathbf{M}_{ie} + \mathbf{K}_{ei} \mathbf{K}_{ii}^{-1} \mathbf{M}_{ii} \mathbf{K}_{ii}^{-1} \mathbf{K}_{ie} \end{aligned} \quad (2.24)$$

Using this, the dynamic equation for the whole system (2.16) can be reduced to the following for an undamped system

$$\mathbf{m}_{\text{Guyan}} \ddot{\mathbf{v}}_e + \mathbf{k}_{\text{Guyan}} \mathbf{v}_e = \mathbf{Q}_e \quad (2.25)$$

This results tells us that the whole system now is being represented only by the external node's accelerations, displacements and loads. Applying the Guyan reduction technique will therefore reduce the number of degrees of freedom in the system drastically.

If the Guyan reduction is applied to static problems, the exact solution is found. It is also computationally efficient, thus being a widely used reduction technique. However, if the structure has a dynamic behaviour, an approximation is introduced by neglecting the dynamic response of the internal nodes.

2.2.2 CMS Model Reduction

The Component Mode Synthesis (CMS) model reduction was introduced by Craig and Bampton [1] in 1968. The method is therefore also called the "Craig-Bampton Method". When applying CMS reduction, the term $\mathbf{v}_i^{\text{dyn}}$ is no longer neglected. Instead it is approximated by a linear combination of eigenvalues. In the following, we assume that we originally have n degrees of freedom. By partitioning through (2.17), we get p external and $n - p$ internal degrees of freedom.

In order to approximate $\mathbf{v}_i^{\text{dyn}}$, we remember that $\mathbf{v}_i^{\text{dyn}}$ correspond to the displacements in the substructure when the external degrees of freedom are fixed. This corresponds the case where

$$\mathbf{Q}_i = \mathbf{v}_e = \mathbf{0} \quad (2.26)$$

Inserting this into (2.17) gives the equation

$$\mathbf{M}_{ii}\ddot{\mathbf{v}}_i^{\text{dyn}} + \mathbf{K}_{ii}\mathbf{v}_i^{\text{dyn}} = \mathbf{0} \quad (2.27)$$

When considering simple harmonic motion, the displacement $\mathbf{v}_i^{\text{dyn}}$ may be expressed as

$$\mathbf{v}_i^{\text{dyn}} = \boldsymbol{\phi} \sin \omega t \quad (2.28)$$

where $\boldsymbol{\phi}$ is the eigenvector defined by the eigenvalue problem

$$(\mathbf{K}_{ii} - \omega^2 \mathbf{M}_{ii})\boldsymbol{\phi} = \mathbf{0} \quad (2.29)$$

It is possible to describe an arbitrary displacement as a linear combination of the $(n - p)$ eigenmodes. Using a selection s of the eigenmodes, $\mathbf{v}_i^{\text{dyn}}$ can be approximated as

$$\mathbf{v}_i^{\text{dyn}} = \sum_{k=1}^s \boldsymbol{\phi}_k y_k = \boldsymbol{\Phi} \mathbf{y} \quad s < n - p \quad (2.30)$$

where

$$\boldsymbol{\Phi} = [\boldsymbol{\phi}_1 \quad \boldsymbol{\phi}_2 \quad \cdots \quad \boldsymbol{\phi}_s] \quad (2.31)$$

is the eigenvector matrix with dimensions $(n - p) \times s$.

Using this result, the displacement vector \mathbf{v} can be expressed as

$$\mathbf{v} = \begin{bmatrix} \mathbf{v}_e \\ \mathbf{v}_i \end{bmatrix} = \begin{bmatrix} \mathbf{I} & \mathbf{0} \\ \mathbf{B} & \boldsymbol{\Phi} \end{bmatrix} \begin{bmatrix} \mathbf{v}_e \\ \mathbf{y} \end{bmatrix} = \mathbf{H}_{\text{CMS}} \begin{bmatrix} \mathbf{v}_e \\ \mathbf{y} \end{bmatrix} \quad (2.32)$$

When combining the substructure dynamic equation (2.17) with the time derivatives of (2.32) and pre-multiplying with $\mathbf{H}_{\text{CMS}}^T$ we get

$$\begin{bmatrix} \mathbf{m}_{11} & \mathbf{m}_{12} \\ \mathbf{m}_{21} & \mathbf{I} \end{bmatrix} \begin{bmatrix} \ddot{\mathbf{v}}_e \\ \dot{\mathbf{y}} \end{bmatrix} + \begin{bmatrix} \mathbf{k}_{11} & \mathbf{0} \\ \mathbf{0} & \mathbf{k}_{22} \end{bmatrix} \begin{bmatrix} \mathbf{v}_e \\ \mathbf{y} \end{bmatrix} = \begin{bmatrix} \mathbf{q}_1 \\ \mathbf{q}_2 \end{bmatrix} \quad (2.33)$$

where

$$\begin{aligned} \mathbf{m}_{11} &= \mathbf{M}_{ee} + \mathbf{B}^T \mathbf{M}_{ie} + \mathbf{M}_{ei} \mathbf{B} + \mathbf{B}^T \mathbf{M}_{ii} \mathbf{B} \\ \mathbf{m}_{12} &= \mathbf{m}_{21}^T = \mathbf{M}_{ei} \Phi + \mathbf{B}^T \mathbf{M}_{ii} \Phi \\ \mathbf{k}_{11} &= \mathbf{K}_{ee} + \mathbf{K}_{ie}^T \mathbf{B} \\ \mathbf{k}_{22} &= \begin{bmatrix} \omega_1^2 & & & \\ & \omega_2^2 & & \\ & & \ddots & \\ & & & \omega_{n-p}^2 \end{bmatrix} \\ \mathbf{q}_1 &= \mathbf{Q}_e + \mathbf{B}^T \mathbf{Q}_i \\ \mathbf{q}_2 &= \Phi^T \mathbf{Q}_i \end{aligned} \quad (2.34)$$

For the \mathbf{k}_{22} matrix, $\omega_1^2, \omega_2^2, \dots, \omega_{n-p}^2$ are the eigenvalues corresponding to the eigenmodes of eigenvector matrix Φ .

The substructure will be fully represented - meaning that no approximations are introduced - if all eigenmodes of the substructure is taken into account, thus setting $s = n - p$. Setting $s = 0$ will give the Guyan reduction presented in Section 2.2.1. When applying CMS reduction, one tries to keep s as small as possible for reducing the problem size. A criteria often used in practice is to include all eigenmodes having frequencies up to 1.8 or 2 times the highest frequency one wants to compute in the global structure. [2]

2.3 Using ANSYS for Solving Finite Element Problems

ANSYS is an American company founded in 1970 which develops and sells finite element simulation software. Their key product, the Mechanical APDL program, has the analysis capability to solve static and dynamic structural analyses, steady-state and transient heat transfer problems, mode-frequency and buckling eigenvalue problems, static or time-varying magnetic analyses, and various types of field and coupled-field applications.

2.3.1 ANSYS Parametric Design Language (APDL)

The Mechanical APDL program uses a command-line syntax called ANSYS Parametric Design Language (APDL) for writing commands to the FE solver. The language offers the possibility to use features like repeating commands, macros, if-then-else-branching, for-loops and matrix operations with APDL Math. In the last years, Mechanical APDL

has been replaced by, among others, the new products ANSYS Workbench and ANSYS Mechanical. However, all new ANSYS products still use the the FE solver implemented in Mechanical APDL. Learning to use the APDL language for writing FE solver commands is therefore of great importance. An example of APDL syntax can be seen in Section 2.3.2 and in the `onExportData()` method of `ModRed.py`, Section A.1. However, specifics of the APDL syntax will not be presented here. The Mechanical APDL Element Reference [3] serves as a great guide for referencing specific commands.

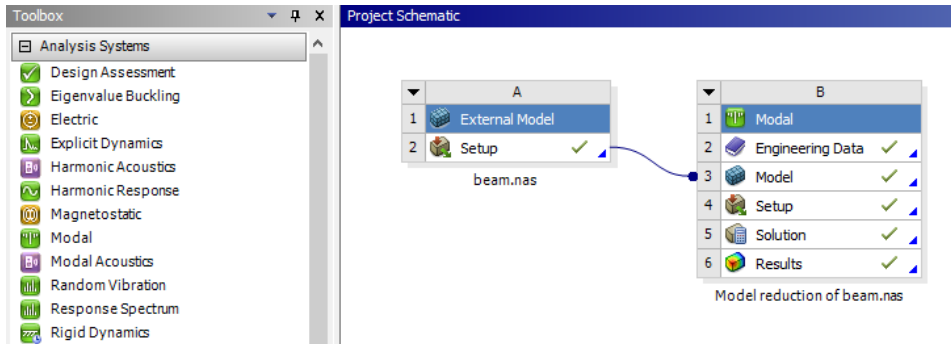


Figure 2.3: Example of a modal analysis set up in ANSYS Workbench that will be performed in the ANSYS Mechanical program.

Figure 2.3 shows the ANSYS Workbench environment that is used to analyse a simple beam constructed by shell elements. The ANSYS Workbench environment makes it easy to perform multi-domain analyses by dragging-and-dropping the analysis type seen to the left. When an analysis is opened, the user is taken to the ANSYS Mechanical program, where the analyses will be performed. When an analysis is correctly defined and the user hits "Solve", the following happens:

1. All items in the ANSYS Mechanical tree are converted to APDL commands, and written to the file `ds.dat` located in the ANSYS solver directory, illustrated in Figure 2.4.
2. Mechanical APDL is invoked in the background, performs the commands defined in the `ds.dat` file and returns results written in the file `file.rst`
3. The result file is read by ANSYS Mechanical, and displayed graphically to the user.

This means that it would be possible to write custom APDL commands directly to the `ds.dat` file for solving your specific problem. This is what is utilised in the `ModRed` application, Section 3.1

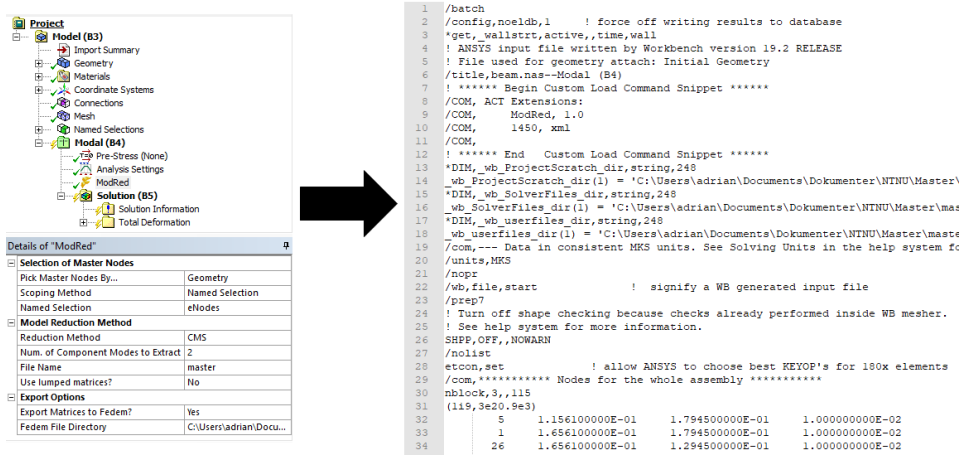


Figure 2.4: When the user hits "Solve" in ANSYS Mechanical, all analysis settings are translated to APDL code and sent to APDL in the background.

2.3.2 Performing Model Reduction in ANSYS

Model reduction is currently (Version 19.2) not available in the ANSYS Mechanical program. However, multiple model reduction techniques are available through APDL commands, among them Guyan (static) reduction and CMS reduction. Guyan reduction could be applied to both structural and non-structural analysis types. If Guyan reduction is applied to non-structural analyses, the \mathbf{K} and \mathbf{M} matrices are no longer expressing stiffness and mass quantities. Instead, they are matrices representing zero order terms (\mathbf{K}) and second order terms (\mathbf{M}). The CMS reduction implemented in APDL is not as versatile, and can only be applied to structural analyses. When performing model reduction in ANSYS, the process is divided into three separate steps, namely the generation pass, use pass and expansion pass.

The Generation Pass

The generation pass could further be divided into two parts: model generation and superelement generation.

The **model generation** part involves defining the element types, material properties, model geometry and specifying the jobname for reuse in the later use and expansion passes. If the model is already defined, as will be the case in the ModRed application, this part would involve importing the model and its properties correctly into the ANSYS environment.

The **superelement creation** part involves condensing the full model into one or multiple superelements. The superelement is defined by selecting external (master) nodes that will serve as the interface between other elements and the superelement. The master nodes

need to be defined even though the superelement is not to be connected to other non-superelements. Master nodes need to be selected with care, as forces or constraints will later be applied only to the master nodes. The master nodes are therefore typically selected to be at the joints or extremities of the full model. Figure 2.5 shows how one can model a bridge using super elements. It is modeled with bar elements taking axial deformation, as described in Section 2.1. By selecting green nodes as external nodes, 9 super elements can be created, one for every truss in the bridge. Their ID is marked with a square in Figure 2.5. The external nodes are selected at the joints, as this is where forces or constraints will be applied later. The external nodes' ID is marked with a circle.

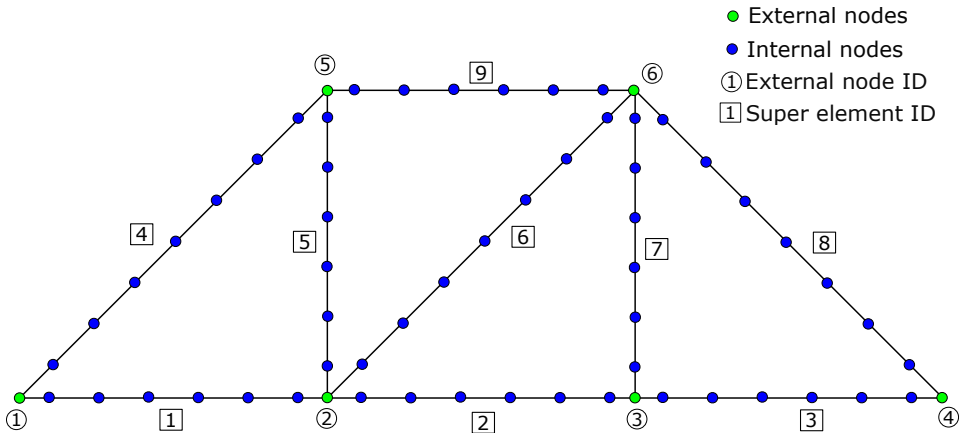


Figure 2.5: Structure simplified into finite elements

When the `solve` command is issued at the generation pass, multiple files are generated. Some are written in ASCII format, making it possible to read and edit them. However, the majority of files are written in the non-editable binary form for preserving the calculations and making them transportable between different computer systems. The most important generated binary files during a model reduction are

- .sub : The superelement matrix file, which contains the reduced matrices $\mathbf{m} = \mathbf{H}^T \mathbf{M} \mathbf{H}$, $\mathbf{k} = \mathbf{H}^T \mathbf{K} \mathbf{H}$ and load vectors if any load is applied during the generation pass.
- .emat : Element matrices file containing matrices for all elements in the model.
- .esav : Element saved data files
- .selo : Superelement load vector data from generation pass.
- .ln22 : Factorized stiffness matrix from the sparse solver
- .full : Assembled global stiffness and mass matrices for the full model.
- .db : Database file.

Among these, the `.sub` file is of particular interest, because it contains the mass and stiffness matrix for the reduced system. At a later stage, this file will be read and the

matrices converted and written out in ASCII format.

The following APDL code snippet shows a small example of a generation pass implemented in APDL code. It shows how one can define variables, as well as modelling a line, meshing it and selecting the element type to be used for the analysis.

Example of a generation pass in APDL

```
1 !
2 ! Example of a generation pass in APDL
3 !
4
5 / title , Cantilever Beam
6 /filename , gen          ! Changing filename for clarity
7 save                    ! Saving
8
9 ! Defining parameters
10 *set ,youngs,20.58e10 ! Young's modulus [N/m^2]
11 *set , density,7800   ! Density [kg/m^3]
12 *set , length , 0.6   ! Length [m]
13 *set , nElements, 10  ! Number of elements used
14 /prep7
15 k ,1,0,0              ! Enter Keypoints
16 k ,2, length,0
17 l ,1,2                ! Create Line between keypoints
18 et ,2,beam188         ! Element Type for non-super elements
19 mp,ex,1,youngs        ! Defining Young's Modulus
20 mp,prxy,1,0.33        ! Poisson's Ratio
21 mp,dens,1,density     ! Density
22 sectype ,1,beam, rect , , 0 ! Setting the beam's cross section
23 secoffset , cent      ! Setting section offset
24 secdata , 0.08, 0.005,0,0,0,0,0,0,0,0,0,0,0 ! Defining cross-section
25 lsize , all ,,, nElements ! Number of elements
26 lmesh , all , , nElements ! Mesh Line
27
28 /solu
29 antype, substr ! Defining a substructuring / CMS analysis type.
30 seopt ,gen ,2,0,0,0 ! Defining options for substructuring analysis . Generating K
    and M matrices
31 lumpm,0 ! Not using lumped mass matrix.
32 m , 1, all ! Selecting node 1 and 2 as master nodes. "Giving them access" to all
    degrees of freedom.
33 m , 2, all
34 save
35 solve ! Issuing the solve command
36 finish
37 /quit
```

The Use Pass

In the use pass, the superelement is used in analyses. Either by being a part of the model, connected to other non-superelements, or the entire model may be a superelement. Any analysis type except explicit dynamics analysis could be performed in the use pass, just like any other analysis. The difference being that your model now contains the superelement generated in the generation pass. The superelement portion of the model needs to be defined with the ANSYS element type MATRIX50 for the solver to know it is a super element. When the `solve` command is issued, the results will consist of a complete solution for the non-superelements, and a reduced solution containing the solution just at the selected master nodes, for the super elements. The reduced solution will be written to the file `use.dsub`.

The Expansion Pass

In the expansion pass, the reduced solution `use.dsub` is expanded, and results at all degrees of freedom in the superelement is calculated. Multiple expansion passes could be issued if multiple superelements were created in the use pass. The solver uses the files `gen.asav`, `gen.full`, `gen.sub`, `gen.ln22`, `gen.db` and `gen.seld` from the generation pass and `use.dsub` from the use pass. The type of expansion pass method is automatically detected by the solver.

2.3.3 Accuracy of Model Reduction Performed in ANSYS

The Mechanical APDL program has an extensive database of verification test cases used for quality assurance of its element types and solution algorithms. The test cases are based on validated results from published work, and are used for validating new versions of the program with extended functionality. A full overview of available test cases can be seen in [4].

Multiple test cases are available for testing the model reduction methods of Mechanical APDL. This section will present one of them, namely an automotive suspension system seen in Figure 2.6. The model is used for demonstrating the benefits of CMS reduction. Three super elements are created; left and right wheel and the main frame. The main frame is then constrained in the top bolts. Next, a modal analysis is performed, and the first 100 eigenfrequencies are extracted.

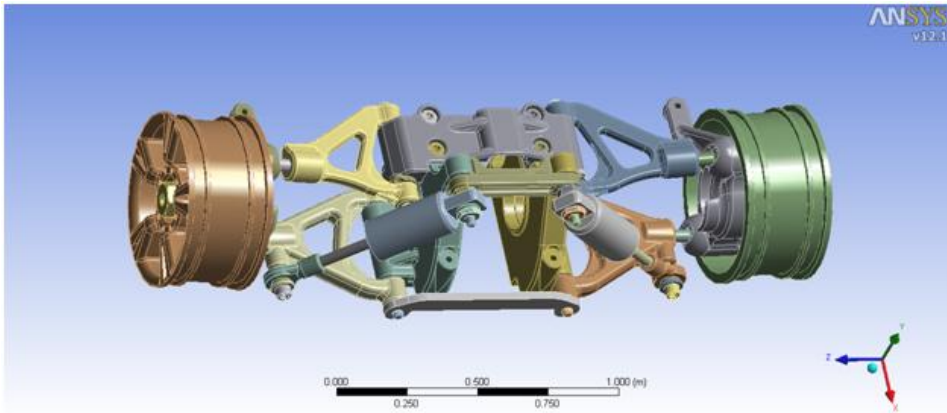


Figure 2.6: Automotive suspension system used for testing CMS reduction accuracy

Mode #	Full Model [Hz]	CMS Model [Hz]	% Diff
1	31.12	31.31	0.59
2	33.13	33.12	0.03
3	44.66	44.69	0.08
4	53.17	53.10	0.12
5	87.88	88.18	0.34

Table 2.1: Comparison of found eigenfrequencies for the automotive suspension assembly in the full model vs the CMS reduced model

An excerpt of the results is presented in Table 2.1, where the resulting 5 first eigenfrequencies for the corresponding modes are shown. It shows a very little loss of accuracy - a maximum of 0.59% - between analysing the full model and the model created by super elements. The full results could be seen in [5].

Full Model [s]	CMS Model	
	Generation Pass [s]	Use + Expansion Pass [s]
1291.0	1336.0	3.0

Table 2.2: Elapsed time for solving the full model and applying CMS reduction. Note that the generation pass is only necessary to do once when a geometry is set.

The improvement in solution time is also significant when using CMS reduction, as can be seen in Table 2.2, where the elapsed time for the modal analysis to calculate the first 100 frequencies of the model is shown. The results show that creating the super elements in the generation pass is a relatively expensive process, with an elapsed time close to solving the

full model. However, once the super elements are created, solving the modal analysis in the use and expansion passes is over 400 times faster than solving the full model. Applying CMS reduction to finite element models can therefore be a major advantage, as it enables a more agile development process, where multiple analyses could be performed rapidly, with a high accuracy for one design.

2.3.4 ANSYS ACT

As model reduction is currently not available through the ANSYS Mechanical application, one needs to create a custom way of accessing the model reduction techniques of APDL through the graphical user interface of ANSYS Mechanical. ANSYS Customization Toolkit (ACT) provides this possibility by enabling users to create apps for the ANSYS environment that could be tailored to fit their engineering problem. ACT can currently (Version 19.2) be used to customize the following ANSYS products:

- AIM
- DesignModeler
- DesignExplorer
- Electronics Desktop
- Fluent
- Mechanical
- SpaceClaim
- Workbench

2.3.5 Structure of an ACT Application

Two basic parts make up an ACT extension:

- An **XML file** defining context, custom GUIs and callbacks to functions.
- An **IronPython script** that contains functions responding to user interactions and GUI events, as well as the app's behavior. The IronPython language is an open-source implementation of the Python programming language. Its strength is its tight integration with the .NET Framework. Using IronPython, the user can use both the .NET Framework and Python libraries. In addition, other .NET languages can use the IronPython code. Further documentation can be seen in [6]

The extension may also contain components like external Python libraries, input files, and images to be displayed in the app. Once the extension is developed, it can be shared in two different formats, seen in Figure 2.7.

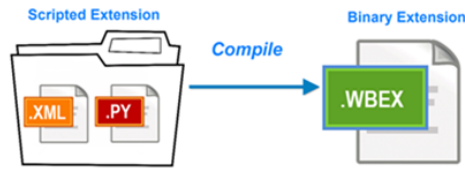


Figure 2.7: The two formats used for sharing ACT extensions: scripted or binary. ¹

The **scripted extension** contains the XML and IronPython script, as well as subdirectories containing additional code, images and image files. The contents of the scripted extension is all editable, allowing easy modification of functions in the extension.

The **binary extension** is a binary WBEX (WorkBench Extension) file that is generated when the scripted extension is built and compiled. The file could easily be shared and installed on another computer with an ANSYS license. The contents of the extension can not be edited, meaning that a new version of the WBEX file needs to be built, compiled and shared for the users to see changes made in the extension.

For the ANSYS Mechanical program, ANSYS ACT offers a wide range of possibilities for customisation:

- APDL macros can be run in the background, opening for automation of repetitive solver steps
- Having access to APDL scripting through ANSYS Mechanical enables the user to use solver capabilities of APDL that are not exposed in ANSYS Mechanical.
- Pre-processing features can be added, like custom loads and boundary conditions.
- Post-processing features could be developed in order to show custom results specifically tailored for the problem to solve.
- Third-party solvers are possible to integrate in the ANSYS environment for solver customisation.
- Graphics in ANSYS Mechanical is possible to tailor, opening possibilities to display custom information like drawing lines, surfaces and text descriptions.

2.3.6 ANSYS Element Library

The ANSYS element library contains all possible element types ANSYS offers for finite element analysis. Currently (Version 19.2) it contains 146 different elements, each specialized for its intended use. Elements have the following characteristics:

¹Image courtesy of [7].

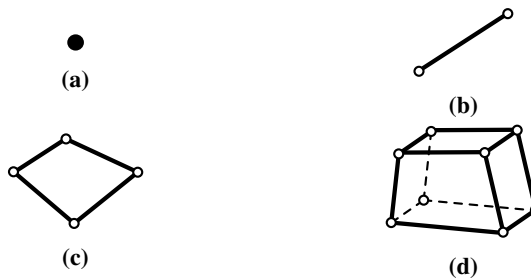


Figure 2.8: The four main element types: (a) Mass element, (b) Line element, (c) Area element, (d) Volume element. Element illustrations taken from [8]

- **Element name:** All element types have a unique name on a maximum of eight characters such as BEAM188. It consists of a group label (BEAM) as well as a unique number (188).
- **2D/3D:** The model is set in 2D or 3D depending on what elements to use. 2D models are defined in the X-Y plane and run faster than equivalent 3D models. The model becomes 3D if any 3D elements are used. A 2D element may be used in a 3D model.
- **Element Shape:** All elements are in general divided into four shapes, seen in Figure 2.8:
 - Point elements are defined by one node. Point elements are typically mass elements.
 - Line elements are represented by a line or arc that is connecting two or three nodes. Typical elements are beams, pipes and axisymmetric shells.
 - Area elements have triangular or quadrilateral shape. It might be a 2D solid element or a shell element.
 - Volume elements have tetrahedral or brick shape. Typical elements are 3D solid elements.
- **Discipline:** Specialized elements for the following disciplines are available:

– Structural	– Electric Circuit	– Load
– Thermal	– Coupled-Field	– Meshing
– Acoustic	– Contact	– Reinforcing
– Diffusion	– Combination	– User-defined
– Fluid	– Matrix	
– Magnetic Electric	– Infinite	

SHELL181

The SHELL181 element (Figure 2.9) is a 4-node (I, J, K, L) structural shell element for 3D space. It has six degrees of freedom at each node: Translation in x, y, z and rotation about the x, y, z axis. It is suitable for analyzing thin to moderately-thick shell structures. Its shell thickness needs to be defined with the APDL commands

Commands for Setting Thickness at SHELL181 Element

- 1 setcype, , shell
 - 2 secdata, thickness
-
-

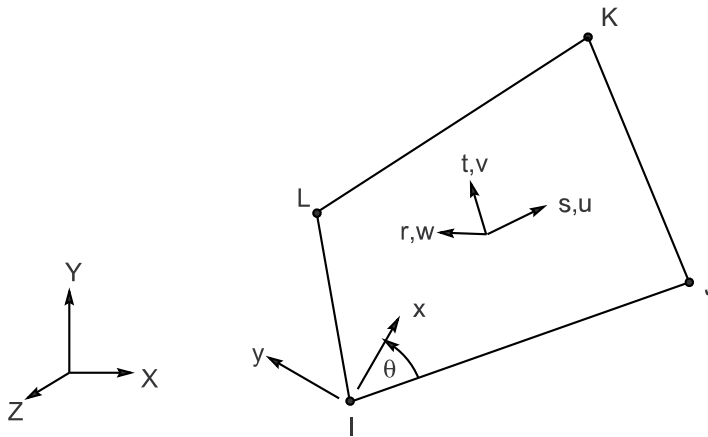


Figure 2.9: The SHELL181 element from the ANSYS element library [9]

For defining the shape functions of SHELL181, the following definitions and notations are used, referencing Figure 2.9: [8]

- As this is a shell element, the element coordinate system is not the same as the global Cartesian system. For SHELL181, u and v are in-plane motions and w is out-of-plane motion.
- Coordinates s, t and r are normalized, going from -1.0 on one side of the element, to $+1.0$ on the other side. They are not necessarily orthogonal to each other.
- Subscripted variables such as u_J refer to the u motion at node J .

For the stiffness and mass matrix, the following shape functions are used:

$$\begin{aligned}
 u = \frac{1}{4} & (u_I(1-s)(1-t) + u_J(1+s)(1-t) \\
 & + u_K(1+s)(1+t) + u_L(1-s)(1+t))
 \end{aligned}
 \tag{2.35}$$

$$v = \frac{1}{4}(v_I(1-s)(1-t) + \dots \text{(analogous to } u)) \quad (2.36)$$

$$w = \frac{1}{4}(w_I(1-s)(1-t) + \dots \text{(analogous to } u)) \quad (2.37)$$

$$\theta_x = \frac{1}{4}(\theta_{x,I}(1-s)(1-t) + \theta_{x,J}(1+s)(1-t) + \theta_{x,K}(1+s)(1+t) + \theta_{x,L}(1-s)(1+t)) \quad (2.38)$$

$$\theta_y = \frac{1}{4}(\theta_{y,I}(1-s)(1-t) + \dots \text{(analogous to } \theta_x)) \quad (2.39)$$

$$\theta_z = \frac{1}{4}(\theta_{z,I}(1-s)(1-t) + \dots \text{(analogous to } \theta_x)) \quad (2.40)$$

If a lumped mass approximation is selected, only (2.35) - (2.37) is used.

2.3.7 Degree of Freedom Ordering

ANSYS use an internal ordering method for the set of degrees of freedom (DOFs). When assembling the full mass and stiffness matrix \mathbf{M} , \mathbf{K} , ANSYS saves matrix data in a binary file located in the solver directory with the extension `.FULL`. Items saved in the `.FULL` file uses an **internal** solver ordering. The ordering is obtained by an internal ANSYS algorithm that reorders the equations in order to minimize the solver time and disk requirements. The DOF ordering is specified in the file `filename.mapping`, written out by setting the `Mapping` parameter to `Yes` in the APDL command `HBMAT`. An example of the `.mapping` file for the simple *twoQUAD4* model seen in Figure 4.1 is shown here.

Mapping file for the twoQUAD4 model

Matrix Eqn	Node	DOF
1	5	UX
2	5	UY
3	5	UZ
4	5	ROTX
5	5	ROTY
6	5	ROTZ
7	4	UX
8	4	UY
9	4	UZ
10	4	ROTX
11	4	ROTY
12	4	ROTZ
13	1	UX
14	1	UY
15	1	UZ
16	1	ROTX
17	1	ROTY

18	1	ROTZ
19	2	UX
20	2	UY
21	2	UZ
22	2	ROTX
23	2	ROTY
24	2	ROTZ
25	6	UX
26	6	UY
27	6	UZ
28	6	ROTX
29	6	ROTY
30	6	ROTZ
31	3	UX
32	3	UY
33	3	UZ
34	3	ROTX
35	3	ROTY
36	3	ROTZ

2.4 Fedem

Fedem, an acronym for Finite Element Dynamics in Elastic Mechanisms, is a computer program that provides features for creating, solving and post-processing a model in a 3D graphical environment. The program has multiple solver modules for performing different types of calculations. Detailed descriptions of each module can be seen in [10] (Fedem Version 7.2), and are briefly discussed here:

- **Reducer:** Performs a CMS reduction of the mass and stiffness matrices of a FE model for faster simulation of nonlinear dynamics.
- **Dynamics Solver:** Performs a non-linear dynamics simulation of the superelements' reaction over time to displacements and control system output.
- **Stress Recovery:** Recovers stresses and strain in the internal nodes from the deformations of the external nodes imposed at the model.
- **Mode Shape Recovery:** Recovers mode shapes from the eigenvalue results of the Dynamics solver.
- **Strain Rosette Analysis:** Applies virtual strain gauges on the FE model and outputs strain and stresses from the model over time similar to output from real strain gauges.
- **Strain Coat Analysis:** Recovers stresses and strains from the coat elements in a FE model, outputting the recovered stresses and strains for the model over the entire time history.

- **Curve Export Utility:** Allows the user to automatically export result curves to a single ASCII file.

2.4.1 Model Reduction in Fedem

The Fedem Reducer has implemented the Guyan reduction and CMS reduction techniques presented in Section 2.2 for effectively solving dynamic problems. The reduction technique is well suited for flexible mechanism analyses due to it preserving the effective masses and inertias of the model. The external nodes needed for the reduction are defined during modelling of the mechanism as "triads". The triads are defined on the connection points for joints, springs, dampers, external loads external load, control inputs or other points of interest. The reduction begins automatically when a dynamics simulation is started, and Fedem determines which parts that need to be reduced based on how the triads are chosen and their connection to the rest of the model. The number of component modes to use for the reduction is specified by the user before reduction. It is recommended to include the lowest modes of vibration in order to achieve good results.

2.4.2 Importing reduced models to Fedem

As Fedem does not support modelling or meshing of finite element models, they need to be created in external CAE systems, stored in separate files and imported into Fedem. Fedem supports importing files using the Nastran Bulk Data Format (.nas or .bdf), SESAM Input Interface File Format (.fem) as well as the older Fedem Link Model format (.flm). After reading these files, Fedem stores the info retrieved in the Fedem Technology Link Format (.ftl).

The .ftl file format contains all data needed for defining FE parts. It is defined in ASCII format, and can thus easily be edited using a text editor. The file contains a set of identifiers and parameters expressed with the same syntax:

```
identifier{id value1 value2 ... valueN {reference id text}}
```

where the parameters are listed in Table 2.3 taken from [10]

An example of an identifier with attributes is

```
QUAD4{4 22 34 12 32{PMAT 1}}
```

This defines a 4-noded tetrahedron element with ID=4 that is coupled to the nodes 22, 34, 12 and 32. The element uses an attribute of type PMAT with ID=1.

```
PMAT{1 2.10e+11 8.00e+10 2.90e-01 7.82e+03}
```

This defines the material property that is referred to in the QUAD4 element. The decimal numbers describe material parameters like Young's modulus, shear modulus, Poisson's ratio and density. A comprehensive guide of the available identifiers can be found in [10, p. 298]

Name	Description
<i>identifier*</i>	Specifies field type (e.g., element type, attribute type).
<i>id*</i>	Unique ID for the field (relative to the other fields with the same identifier).
<i>value1 ... valueN</i>	Primary values for the object (can be text, integers, or decimal digits).
<i>references</i>	Additional data or other fields can be referred to using this field.
<i>reference and id</i>	Field reference (reference specified in combination with a valid ID).
<i>text</i>	Can be used as additional information for a field reference or as an optional tag (e.g., a group name)

Table 2.3: Syntax of Technology Link Format

In order to perform calculations on imported models, Fedem does also need matrix files from the reduced models. Those are saved internally as binary `.fmx` files after model reduction is performed. Binary `.fmx` files are created for the reduced stiffness and mass matrices, as well as for the gravity vector described in Section 2.4.3

2.4.3 Gravity Vectors in Fedem

Gravitational forces in Fedem are calculated from unit gravitational acceleration vectors, as described in [11, p. 118]. Unit acceleration in the x , y and z direction is denoted \mathbf{U}_x , \mathbf{U}_y and \mathbf{U}_z respectively. The unit vectors are constructed so that for all degrees of freedom in \mathbf{U}_x that correspond to x translation, the acceleration component of \mathbf{U}_x is set to 1, otherwise 0. \mathbf{U}_x will be of size $((n_e + n_i) \cdot DOFs) \times 1$ where n_e is the number of external nodes, and n_i is the number of internal nodes. The total number of nodes in the model is $n = n_e + n_i$. For an element with 6 DOFs per node, \mathbf{U}_x , \mathbf{U}_y and \mathbf{U}_z will then be

$$\begin{aligned}\mathbf{U}_x &= [1 \ 0 \ 0 \ 0 \ 0 \ 1 \ 0 \ 0 \ 0 \ \dots \ 0]^T \\ \mathbf{U}_y &= [0 \ 1 \ 0 \ 0 \ 0 \ 0 \ 1 \ 0 \ 0 \ \dots \ 0]^T \\ \mathbf{U}_z &= [0 \ 0 \ 1 \ 0 \ 0 \ 0 \ 0 \ 1 \ 0 \ \dots \ 0]^T\end{aligned}$$

The gravitational forces \mathbf{G}_x corresponding to \mathbf{U}_x are calculated from

$$\begin{bmatrix} \mathbf{G}_{xe} \\ \mathbf{G}_{xi} \end{bmatrix} = \begin{bmatrix} \mathbf{M}_{ee} & \mathbf{M}_{ei} \\ \mathbf{M}_{ie} & \mathbf{M}_{ii} \end{bmatrix} \begin{bmatrix} \mathbf{U}_{xe} \\ \mathbf{U}_{xi} \end{bmatrix} = \begin{bmatrix} \mathbf{M}_{ee}\mathbf{U}_{xe} + \mathbf{M}_{ei}\mathbf{U}_{xi} \\ \mathbf{M}_{ie}\mathbf{U}_{xe} + \mathbf{M}_{ii}\mathbf{U}_{xi} \end{bmatrix} \quad (2.41)$$

The forces are then reduced by the CMS transformation matrix \mathbf{H} to \mathbf{g}_x :

$$\begin{bmatrix} \mathbf{g}_{xe} \\ \mathbf{g}_{xg} \end{bmatrix} = \begin{bmatrix} \mathbf{I} & \mathbf{B}^T \\ \mathbf{0} & \mathbf{\Phi} \end{bmatrix} \begin{bmatrix} \mathbf{G}_{xe} \\ \mathbf{G}_{xi} \end{bmatrix} = \begin{bmatrix} \mathbf{G}_{xe} + \mathbf{B}^T \mathbf{G}_{xi} \\ \mathbf{\Phi}^T \mathbf{G}_{xi} \end{bmatrix} \quad (2.42)$$

When inserting (2.41), and taking into account the symmetry property

$$\mathbf{M}_{ei} = \mathbf{M}_{ie}^T \quad (2.43)$$

the full expression for \mathbf{g}_{xe} and \mathbf{g}_{xg} becomes

$$\begin{aligned} \mathbf{g}_{xe} &= \mathbf{M}_{ie}^T \mathbf{U}_{xi} + \mathbf{M}_{ee} \mathbf{U}_{xe} + \mathbf{B}^T \mathbf{M}_{ii} \mathbf{U}_{xi} + \mathbf{B}^T \mathbf{M}_{ie} \mathbf{U}_{xe} \\ \mathbf{g}_{xg} &= \mathbf{\Phi}^T \mathbf{M}_{ii} \mathbf{U}_{xi} + \mathbf{\Phi}^T \mathbf{M}_{ie} \mathbf{U}_{xe} \end{aligned} \quad (2.44)$$

Finding gravitational forces in the y and z direction is an equivalent process by change of indexes:

$$\begin{aligned} \mathbf{g}_{ye} &= \mathbf{M}_{ie}^T \mathbf{U}_{yi} + \mathbf{M}_{ee} \mathbf{U}_{ye} + \mathbf{B}^T \mathbf{M}_{ii} \mathbf{U}_{yi} + \mathbf{B}^T \mathbf{M}_{ie} \mathbf{U}_{ye} \\ \mathbf{g}_{yg} &= \mathbf{\Phi}^T \mathbf{M}_{ii} \mathbf{U}_{yi} + \mathbf{\Phi}^T \mathbf{M}_{ie} \mathbf{U}_{ye} \end{aligned} \quad (2.45)$$

$$\begin{aligned} \mathbf{g}_{ze} &= \mathbf{M}_{ie}^T \mathbf{U}_{zi} + \mathbf{M}_{ee} \mathbf{U}_{ze} + \mathbf{B}^T \mathbf{M}_{ii} \mathbf{U}_{zi} + \mathbf{B}^T \mathbf{M}_{ie} \mathbf{U}_{ze} \\ \mathbf{g}_{zg} &= \mathbf{\Phi}^T \mathbf{M}_{ii} \mathbf{U}_{zi} + \mathbf{\Phi}^T \mathbf{M}_{ie} \mathbf{U}_{ze} \end{aligned} \quad (2.46)$$

The matrix used by Fedem for calculating gravitational forces is then the assembled matrix \mathbf{G} :

$$\mathbf{G} = \begin{bmatrix} \mathbf{g}_{xe} & \mathbf{g}_{ye} & \mathbf{g}_{ze} \\ \mathbf{g}_{xg} & \mathbf{g}_{yg} & \mathbf{g}_{zg} \end{bmatrix} \quad (2.47)$$

\mathbf{G} will be of size $((p \cdot DOFs) + s) \times 3$ where p is the number of external nodes, $DOFs$ is the number of degrees of freedom per node and s is the number of selected component modes.

Calculating the Gravity Vector from the Reduced Mass Matrix

The gravity vectors in Fedem could also be calculated from the reduced mass matrix $\mathbf{m} = \mathbf{H}^T \mathbf{M} \mathbf{H}$. The gravity vectors could then simply be calculated as

$$\begin{bmatrix} \mathbf{g}_{xe} \\ \mathbf{g}_{xg} \end{bmatrix} = \begin{bmatrix} \mathbf{m}_{11} & \mathbf{m}_{12} \\ \mathbf{m}_{21} & \mathbf{m}_{22} \end{bmatrix} \begin{bmatrix} \mathbf{U}_{xe} \\ \mathbf{U}_{xc} \end{bmatrix} = \begin{bmatrix} \mathbf{m}_{11} \mathbf{u}_{xi} + \mathbf{m}_{12} \mathbf{u}_{xe} \\ \mathbf{m}_{21} \mathbf{u}_{xi} + \mathbf{m}_{22} \mathbf{u}_{xe} \end{bmatrix} \quad (2.48)$$

The displacement vector \mathbf{U}_{xe} is the same as described in Section 2.4.3. \mathbf{U}_{xc} is of size $(s \times 1)$ and is set to 0. Expanding (2.48) gives for all directions:

$$\begin{aligned} \mathbf{g}_x &= \mathbf{H}^T \mathbf{M} \mathbf{H} \mathbf{U}_x^{\text{red}} \\ \mathbf{g}_y &= \mathbf{H}^T \mathbf{M} \mathbf{H} \mathbf{U}_y^{\text{red}} \\ \mathbf{g}_z &= \mathbf{H}^T \mathbf{M} \mathbf{H} \mathbf{U}_z^{\text{red}} \end{aligned} \quad (2.49)$$

Comparing this to what is given when using the full mass matrix, we see that gravity vectors from the full mass matrix are given by

$$\begin{aligned} \mathbf{g}_x^{\text{full}} &= \mathbf{H}^T \mathbf{M} \mathbf{U}_x^{\text{full}} \\ \mathbf{g}_y^{\text{full}} &= \mathbf{H}^T \mathbf{M} \mathbf{U}_y^{\text{full}} \\ \mathbf{g}_z^{\text{full}} &= \mathbf{H}^T \mathbf{M} \mathbf{U}_z^{\text{full}} \end{aligned} \quad (2.50)$$

where $\mathbf{U}_i^{\text{full}}$ are defined as in Section 2.4.3.

2.4.4 The FFQ4 Shell Element in Fedem

FFQ4, seen in Figure 2.10, is a 4-node quadrilateral shell element used for modelling shell structures in Fedem. It is composed of a Quadrilateral plate Bending Element with Shear deformation (QBESH) and a Quadrilateral Membrane element with Rotational degrees of Freedom (QMRF). The element nodes are numbered clockwise 1-2-3-4, referring to Figure 2.10. Each of the four nodes have six degrees of freedom: u, v, w, r_x, r_y, r_z where r_i refer to rotation about axis i .

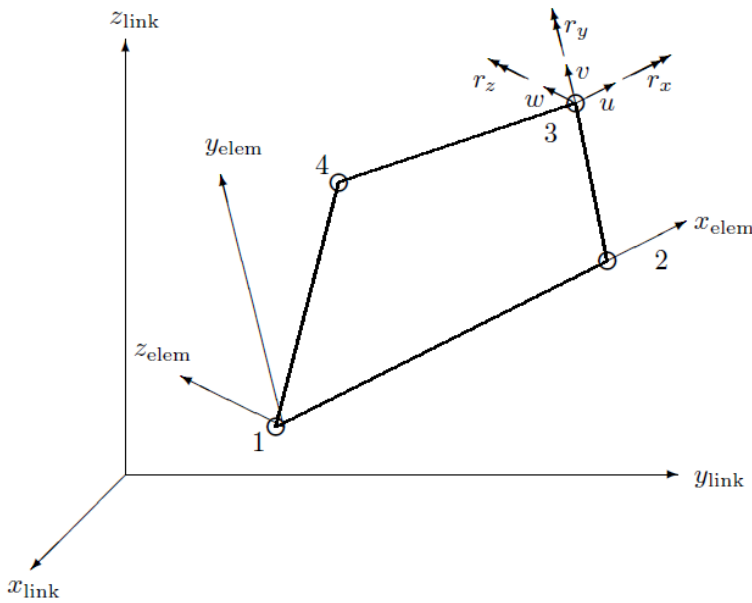


Figure 2.10: FFQ4 element used in Fedem ²

The FFQ4 shell element is unfortunately poorly documented, but is based on free formulation theory developed by Pål Bergan, Magne Nygård et.al. where an element is constructed

²Image courtesy of [12]

of two parts: one for describing rigid body modes, and one describing higher order terms [13, 14]. From Fedem version 7.3, the FFQ4 shell element is replaced with a new element: ANDES, proposed by Felippa, Militello in [15] and researched by Haugen, Skallerup in [16].

2.4.5 Degree of Freedom Ordering in Fedem

The mass and stiffness matrix in Fedem is, similarly to ANSYS, restructured by the Fedem solver, in order to reduce disk requirements and make the solving process more efficient. The mapping of equation number to corresponding degree of freedom is defined in the file `MEQN.res`, which is written to the Fedem solver directory by the additional solver option `-debug 3`. This mapping is used in the CMS reduction process, when the full mass matrix is partitioned into

$$\mathbf{M} = \begin{bmatrix} \mathbf{M}_{ee} & \mathbf{M}_{ei} \\ \mathbf{M}_{ie} & \mathbf{M}_{ii} \end{bmatrix}$$

The partitioning process is easier understood by a small example. We create a custom mass matrix where every element in the matrix is set to its corresponding index number (row-column):

$$\mathbf{M} = \begin{bmatrix} 11 & 12 & 13 & 14 & 15 \\ 21 & 22 & 23 & 24 & 25 \\ 31 & 32 & 33 & 34 & 35 \\ 41 & 42 & 43 & 44 & 45 \\ 51 & 52 & 53 & 54 & 55 \end{bmatrix}$$

The mapping of degree of freedom to the corresponding equation is given by the following Python-style dictionary:

```

1 mapping = {
2     1: 3,
3     2: 4,
4     3: 5,
5     4: 1,
6     5: 2
7 }
```

The dictionary maps DOF number to the corresponding equation: `mapping[dof] = equation`, e.g. `mapping[1] = 3, mapping[4] = 1`.

When the matrix is partitioned, the equations corresponding to the external nodes are moved to \mathbf{M}_{ee} , while the equations corresponding to the internal nodes are moved to the \mathbf{M}_{ii} part of the matrix. The \mathbf{M}_{ei} , \mathbf{M}_{ie} parts of the matrix will be a combination of the equations from the external and internal nodes. In addition, Fedem sorts the partitioned parts by **ascending equation number**.

The partitioning process will first move the rows corresponding to the equation numbers of the external and internal degrees of freedom. If the external degrees of freedom are chosen to be [2 5] and internal degrees of freedom are [1 3 4], the M matrix will be restructured to the following after moving the **rows**:

$$\begin{bmatrix} 21 & 22 & 23 & 24 & 25 \\ 41 & 42 & 43 & 44 & 45 \\ 11 & 12 & 13 & 14 & 15 \\ 31 & 32 & 33 & 34 & 35 \\ 51 & 52 & 53 & 54 & 55 \end{bmatrix}$$

Next, the **columns** are moved, resulting in the partitioned matrix

$$\begin{bmatrix} 22 & 24 & 21 & 23 & 25 \\ 42 & 44 & 41 & 43 & 45 \\ 12 & 14 & 11 & 13 & 15 \\ 32 & 34 & 31 & 33 & 35 \\ 52 & 54 & 51 & 53 & 55 \end{bmatrix}$$

The partitions of the matrix will then be

$$\begin{aligned} \mathbf{M}_{ee} &= \begin{bmatrix} 22 & 24 \\ 42 & 44 \end{bmatrix} & \mathbf{M}_{ei} &= \begin{bmatrix} 21 & 23 & 25 \\ 41 & 43 & 45 \end{bmatrix} \\ \mathbf{M}_{ie} &= \begin{bmatrix} 12 & 14 \\ 32 & 34 \\ 52 & 54 \end{bmatrix} & \mathbf{M}_{ii} &= \begin{bmatrix} 11 & 13 & 15 \\ 31 & 33 & 35 \\ 51 & 53 & 55 \end{bmatrix} \end{aligned}$$

2.5 Software Development

When developing software, many related processes lead to the production of a software system; specifications for the software's functionality are set, the software is developed and it is validated to ensure it meets the set specifications. Hence, the process of how one develops software should be investigated in order to optimize the process. Multiple models for software processes exist. For simplification, it is possible to divide the type of processes in two: plan-driven processes and agile processes. In **plan-driven processes** all process activities are planned in advance of the development. Progress made in the project as well as the resulting product is then measured against the plans set before development has started. In **agile processes**, the planning is continuous, and is a subject of change through the whole development process. Incremental development is one type of agile process.

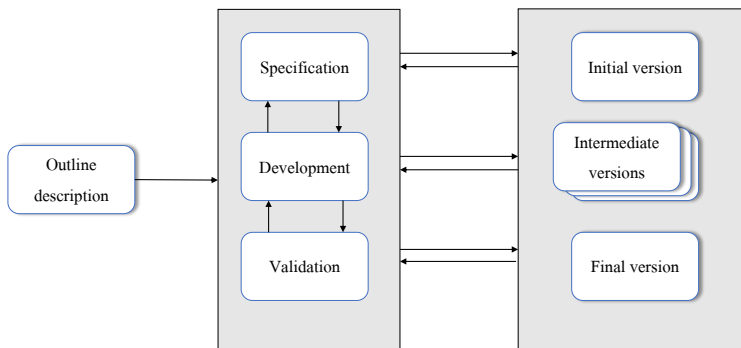


Figure 2.11: Incremental development process.

When software is developed incrementally, it is based on the idea that one develops an initial version of the software. This is then tested and feedback is generated. The software will then evolve in incremental versions until a system meeting the requirements set has been developed. The specification, development and validation phases of the project will always be subject for change through the whole development. This is illustrated in Figure 2.11. In this way it reflects the way humans solve problems: a complete solution to a problem is rarely known in advance, but we move closer to a solution in small steps, and backtrack if a mistake is discovered. Sommerville [17] states three major advantages to incremental development:

- There is little cost related to implementing changes in requirements, as the specification, development and validation processes are always continuous and open for change. This means the amount of analysis and documentation that has to be redone when requirements change is reduced.
- It is easier to get feedback on work in development, as one at an early stage of the development process tries to have a working, initial version of the product. For people outside the development process, it is easier to judge a product and perhaps edit its specifications if they can have it demonstrated.
- It is possible to gain value from a product at an early stage in the process, as some stand-alone features might be available for use.

2.5.1 Validating Software by Unit Testing

Unit testing is the process of testing the small units that make up a computer program. Individual functions are usually seen as the simplest type of component, and is therefore usually the target for unit testing. When performing unit testing, the aim is simply to test if your functions behave as expected. This is accomplished by calling these routines with a range of different input parameters, and monitoring if the expected result is returned or produced. Ideally, functions should be tested in isolation. This means that the test should

be free of dependencies to other parts of the program. In this way, we ensure that we are actually testing the desired unit of the program.

IronPython offers an automated unit testing framework `unittest` [18] for writing and running tests. It gives the possibility to run all tests written for a program, and reports the outcome of each individual test. The automated tests of `unittest` consist of three parts:

1. **Setup part:** Here, one can set up the initial state of the program. The setup is run before each single test, ensuring equal initial conditions for each test.
2. **Call part:** This is where the object or function to test is being called from the testing framework.
3. **Assertion part:** The behavior of the function is compared to what is expected. An evaluation of the assertion that evaluates to `True` means that the test has been successful, if it is `False`, it has failed.

The following example, taken from [18] with small modifications, shows a small, basic example of testing the behavior of Python's `random` module

Basic Unit Test Example

```
1 import random
2 import unittest
3
4 class TestSequenceFunctions( unittest .TestCase):
5
6     def setUp( self ):
7         self .seq = range(10)
8
9     def test_shuffle ( self ):
10        # make sure the shuffled sequence does not lose any elements
11        random. shuffle ( self .seq)
12        self .seq. sort ()
13        self . assertEquals ( self .seq, range(10))
14
15    def test_choice ( self ):
16        element = random.choice( self .seq)
17        self . assertTrue (element in self .seq)
18
19 if __name__ == '__main__':
20    unittest .main()
```

Chapter 3

Method

ANSYS supports the same CMS model reduction as implemented in Fedem. This means that ANSYS is able to generate the same reduced mass, stiffness and transformation matrix \mathbf{H} as needed by Fedem to perform dynamics analyses. And this again means that the reduced models from ANSYS could be used in the Fedem environment, allowing the following benefits:

- Creating models could be performed in ANSYS with the CAD programs ANSYS SpaceClaim or ANSYS DesignModeler. The CAD programs support parametrization of the models, which in return facilitates the analysis of design changes.
- The ANSYS Meshing program could be used to create finite element meshes. The program supports advanced mesh configuration tools like automated inflation, physics-aware meshing and controls for moving, merging and editing nodes and elements.
- When meshing is performed in ANSYS, the full finite element library [9] of ANSYS is available. Currently (Version 19.2) it contains 146 different elements. Each element type is thoroughly documented, and specific elements for different applications are available.
- The extensive material data and material designer capabilities of ANSYS could be integrated in the models. This makes it possible to model composite materials, anisotropic material and more in the analysis.
- Using a combination of ANSYS and Fedem for analyses would make it possible to verify results to a larger extent than what is currently possible. Identical analyses could be performed using both systems, thus extending the possibilities for verifying results.

This chapter will introduce the ANSYS ACT extension "ModRed" that will enable an easy-to-use integration of CMS model reduction performed in ANSYS into Fedem. Its implementation details will be explained, and different use cases will be presented.

3.1 The ModRed Extension for Performing CMS Model Reduction in ANSYS

In order to open and analyze models in Fedem, the following matrices and data is needed:

- The reduced mass matrix by CMS reduction: $\mathbf{m} = \mathbf{H}^T \mathbf{M} \mathbf{H}$
- The reduced stiffness matrix by CMS reduction: $\mathbf{k} = \mathbf{H}^T \mathbf{K} \mathbf{H}$
- A FE mesh in `.nas`, `.fem` or `.ftl` format
- The assembled gravity vectors \mathbf{G}

The ACT extension "ModRed" is able to create all of these matrices and data, enabling easier integration between ANSYS and Fedem.

3.1.1 Installation

The ModRed extension will be available both as a compiled WBEX file, as well as a scripted extension containing the full source code. Installing the extension is an easy procedure, but differs slightly depending on what type is to be installed:

Compiled WBEX extension

In ANSYS Workbench, go to

Extensions → Install Extension

and select the `.wbex` file in the file dialog. This will install the extension to the ANSYS environment. If the extension does not show, try checking the "Loaded" box for ModRed under *Extensions → Manage Extensions*

Scripted extension

In ANSYS Workbench, open the ACT Start Page tab. From there, go to *Manage Extensions*, click the *Settings* icon and select *Add Folder*. Then select the folder containing the file `ModRed.xml`. This will install the extension into the environment, and also allows editing the source code while using the extension. When opening the ANSYS Mechanical program, the ModRed extension is added to the toolbar, as seen in Figure 3.1

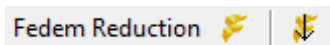


Figure 3.1: The ModRed extension loaded into the toolbar. The left button places ModRed in the ANSYS Mechanical solution tree. The right button exports Fedem data.

Figure 3.1 shows the ModRed app loaded into the ANSYS Mechanical model tree, as a new "Solution" type. This allows the extension to edit solver commands as well as performing post-processing manipulation of solver data.

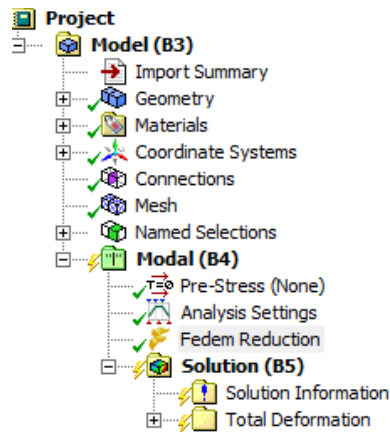


Figure 3.2: The ModRed extension loaded into ANSYS Mechanical

3.1.2 Importing a model into ANSYS Mechanical

There are multiple ways to import models into the ANSYS Mechanical program. Common for all is that this process is handled by ANSYS Workbench.

Importing an Existing Mesh

If the model and mesh is already defined, ANSYS may import the mesh file through the *External Model* module, Figure 3.3

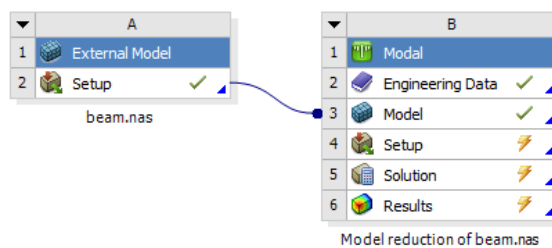


Figure 3.3: Importing an existing mesh into ANSYS Mechanical

From the *External Model* module, the file formats possible to import are listed in Table 3.1

APDL Common Database	Abaqus Input	NASTRAN Bulk Data	Fluent Input	ICEM CFD	LS-DYNA
.cdb	.inp	.bdf, .dat, .nas	.msh, .cas	.uns	.k

Table 3.1: Supported mesh file formats for import to ANSYS Mechanical

When using the *External Model* module to import a meshed model, ANSYS will convert the element type used in the external system to the most similar ANSYS element type in order to be able to solve the system.

Modelling From Scratch in ANSYS

If the modelling is to be performed in ANSYS, a *Geometry* component is used to import the model from the modelling program and into ANSYS Mechanical. Figure 3.4 shows a model made in DesignModeler being imported into ANSYS Mechanical. This method is the most versatile, as the geometry made in ANSYS is possible to parametrize, making it possible to perform analyses on multiple geometrical variations.

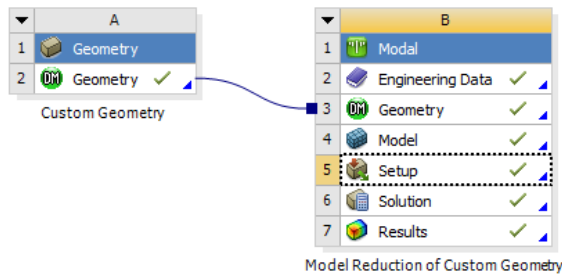


Figure 3.4: Import a Native Model

3.1.3 Defining Settings for the ModRed Extension

Details of "Fedem Reduction" ⌵	
[-] Selection of Master Nodes	
Pick Master Nodes By...	Geometry
Scoping Method	Named Selection
Named Selection	eNodes
[-] Model Reduction Method	
Reduction Method	CMS
Num. of Component Modes to Extract	2
File Name	master
Use lumped matrices?	No
[-] Export Options	
Export Matrices to Fedem?	Yes
Fedem File Directory	C:\Users\adrian\Documents\Dok...

Figure 3.5: The ModRed extension loaded into ANSYS Mechanical

Selecting external nodes

As ANSYS is not going to perform any analyses, only calculate the reduced stiffness and mass matrices, one only needs to perform the generation pass of CMS reduction in ANSYS. Here, the user selects what nodes to be used as external or "triads" in Fedem. ANSYS will then mark these as master nodes when the APDL solver is invoked. The external nodes may be selected in two ways:

Geometry Selection: Using the integrated ANSYS tool for selecting geometry, the user may simply select the nodes that will be set as triads in Fedem. Using the node selection tool, Figure 3.6, changes the view to render the mesh, and allows only nodal points to be selected. Press *Ctrl* while selecting nodes to select multiple nodes.



Figure 3.6: Node selection tool

Named Selection: ANSYS also allows selecting the desired nodes and saving them to a *Named Selection* which may be accessed from the ModRed settings. In this way, the user can select and save multiple sets of external nodes in order to decide what will yield the best results.

Setting Model Reduction Settings

The ModRed extension allows the user to select two types of model reduction: Guyan and CMS reduction. If the *Guyan* reduction option is selected, no additional component modes

are selected, and the transformation matrix is $\mathbf{H} = \mathbf{H}_{\text{Guyan}}$ from Section 2.2.1. If the *CMS* reduction method is chosen, the user also gets the possibility of setting the number of component modes to be extracted from the model. The transformation matrix will then be set as $\mathbf{H} = \mathbf{H}_{\text{CMS}}$, from Section 2.2.2.

The *File name* tab allows the user set a file name that will be given to the `.ftl` file and binary `.fmx` files to be generated. This is useful if multiple parts is to be reduced. A unique name for each part will then make the process of importing the data into Fedem easier.

The user can also select whether or not to use lumped matrices. Setting the option to *Yes* will force the APDL solver to use the `LUMPM` command that makes the solver use a lumped mass approximation for reducing the disk space needed to save the matrices.

Setting Export Options

In the *Export Options*, the user can select whether the matrices are to be exported or not. This is beneficial if the user wants to force ANSYS to not export new matrices on every *Solve* command. In the *Fedem File Directory* tab, the user selects to which folder to output the generated `.ftl` and `.fmx` files.

3.1.4 The APDL Commands Written by ModRed

Once all necessary settings to perform the model reduction has been set, the model is ready to be solved. This is done by hitting the *Solve* button. This will then invoke the `onSolve()` method in `ModRed.py`, which adds APDL commands to the `ds.dat` file to be read by the solver, as explained in Section 2.3.1. The code below shows an excerpt from the `ds.dat` file that is generated when solving a model with nodes 1, 2, 3, 4 selected as the external nodes. See the comments marked with `!` for more information about each command. Extensive documentation for each command is found in [19].

Excerpt from ds.dat

```
1 ! ***** Begin Command Snippet *****
2 !
3 ! Commands written by ModRed
4 !
5 finish
6 /filename, master      !Setting the file name
7 save
8 /solu
9 antype, substr !Defining a substructure analysis (includes both Guyan and CMS)
10 outpr, nsol, all      ! Print command that must be defined if iokey=="tcms"
11 cmsopt, fix ,2,,,,, tcms !Fixed interface normal modes with 2 component modes
12 seopt, master, 2, 0, 0, ,      ! Super element name and generating mass+ stiffness matrix
13 m, 1, all            ! Setting node 1-4 as master nodes, "having access to" all degrees of
    freedom
```

```

14 m, 2, all
15 m, 3, all
16 m, 4, all
17 all          ! Selecting all nodes before solving
18 solve
19 save
20 finish
21 ! Master node selection finished
22 ! Exporting matrices ...
23 *dmat, cst, d, import, tcms, master.tcms, cst ! Importing the CST matrix
24 *dmat, nor, d, import, tcms, master.tcms, nor ! Importing the NOR matrix
25 save
26 *export, cst, mmf, CST.mmf ! Exporting CST and NOR matrices to MMF format
27 *export, nor, mmf, NOR.mmf
28 /aux2
29 fileaux2, master, sub ! Specifying to dump the reduced matrices
30 hbmat, M_red, hbmat, , ascii, mass, no ! Dumping reduced mass and stiffness matrix to
    Harwell-Boeing format (16-decimal precision)
31 hbmat, K_red, hbmat, , ascii, stiff, no
32 fileaux2, master, full ! Specifying to dump full matrices
33 hbmat, M_full, hbmat, , ascii, mass, no, yes ! Dumping full mass matrix in
    Harwell-Boeing format (16-decimal precision)
34 finish
35 *smat, M_red, d, import, hbmat, M_red.hbmat, ascii ! Importing Harwell-Boeing format
    matrices
36 *smat, M_full, d, import, hbmat, M_full.hbmat, ascii
37 *smat, K_red, d, import, hbmat, K_red.hbmat, ascii
38 *export, M_red, mmf, M_red.mmf ! Exporting matrices in MMF format (maintains
    16-decimals precision)
39 *export, M_full, mmf, M_full.mmf
40 *export, K_red, mmf, K_red.mmf
41 save

```

3.1.5 Matrix Export

When the model has been solved with the added APDL commands seen in Section 3.1.4, matrices are then exported to the ANSYS solver directory. This can be accessed by right-clicking the *Solution* in the ANSYS Mechanical tree and then selecting *Open Solver Files Directory*. The following matrices are written to this directory:

- `M_full.mmf`: This is the full mass matrix, saved in a sparse format, and formatted in the MatrixMarket file format. Note also the following:
 - The matrix is saved column-first.
 - In order to utilize symmetry and reducing storage space needed, only the lower triangular part of the matrix is saved in the `.mmf` file.

- `M_full.mapping`: Mapping file for the full mass matrix. This relates node to the corresponding equation in the matrix.
- `M_red.mmf`, `K_red.mmf`: The reduced mass and stiffness matrix, saved in sparse format.
- `CST.mmf`: The static part of the transformation matrix \mathbf{H} , namely

$$\text{CST} = \begin{bmatrix} \mathbf{I} \\ \mathbf{B} \end{bmatrix} \quad (3.1)$$

CST is saved in dense form, column-first

- `NOR.mmf`: The component modes part of the transformation matrix \mathbf{H} , namely

$$\text{NOR} = \begin{bmatrix} \mathbf{0} \\ \mathbf{\Phi} \end{bmatrix}$$

NOR is saved in dense form, column-first

3.1.6 Generation of files to be read by Fedem

When the model has been solved, the user can click the *Export Fedem Data* button in the ModRed toolbar, thus invoking the `onExportData()` method in `modRed.py`. The full mass matrix will then be partitioned into external and internal parts as described in Section 2.4.5.

For visualising the solved model in Fedem, a `.ftl` file of the model is created and saved in the specified Fedem directory. Binary `.fmx` files containing the reduced mass and stiffness matrix and gravity vectors are also generated.

3.2 Verifying Data From the ModRed Extension

In order to verify that all matrix handling is performed correctly, an IronPython Unit Test framework has been set up. Here, all methods that are involved in matrix calculations are tested for common errors for ensuring correct code and matrix calculations. An excerpt of what is tested is shown in the following section.

Verification of Matrices

The exported matrices from ANSYS in MatrixMarket Format need to be read by the ModRed extension in order to generate the needed Fedem files. As the used math library `Math.NET.Numerics.LinearAlgebra` [20] does not have any built-in functions for reading MatrixMarket formatted files, this method has to be written customly for the ModRed app. Thus, it is important that the matrices are read correctly before further calculations.

Testing that the matrix is read properly could be done through unit testing. The following code snippet shows how it is verified that the matrix is correctly read into IronPython by verifying the number of rows and columns on a test file, as well as checking some entries custom-picked from the ASCII-formatted `.mmf` file.

```
1 def test_readMMFMatrix_full( self ):
2     path = self . _resourcesFolder_ + "\\beam.nas" + "\\Ansys" + "\\M_full.mmf"
3     mat = ModRed.readMMFMatrix(path, "full")
4
5     # Checking if dimensions are correct
6     self . assertEquals (mat.RowCount, 630)
7     self . assertEquals (mat.ColumnCount, 630)
8
9     # Checking some values at the boundaries :
10    self . assertEquals (mat [0,0], 8.688888888888170E-02)
11    self . assertEquals (mat[0, 629], 0)
12    self . assertEquals (mat[629, 0], 0)
13    self . assertEquals (mat[629, 629], 1.448148148148030E-16)
14
15    # Testing random values :
16    self . assertEquals (mat [23,23], 1.448148148148030E-16)
17    self . assertEquals (mat [285,243], 3.620370370370130E-07)
18    self . assertEquals (mat [291,291], 5.792592592592110E-06)
```

Testing for Symmetry

The generated mass and stiffness matrix should be symmetrical, both for the full and for the reduced versions. This is verified by the following simple test, which is also performed on the reduced mass matrix:

```
1 def test_readMMFMatrix_full_isSymmetric( self ):
2     path = self . _resourcesFolder_ + "\\beam.nas" + "\\Ansys" + "\\M_full.mmf"
3     mat = ModRed.readMMFMatrix(path, "full")
4
5     # Checking if matrix is symmetric:
6     for row in range(0, mat.RowCount):
7         for col in range(0, mat.ColumnCount):
8             self . assertEquals (mat[row, col ], mat[col , row])
```

Testing for Consistent Mass

In order to verify that the mass matrix has a consistent mass after reduction, one can compute the total mass of the system by pre- and post-multiplying the system with unit translation in x , y and z direction:

$$m_{\text{total}} = \mathbf{u}_{\text{trans}}^T \mathbf{m} \mathbf{u}_{\text{trans}} \quad (3.2)$$

This is then compared with the mass of the model computed by Fedem. The same test is done with the full mass matrices by the following unit test:

```
1  def test_readMMFMatrix_red.correctMass( self ) :
2      """
3      Applying unit translation in x, y, z direction
4      and verifying that it is the same mass as computed in Fedem
5      """
6      path = self._resourcesFolder_ + "\\beam.nas" + "\\Ansys" + "\\M_red.mmf"
7      mat = ModRed.readMMFMatrix(path, "sub")
8      correct_mass = 3.12800E+01
9
10     for i in [1, 2, 3]:
11         u = ModRed.createUnitVector(i, 4, 2)
12         m = u.Transpose().Multiply(mat).Multiply(u)[0,0]
13         self.assertAlmostEqual(correct_mass, m)
```

Note that the `assertAlmostEqual` method is used, as there will be a small difference in results because of floating point precision. The `assertAlmostEqual` method tests for equality down to 7 decimal places by default.

Testing if matrix is diagonal

As both mass and stiffness matrices will be diagonal, one could check if elements not on the diagonal are zero. This is best tested after the matrix has been partitioned, as the partitioned matrix should remain diagonal.

```
1  def test_massMatrix_diagonality ( self ) :
2      #
3      # Verifying that a lumped matrix is diagonal
4      #
5      mat = ModRed.readMMFMatrix(self._resourcesFolder_ + "\\beam.nas" + "\\Ansys" +
6      "\\M_full_lumped.mmf", "full")
7      for row in range(0, mat.RowCount):
8          for col in range(0, mat.ColumnCount):
9              if row == col:
```

```

9         self . assertTrue ( mat[row, col] != 0.0 )
10     else :
11         self . assertTrue ( mat[row, col] == 0.0)

```

Testing if Matrices are Partitioned Correctly

Testing if the matrices are partitioned correctly may only be partially done. This is due to the fact that Fedem, which is used as a reference, does only export the full, unpartitioned mass matrix \mathbf{M} , the \mathbf{M}_{ee} part of the partitioned matrix, and the reduced mass matrix \mathbf{m} . ANSYS does only export the full mass matrix \mathbf{M} and the reduced mass matrix \mathbf{m} . This implies that it could only be verified that the \mathbf{M}_{ee} part of the matrix is partitioned correctly. It should be noted that this only tells us that \mathbf{M}_{ee} is partitioned in an identical way as done by the Fedem reducer, not that it is identical to what ANSYS does, because ANSYS does not export the partitioned matrix. An example of testing the partitionMatrix function is shown in the following:

Test of partitionMatrix function

```

1  def test_partitionMatrix_beam_Mee ( self ):
2      # Testing with M_full from beam.nas (medium model)
3      mapping = ModRed.readMappingFileFedem(self._resourcesFolder + "\\beam.nas" +
4      "\\Fedem" + "\\MEQN.res")
5      M_full = ModRed.readFedemMatrix(self._resourcesFolder + "\\beam.nas" +
6      "\\Fedem" + "\\M_full.res")
7      eNodes = [1, 2, 3, 4]
8      iNodes = [i for i in range(5, 106)]
9      M_full_partitioned = ModRed.partitionMatrix(M_full, eNodes, iNodes, mapping, 6)
10     # Using M_ee from Fedem as reference
11     M_ee = ModRed.readFedemVector(self._resourcesFolder + "\\beam.nas" +
12     "\\Fedem" + "\\M_ee.res")
13
14     for row in range(0, M_ee.RowCount):
15         for col in range(0, M_ee.ColumnCount):
16             self . assertAlmostEqual( M_full_partitioned [row, col], M_ee[row, col] )

```

Chapter 4

Results

Testing has been performed with the three FE models listed in Table 4.1. All models are NASTRAN `.nas` files created in an external CAD system. For acquiring matrices from ANSYS, the model is imported through the *External Module* component and opened in ANSYS Mechanical. The ModRed extension is then used for selecting external nodes and exporting the needed matrices. To acquire matrices from Fedem, the same external nodes as defined in ANSYS have been selected, and the `fedem_reducer` module is run. The matrices are then read from the result file `fedem_reducer.res` for analyses.

	Small	Medium	Large
Fedem element	QUAD4	QUAD4	QUAD4
ANSYS element	SHELL181	SHELL181	SHELL181
Number of nodes	6	105	7869
Number of elements	2	80	7680
Material	Steel	Steel	Aluminium
Thickness [m]	0.02	0.02	0.01
Number of selected external nodes	2	4	4
File name	twoQUAD4	beam	plate

Table 4.1: FE models used for testing the ModRed extension

The small model can be seen in Figure 4.1. It is very simple, containing two 4-node shell elements, with node 1 and 3 chosen to be external. The model is used because it facilitates easier debugging of its corresponding matrices and degree of freedom ordering.

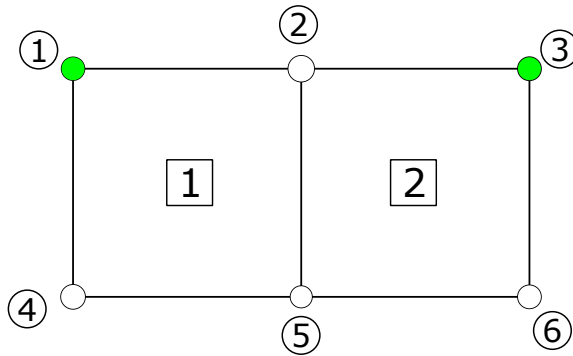


Figure 4.1: The small test model put together of two 4-node shell elements. The external nodes are marked green.

4.1 Similarity of Mass Matrices between ANSYS and Fedem

When analyses of dynamics is performed in Fedem, it is vital that the structural matrices for the models yield identical results, independent of what the origin of matrices are. When the ModRed app is to be used for enabling analyses in Fedem, FE models and their associated structural matrices are likely to be of two different origins:

1. The FE model will be made in an external CAD software and exported to a file format that could be imported by Fedem. The external nodes (triads) will then be selected in Fedem, and the Fedem reducer will be used to generate the reduced mass matrix \mathbf{m} used in calculations.
2. The FE model is created or imported to ANSYS. The external nodes will then be selected in ANSYS Mechanical, and the internal ANSYS APDL solver will be used for calculating the reduced mass and stiffness matrix. These matrices will then be saved in a binary `.fm \times` format and read by Fedem. The Fedem reducer will not be used.

In order to test the similarity of structural matrices in ANSYS and Fedem, structural matrices for the models seen in Table 4.1 have been generated in both ANSYS and Fedem and then compared. In order to quantify their internal difference, the Euclidean distance d between the sum of the two matrices is calculated. Then the relative difference α of the distance compared to the reference matrix is calculated:

$$d = \sqrt{\sum_{i=1}^n \sum_{j=1}^n (\mathbf{a}_{ij} - \mathbf{b}_{ij})^2}$$

$$\alpha = \frac{d}{\sum_{i=1}^n \sum_{j=1}^n \mathbf{b}_{ij}}$$

Here, \mathbf{a}_{ij} is the elements of the matrix that will be compared, and \mathbf{b}_{ij} is the matrix used as reference. n is the total number of rows in the matrices. When using this method for comparing matrices, $\alpha = 0$ means the two matrices are identical. $\alpha = 1$ would imply there is a 100% difference between the matrices. I.e. a matrix filled with 2's compared to a matrix filled with 1's used as a reference, would give $\alpha = 1$. In the analysis, Fedem mass matrices are used as a reference in all models.

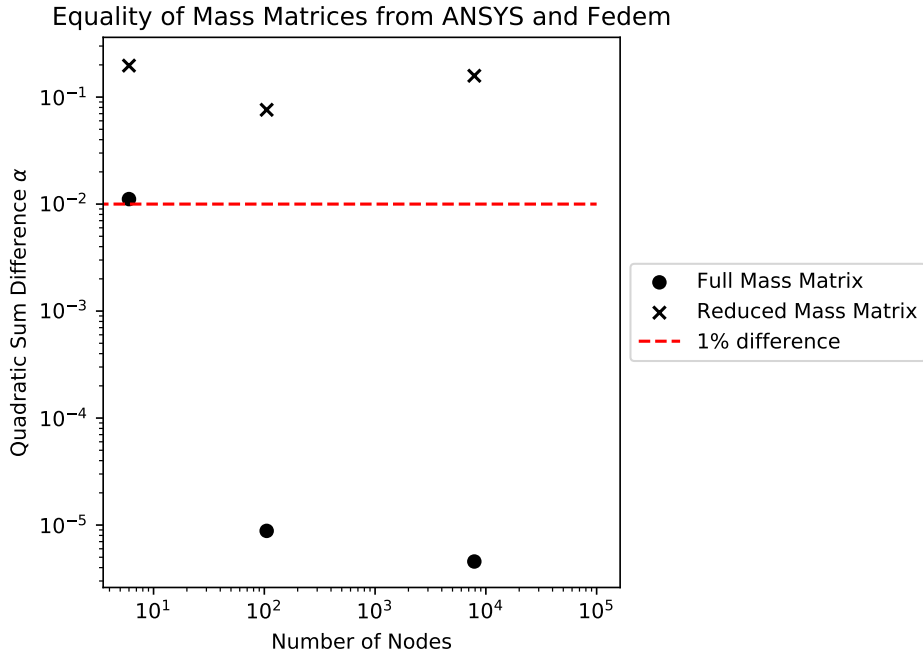


Figure 4.2: Percentage difference of full and reduced mass matrices from Fedem and ANSYS in relation to the number of nodes in the model

Figure 4.2 shows the percentage difference α of the full and reduced mass matrices of the test models. The full mass matrices yield good results for models with > 100 nodes, with $\alpha < 10^{-5}$. For the small model with only six nodes, about 1% difference is obtained. The reduced mass matrices however, show overall relatively poor results, with α values indicating 10 – 15% difference between the reduced ANSYS and Fedem matrices. This means that if one were to set an acceptance limit of 1% difference, indicated in the figure, none of the reduced matrices would pass. Despite their poor α results, the reduced matrices are used in the following results, where they are, surprisingly, proven to yield some very good results. The results are further discussed in Section 5.1.

4.2 Testing for Consistent Mass in Models

As the full mass matrix is reduced in order to be used by Fedem, a key test for verifying that the reduced mass matrices are reduced correctly is to calculate the total mass of the model from the reduced mass matrix, and compare it to the total mass calculated from the full model. The model's total mass could be found by pre- and post-multiplying the mass matrix with unit translation in the x , y and z direction, similar to what is done in Section 2.4.3:

$$m_{\text{total}} = \mathbf{U}_{x,\text{red}}^T \mathbf{m} \mathbf{U}_{x,\text{red}} \quad (4.1)$$

The calculated total mass from reduced mass matrices in ANSYS is then compared to the calculated total mass from the full mass matrix from Fedem for all models. The full Fedem mass matrix is used as a reference in all calculations. The reference weight is found as an average of resulting weight of the full Fedem model after x , y and z translation. Mass calculated from the reduced mass matrices is also found as an average of x , y and z translation.

	Ref. [kg]	Diff. [%]
Small	312.8	-2.24×10^{-12}
Medium	31.28	-1.14×10^{-10}
Large	5.184	8.22×10^{-10}

Table 4.2: Mass of test models calculated from the reduced mass matrix compared to mass calculated from the full mass matrix

Table 4.2 shows resulting percentage difference of mass calculated from the reduced mass matrix compared to mass calculated from the full mass matrix. For all models, there is close to no difference between the calculated masses. This indicates that the reduction process has been successful.

4.3 Calculation of Gravity Vectors

Gravity vectors have been calculated for all three test models and compared to the gravity vector given by Fedem, which is calculated from the full mass matrix. For quantifying the vectors' similarity to the Fedem gravity vector, the relative difference α based on the Euclidean distance (4.1) is used. However, because we are now comparing two vectors, namely \mathbf{g}_x from Fedem and ANSYS, the Euclidean distance d is found by summing all entries in the 1-dimensional vector. The same is done in y and z direction. The gravity vector from Fedem is used as reference for all models.

Both the full and reduced mass matrix from ANSYS have been used for calculating gravity vectors, following the equation set described in Section 2.4.3.

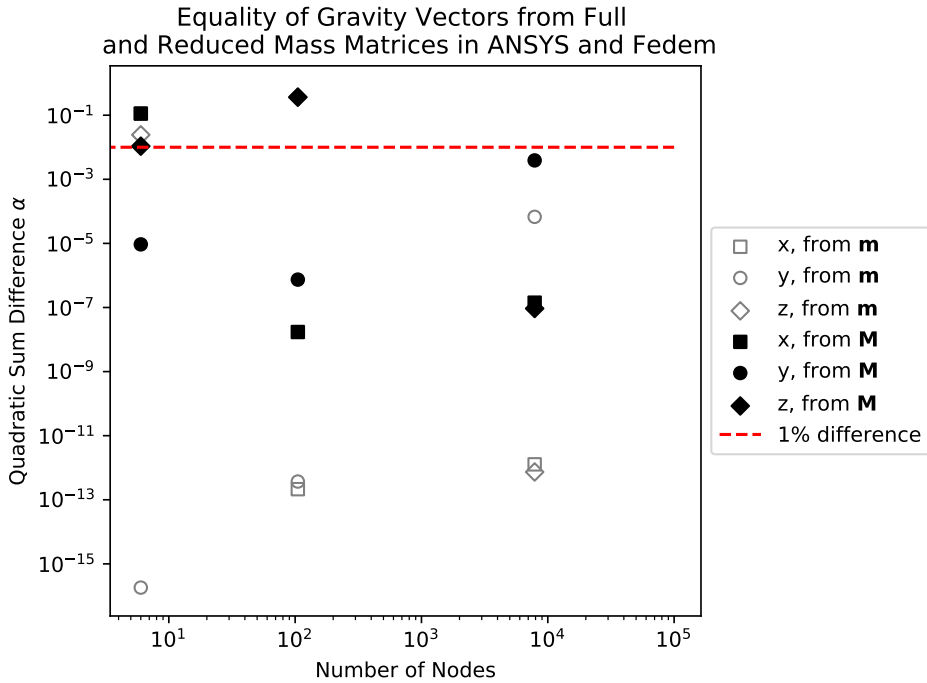


Figure 4.3: Relative difference of the gravity vector from Fedem and ANSYS, calculated with Fedem vectors used as reference

Figure 4.3 presents the resulting differences α between the gravity vectors. Resulting vectors when using the full mass matrix \mathbf{M} are shown in filled black color, while resulting vectors when using the reduced mass matrix \mathbf{m} are shown in un-filled grey. The results are very varied: some gravity vectors are nearly identical to gravity vectors from Fedem, namely the small model's x vector, the medium model's x and y vector and the large model's z and x vector, all from the reduced mass matrix. They have all $\alpha < 10^{-11}$. If one were to set $\alpha = 0.01$, corresponding to a 1% difference between ANSYS and Fedem vectors, as an acceptance limit, 12 out of 18 vectors are considered to have acceptable results. The worse result are obtained in the z vector from the medium model. Here, using both the full and reduced mass matrix gives 36.5% difference between ANSYS and Fedem vectors. Interestingly, using the reduced mass matrix generally gives better results than using the full mass matrix, with 5 vectors having $\alpha < 10^{-11}$.

There seem to be no obvious patterns regarding what direction gives the best results: both x , y and z vectors have α values close to the 1% difference mark, but are also almost identical to the Fedem reference in other models. Surprisingly, the number of nodes in the model seems to have almost no effect on the α values, although the large model gives acceptable results, with $\alpha < 3.9 \cdot 10^{-3}$ for all vectors.

4.4 Replicating Fedem Gravity Vectors

Due to the varied results when calculating gravity vectors from ANSYS matrices, gravity vectors using \mathbf{M} and \mathbf{H} matrix from Fedem have been calculated in order to replicate the results given by Fedem. The medium model is used for testing, and results are shown in Table 4.3.¹

	\mathbf{g}_x	\mathbf{g}_y	\mathbf{g}_z
α	8.15×10^{-9}	8.94×10^{-9}	3.34×10^{-8}

Table 4.3: Verifying the calculations made in the ModRed extension by replicating gravity vectors from Fedem by using Fedem matrices in all calculations

The results are very close to what Fedem gives, with all vector directions having $\alpha < 3.34 \times 10^{-8}$. Even better results could probably be obtained if a higher precision on the exported Fedem matrices were chosen. Here, six decimals were used. These results verify that the calculations performed in the ModRed extension are able to replicate calculations performed by Fedem. Further comments to why the gravity vectors from using ANSYS matrices do not show better results are discussed in Section 5.1.

4.5 Generation of FTL Files in ANSYS

For models to be displayed in Fedem, a `.ftl` file of the model needs to be generated. It defines node coordinates as well as nodal and element connectivity in the model. When the *Export* button in the ModRed extension is pressed, a `.ftl` file of the model is generated along with binary `.fmx` files containing the reduced stiffness, mass and gravity vectors. For the simple twoQUAD4 model, the generated `.ftl` file is given as

FTL file for the twoQUAD4 model

```

1 FTLVERSION{4 ASCII}
2 #
3 # FTL file generated by ModRed at 2019-06-07 08:59:07
4 #
5
6 #
7 # Nodal coordinates
8 #
9 NODE{1 1 0.0 0.0 0.0}
10 NODE{2 0 1.0 0.0 0.0}
11 NODE{3 1 2.0 0.0 0.0}
12 NODE{4 0 0.0 -1.0 0.0}

```

¹Performed in `test_results.py`, in method `test_quadraticSumDifference.gravityVector_beam.Fedem`

```
13 NODE{5 0 1.0 -1.0 0.0}
14 NODE{6 0 2.0 -1.0 0.0}
15 #
16 # Element definitions
17 #
18 QUAD4{1 5 4 1 2 {PTHICK 1} {PMAT 1}}
19 QUAD4{2 6 5 2 3 {PTHICK 1} {PMAT 1}}
20 #
21 # Local coordinate systems
22 #
23 PCOORDSYS{2 0 0 0 0 0 1 1 0 0}
24 #
25 # Material properties
26 #
27 PMAT{1 2.07e+11 8.02e+10 0.29 7820}
28 #
29 # Shell thicknesses
30 #
31 PTHICK{1 0.02}
32 #
33 # End of file
```

Currently, dummy values for the coordinate system, material properties and shell thickness is inserted, but should in the future be read from the ANSYS API. A future version of Fedem should also be able to detect that the model is reduced externally, and then use the `.fmx` files created by ModRed in analyses. This could for example be accomplished by reading the header of the `.ftl` file. If it says "FTL file generated by ModRed at ...", Fedem should recognize that it is an externally reduced model, and not start its own reducer.

Figure 4.4 shows the two QUAD4 element opened in Fedem from the ModRed generated FTL file. The external nodes in the top left and right, with node ID 1 and 3, are correctly read and marked with a green triad.

4.6 Screencast Demonstration of the ModRed Extension

A video demonstrating the features of ModRed has been made and can be watched at <https://youtu.be/-E5c-ZojjsE>. It is demonstrated how to import models, the different settings for the extension, as well as how to import the generated `.ftl` file into Fedem.

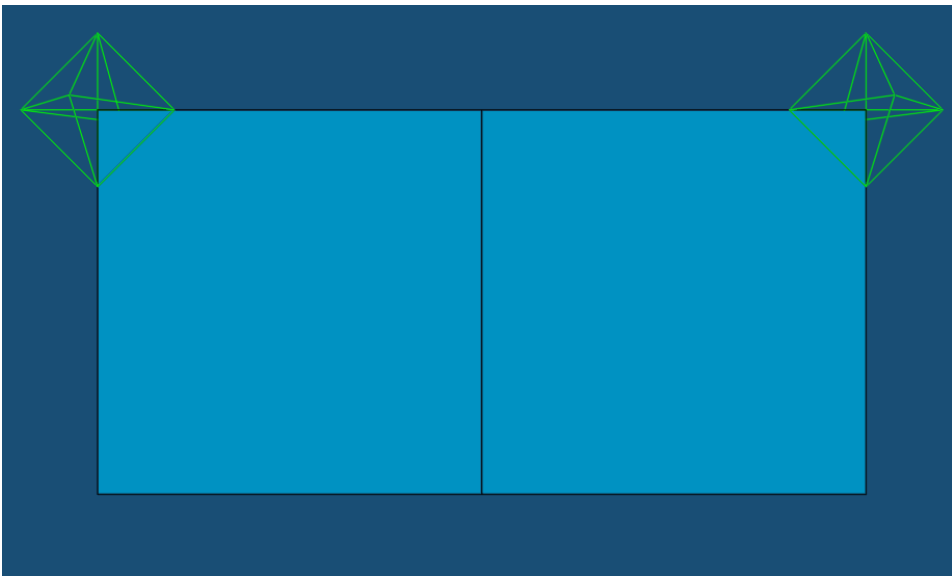


Figure 4.4: Visualisation of the twoQUAD4 element in Fedem, read from the ModRed generated FTL file

Chapter 5

Discussion and Future Work

5.1 Instability of Results

When observing the results obtained from the current version of the ModRed extension, there are some interesting observations to note:

- The full mass matrix M from ANSYS is shown to have much similarity with the mass matrix generated by Fedem when the number of nodes is sufficiently high (over 10^2).
- The reduced mass matrices from ANSYS show less similarity, from 10 to 15% difference from Fedem matrices. However, when the same reduced mass matrices are used for calculating the gravity vector, some very good results are obtained - in fact, the gravity vectors from the reduced mass matrix are more similar to what Fedem gives than the gravity vectors calculated from the full mass matrix.
- Testing the mass matrices for mass consistency after partitioning shows us that no element data is lost in the process.
- If using H and M from Fedem when calculating gravity vectors, identical results are obtained from the ModRed extension as what is calculated by Fedem, indicating that the ModRed extension is replicating the calculations performed in Fedem.

From this, it seems likely that the reason for the inconsistent results comes from how ANSYS is structuring its mass matrices, and that this is not handled correctly by the ModRed extension. In the following, it is shown that partitioning of an ANSYS mass matrix, with the identical partitioning method as used by a Fedem mass matrix, does not generate an ANSYS mass matrix partitioned identically as the Fedem reference.

Test of Matrix Partitioning

```
1 def test_partitionMatrix_difference_twoQUAD4 ( self ) :  
2     *****
```

```

3     Testing if Fedem and ANSYS matrices are partitioned correctly .
4     Using the partitioned Fedem matrix as reference , as this is proven to yield
5     the identical gravity vector as Fedem calculates .
6     """
7     dict.fedem = ModRed.readMappingFileFedem(self._resourcesFolder +
8     "\\twoQUAD4" + "\\Fedem" + "\\MEQN.res")
9     dict_ansys = ModRed.readMappingFile(self._resourcesFolder + "\\twoQUAD4" +
10    "\\Ansys" + "\\M_full_lumped.mapping")
11    M_full_fedem = ModRed.readFedemMatrix(self._resourcesFolder + "\\twoQUAD4" +
12    "\\Fedem" + "\\M_full.res")
13    M_full_ansys = ModRed.readMMFMatrix(self._resourcesFolder + "\\twoQUAD4" +
14    "\\Ansys" + "\\M_full_lumped.mmf", "full")
15
16    eNodes = [1, 3]
17    iNodes = [2, 4, 5, 6]
18    fedem = ModRed.partitionMatrix(M_full_fedem, eNodes, iNodes, dict.fedem, 6,
19    sorting="equation")
20    ansys = ModRed.partitionMatrix(M_full_ansys, eNodes, iNodes, dict_ansys, 6,
21    sorting="equation")
22
23    for row in range(0, ansys.RowCount):
24        for col in range(0, ansys.ColumnCount):
25            if row==col: # both matrices are diagonal
26                self.assertAlmostEqual(fedem[row, col], ansys[row, col], places=1)

```

Result of Matrix Partitioning Test

r: 0,	c: 0,	fedem: 39.1,	ansys: 39.1
r: 1,	c: 1,	fedem: 39.1,	ansys: 39.1
r: 2,	c: 2,	fedem: 39.1,	ansys: 39.1
r: 3,	c: 3,	fedem: 0.278214,	ansys: 0.001303333333333
r: 4,	c: 4,	fedem: 0.393552,	ansys: 0.001303333333333
r: 5,	c: 5,	fedem: 0.651667,	ansys: 1.303333333333e-13
r: 6,	c: 6,	fedem: 39.1,	ansys: 39.1
r: 7,	c: 7,	fedem: 39.1,	ansys: 39.1
r: 8,	c: 8,	fedem: 39.1,	ansys: 39.1
r: 9,	c: 9,	fedem: 0.278214,	ansys: 0.001303333333333
r: 10,	c: 10,	fedem: 0.393552,	ansys: 0.001303333333333
r: 11,	c: 11,	fedem: 0.651667,	ansys: 1.303333333333e-13
r: 12,	c: 12,	fedem: 39.1,	ansys: 78.2
r: 13,	c: 13,	fedem: 39.1,	ansys: 78.2
r: 14,	c: 14,	fedem: 39.1,	ansys: 78.2
r: 15,	c: 15,	fedem: 0.278214,	ansys: 0.002606666666667
r: 16,	c: 16,	fedem: 0.393552,	ansys: 0.002606666666667
r: 17,	c: 17,	fedem: 0.651667,	ansys: 2.606666666667e-13
r: 18,	c: 18,	fedem: 39.1,	ansys: 39.1
r: 19,	c: 19,	fedem: 39.1,	ansys: 39.1
r: 20,	c: 20,	fedem: 39.1,	ansys: 39.1
r: 21,	c: 21,	fedem: 0.278214,	ansys: 0.001303333333333

r: 22, c: 22,	fedem: 0.393552,	ansys: 0.00130333333333
r: 23, c: 23,	fedem: 0.651667,	ansys: 1.30333333333e-13
r: 24, c: 24,	fedem: 78.2,	ansys: 78.2
r: 25, c: 25,	fedem: 78.2,	ansys: 78.2
r: 26, c: 26,	fedem: 78.2,	ansys: 78.2
r: 27, c: 27,	fedem: 0.556428,	ansys: 0.00260666666667
r: 28, c: 28,	fedem: 0.787103,	ansys: 0.00260666666667
r: 29, c: 29,	fedem: 1.30333,	ansys: 2.60666666667e-13
r: 30, c: 30,	fedem: 78.2,	ansys: 39.1
r: 31, c: 31,	fedem: 78.2,	ansys: 39.1
r: 32, c: 32,	fedem: 78.2,	ansys: 39.1
r: 33, c: 33,	fedem: 0.556428,	ansys: 0.00130333333333
r: 34, c: 34,	fedem: 0.787103,	ansys: 0.00130333333333
r: 35, c: 35,	fedem: 1.30333,	ansys: 1.30333333333e-13

The resulting printout of every diagonal item in the mass matrix shows that the partitioning of the two matrices is not identical. Using the partitioned Fedem matrix as a reference, one can see on line 13 that the partitioning of the ANSYS matrix can not be correct. The matrix entry should equal what Fedem gives (39.2), but is set to 78.2. This obviously wrong, as the translational DOFs from Fedem and ANSYS are similar (39.1 and 78.2) and should be set to the same position in both matrices. Attempts to resort the partitioned ANSYS matrix from other parameters than what Fedem uses - namely ascending equation number - has not yet proven successful.

The reason this issue is not straight-forward to resolve, is that it is limited what matrices could be exported from Fedem and ANSYS. From Fedem, one can obtain the full mass matrix, the M_{ee} part of the partitioned matrix and the reduced mass matrix. From ANSYS, you can only obtain the full and reduced mass matrices. This means that the only reference available for how to partition the mass matrix correctly, is M_{ee} from Fedem. It has therefore been assumed through the whole development process that ANSYS partitions the mass matrix identical to how Fedem does it. However, when it is proven that partitioning is handled differently in ANSYS, there is no M_{ee} matrix from ANSYS to use as reference, verifying that the partitioning has been done correctly.

This unresolved issue in matrix partitioning is believed to be the cause for the unstable results when calculating gravity vectors. The reason some of the gravity vectors gave very precise results is probably because large portions of the mass matrix *is* partitioned correctly, as seen in the example. The gravity vectors that gave very good results are therefore believed to have most parts of the mass matrix used for calculating them partitioned correctly.

The issue of matrix partitioning could also explain why the gravity vectors calculated from the reduced mass matrix give better results than using the full mass matrix for this. That is because when using the reduced mass matrix, the issue of matrix partitioning is removed, as the matrix is already partitioned internally by ANSYS before the reduced mass matrix is created from pre- and post-multiplying with \mathbf{H}

Due to the unstable results, it is difficult to make a general conclusion for the accuracy

of results obtained from ModRed. However, it is shown that the extension has the capability of producing precise results, with $\alpha < 10^{-11}$ for some of the gravity vectors. If the assumption that the root of unstable results lies in handling matrix partitioning, there should not be too much work in resolving the issue. In any way, more testing is needed, as well as some extra functionality discussed next, for the ModRed extension to be used effectively.

5.2 Future Work

In the following, suggestions for future development of the ModRed extension are listed. The list is based on experiences from developing the current version of the app, and will hopefully be a helpful reference in the future.

- Extending the elements supported by the ModRed extension should be a priority. Not only will it enable extra functionality for the extension, but will also extend the testing possibilities, thus making it easier to determine its accuracy compared to Fedem.
- When using ANSYS' automatic meshing capabilities, ANSYS is likely to create a mesh containing many different element types for the same model. Shell/membrane elements are automatically used for thin surfaces, different types of solid elements could be used for thicker surfaces, and contact elements are used where ANSYS finds it likely to be a contact point between two surfaces. If the model is to be exported to a `.ftl` file, the different elements created by ANSYS need to be mapped to a corresponding, similar Fedem element. This is currently not supported, as only SHELL181 elements are handled by the extension. "
- When importing the `.ftl` files into Fedem, it must be detected that the model has been externally reduced by ANSYS. This could for example be performed by reading the header of the FTL file, which will say *FTL file created by ModRed at [Date and time]*. If the text is detected, Fedem knows that the model is already reduced.

In the current version of ModRed, the reduced mass matrix is used for calculating the gravity vectors needed by Fedem. Due to this, the calculations can be performed faster, as the \mathbf{H} matrix is no longer needed for calculations, and thus does not have to be read line-for-line into the program. However, a future version of the ModRed extension should probably use the full mass matrix for gravity vector calculations as this should give better results.

- For future development, it is highly recommended to implement the code base in C# instead of the current IronPython language. The reasons for this are many:
 - Visual Studio, the text editor used for developing the ModRed extension, has a very limited support for the IronPython framework. For example is no auto-completion available, and method headers will normally not appear when calling methods written in the extension. This means that every input parameter

for methods used must be manually checked, resulting in tedious manual work, as well as code that is much more likely to contain errors.

- The ANSYS ACT API can be read by a C# environment, meaning that every method available through the API will be listed as auto-completion suggestions when handling ANSYS ACT objects. As the available ACT extension examples are relatively sparse, the development process often contains much trial-and-error for figuring out what methods from the API to use. Due to the lack of auto-completion in Visual Studio, all calls to the API must either be investigated through the ACT Console in ANSYS Mechanical, looked up online or in the Developer’s Guide. This is very time-consuming, but the added help from the IDE if using C# would make this much easier.
- The Visual Studio debugger for IronPython is very unstable and is very likely to crash when connected to ANSYS Mechanical for debugging. Debugging the code is therefore often easier to perform with print statements.
- Overriding ANSYS’ choice of elements for a mesh is proven to be a more complicated process than imagined. One can add the APDL command

APDL command for overriding ANSYS’ choice of mesh elements

et, matid, 181

However, as this is a command sent directly to the APDL solver, the override of element type is not detected by ANSYS Mechanical, meaning there is no graphical feedback telling the user that the material type has been changed. In addition, the override of element type is not reflected in ANSYS ACT, meaning that internal node and element numbering could be altered at solve time.

- When performing model reduction through the ModRed extension, ANSYS Mechanical tells the user that *an unknown error occured*. However, the solver output from the APDL solver contains no errors, and all internal APDL solver files are generated. The error message is therefore probably occurring because of two reasons:
 1. When defining CMS as the solution type in APDL, we are accessing a solution type that is not supported by ANSYS Mechanical. Thus it is likely that the program defaults to show an error message, because it has no graphical interface for showing the CMS solution type.
 2. Many additional files are exported from ANSYS through APDL commands. Additionally, ANSYS moves all solver files to a temporary folder while the APDL solver is running. The export operations are therefore likely to be conflicting with ANSYS’ internal file movement, which could be the cause for a default error message.
- In order to obtain 16-decimal precision on the exported mass and stiffness matrix, the full mass and the reduced stiffness and mass matrices are currently exported twice. This is because it is found that 16-decimal precision matrices are only exported in the Harwell-Boeing (.hbmat) format, accessed through the fileaux2

-> `hbmat` command. After export to the `.hbmat` format, the matrices are imported to an APDL Math matrix, and again exported to the Matrix Market Format. This format was found to be much easier to handle than the `.hbmat` format, because custom import methods for matrices had to be written, as the Math.NET package did not support matrix import in any of the formats. However, this double export is definitely a time- and space-consuming operation, and a method for reading `.hbmat` matrices should be made. Optionally, the APDL command `*MWRITE` should do exactly what we want: writing a matrix to a file in a user-formatted sequence. Unfortunately, it has not been possible to make this command work when writing commands directly to the `ds.dat` file.

Chapter 6

Conclusion

The ANSYS ACT extension ModRed has been developed in order to perform model reduction in ANSYS. By using it, one can access the reduced matrices as well as the needed transformation matrix after model reduction in ANSYS Mechanical. The matrices from ANSYS have been tested for equality against reference matrices from Fedem. The obtained results were varied:

- For large models with more than 100 nodes, the full mass matrices from Fedem and ANSYS are close to identical. However, the reduced mass matrices show overall poorer results, with the comparison method used indicating 10% to 15% difference between the reduced ANSYS and Fedem matrices.
- Despite being relatively unequal, the reduced matrices have identical mass to the full matrices, indicating that no information is lost in the reduction process. When calculating gravity vectors the reduced mass matrices surprisingly give overall better results than when using the full mass matrices in calculations.

A possible explanation to the varied results may be found in the way ANSYS structures their mass matrices differently than Fedem. It has been discovered that matrix partitioning in ANSYS is performed differently than what is the case in Fedem. This may explain why the results seem partially inconsistent. Extended testing using a wider range of test models, as well as resolving the issue on matrix partitioning should therefore be performed before one can conclude on the accuracy of results acquired from the extension. Despite the varied results, the extension has proven the potential to become an easy-to-use method for integrating ANSYS solver capabilities in the Fedem software.

References

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Appendix

A

Source Code for the ModRed ACT Extension

A.1 ModRed.py

ModRed.py

```
1 import clr
2 clr.AddReferenceToFileAndPath(r"C:\Program Files\ANSYS
   Inc\v192\Addins\ACT\bin\Win64\MathNet.Numerics.dll") # Using the Math.NET dll.
   Can not use NumPy, as this is not supported by IronPython.
3 import MathNet.Numerics.LinearAlgebra as la # math library
4 import os
5 import subprocess
6 import datetime # For writing current date in generated FTL file
7 import ctypes
8 import cPickle as pickle # For saving object to file
9 import System
10
11
12 def createUnitVector (pos, nnodes, cmodes=0):
13     """
14     Creates a vector with unit length at the specified position pos.
15
16     :param pos: Int of position to give unit length. 1=UX, 2=UY, 3=UZ, 4=ROTX,
17     5=ROTY, 6=ROTZ
18     :param nnodes: Int of number of nodes to create unit length for.
19     :param cmodes: Number of component nodes to add unit length
20     return matrix of size ((nnodes*6)+cmodes x 1)
21     """
22     if (pos <= 0):
```

```

22     raise Exception("pos argument must be int > 0")
23
24     u = la.Matrix[System.Double].Build.Dense(nnodes*6 + cmodes, 1)    # initializing
    with zeroes
25     for i in range(pos-1, nnodes*6, 6):    # Adding 1 at position pos
26         u[i, 0] = 1
27     for i in range(nnodes*6, nnodes*6 + cmodes):    # Filling component mode
    entries with 0s
28         u[i, 0] = 0
29     return u
30
31
32
33
34 def onCreateModRed(analysis):
35     """
36     Adding the ModRed model object in the model tree
37
38     :param analysis: The currently active analysis
39     """
40     if str(analysis.AnalysisType) == "Modal":
41         # Use the analysis to create the APDL Based Result Evaluation result .
42         analysis.CreateLoadObject("modred", "ModRed")
43
44     else:
45         # Display an error message in the Mechanical message log.
46         ExtAPI.Application.LogError("Can only select ModRed for Modal analysis.")
47
48
49 def saveObject(obj, fname):
50     """
51     Saves the current object to _fedemDir_
52     """
53     testResourcesPath = ExtAPI.ExtensionManager.CurrentExtension.InstallDir +
    "\\test-resources"
54     with open(testResourcesPath + "\\\" + fname + ".p", "wb") as input:
55         pickle.dump(obj, input, pickle.HIGHEST_PROTOCOL)
56
57
58 def onExportData(analysis):
59     """
60     :param analysis: Ansys.ACT.Automation.Mechanical.Analysis object.
61     - Calculates gravity vectors from reduced mass matrix
62     - Creates FTL file for Fedem
63     - Creates FMX files from fmxWriter DLL
64     - All files are saved in solverData._fedemDir
65     """
66
67     ExtAPI.Log.WriteMessage("Exporting data for Fedem...")
68     load = analysis.GetLoadObjects("ModRed")[0]

```

```

69 solverData = SolverData( analysis , load)           # Creating a SolverData object
    for the current analysis .
70
71 # Collecting exported matrices in Math.NET format
72 M_red = readMMFMatrix(analysis.WorkingDir + "M_red.mmf", "sub")
73 K_red = readMMFMatrix(analysis.WorkingDir + "K_red.mmf", "sub")
74
75 G = calculateGravityVectorReduced (M_red, len( solverData . _eNodes),
    int( solverData . _cModes)) # Using reduced mass matrix
76 # Generating FTL file for Fedem
77 ftlFile = open(os.path.join( solverData . _fedemDir, solverData . _filename + ". ftl "),
    "w") # Creating a new ftl file
78 generateFTL(solverData.getNodes(), solverData.getElements(), solverData . _eNodes,
    ftlFile ) # Generating FTL file to be placed in the solver directory .
79 ExtAPI.Log.WriteMessage("FTL file " + str( solverData . _filename ) + ". ftl written to
    " + solverData . _fedemDir)
80
81 # Generating FMX file for Fedem
82 generateFMX(mathNET2list(G), mathNET2list(M_red), mathNET2list(K_red), solverData)
83
84 def calculateGravityVectorReduced (M_red, nENodes, cmodes):
85     """
86     Calculates gravity vectors for Fedem from the reduced mass matrix .
87     :param M_red: The reduced mass matrix .
88     :param nENodes: Number of selected external nodes
89     """
90     # Creating reduced unit vectors
91     u_x = createUnitVector (1, nENodes, cmodes)
92     u_y = createUnitVector (2, nENodes, cmodes)
93     u_z = createUnitVector (3, nENodes, cmodes)
94     u = u_x.Append(u_y).Append(u_z)
95
96     G_red = M_red.Multiply(u)
97     return G_red
98
99
100
101
102
103 def calculateGravityVector (M_full, H, nENodes, nINodes, dofs):
104     """
105     Calculates gravity vectors for Fedem from the full mass matrix and H matrix .
106     Follows equations from page 118 in "Virtual Testing of Mechanical Systems"
107     :param M_full: The full , partitioned mass matrix .
108     :param H: The H matrix
109     H =
110     [[I 0],
111     [B, phi ]]
112     :param nENodes: Number of selected external nodes
113     :param nINodes: Number of selected internal nodes

```

```

114 :param dofs: Number of degrees of freedom for the element
115 """
116 # Extracting the partitions of the M matrix
117 partitions = extractPartitions (M_full, nENodes, dofs)
118 M_ee = partitions [0]
119 M_ei = partitions [1]
120 M_ie = partitions [2]
121 M_ii = partitions [3]
122
123 # Creating unit vectors :
124 u_x_e = createUnitVector (1, nENodes)
125 u_x_i = createUnitVector (1, nINodes)
126 u_y_e = createUnitVector (2, nENodes)
127 u_y_i = createUnitVector (2, nINodes)
128 u_z_e = createUnitVector (3, nENodes)
129 u_z_i = createUnitVector (3, nINodes)
130
131 # Gravitational forces :
132 G_x_i = M_ii.Multiply(u_x_i).Add(M_ie.Multiply(u_x_e)) # M_ii*u_x_i + M_ie *
    u_x_e
133 G_x_e = M_ei.Multiply(u_x_i).Add(M_ee.Multiply(u_x_e))
134 G_y_i = M_ii.Multiply(u_y_i).Add(M_ie.Multiply(u_y_e))
135 G_y_e = M_ei.Multiply(u_y_i).Add(M_ee.Multiply(u_y_e))
136 G_z_i = M_ii.Multiply(u_z_i).Add(M_ie.Multiply(u_z_e))
137 G_z_e = M_ei.Multiply(u_z_i).Add(M_ee.Multiply(u_z_e))
138
139 # Collecting in one matrix :
140 G_x = G_x_e.Stack(G_x_i)
141 G_y = G_y_e.Stack(G_y_i)
142 G_z = G_z_e.Stack(G_z_i)
143
144 # Reducing to unit gravitational acceleration on the full matrix :
145 G_x = H.Transpose().Multiply(G_x)
146 G_y = H.Transpose().Multiply(G_y)
147 G_z = H.Transpose().Multiply(G_z)
148
149 # Collecting gravity vectors in one matrix
150 G = G_x.Append(G_y).Append(G_z)
151 return G
152
153
154
155 def extractPartitions (mat, numENodes, dofs):
156 """
157     Extracts the partitions
158     M =
159     [[M_ee, M_ei],
160     [M_ie, M_ii]]
161     from the matrix mat
162     M_ee of size (numENodes*dofs, numENodes*dofs)

```

```

163
164 :param mat: Symmetric Math.NET matrix already partitioned into M_ee, M_ei, ...
165 :param eNodes: Number of selected external nodes
166 :param dofs: Number of degrees of freedom per node
167 """
168 nRows = mat.RowCount
169 nCols = mat.ColumnCount
170
171 M_ee = mat.SubMatrix(0, numENodes*dofs, 0, numENodes*dofs) # Top left of M
172 M_ei = mat.SubMatrix(0, numENodes*dofs, numENodes*dofs, nCols - numENodes * dofs)
    # Top right
173 M_ie = mat.SubMatrix(numENodes*dofs, nRows - numENodes * dofs, 0,
    numENodes*dofs) # Bottom left
174 M_ii = mat.SubMatrix(numENodes*dofs, nRows - numENodes * dofs, numENodes*dofs,
    nCols - numENodes*dofs) # Bottom right
175 return [ M_ee, M_ei, M_ie, M_ii ]
176
177 def mathNET2list(matrix):
178     """
179     Returns a Python list of the items in the matrix
180
181     :param matrix: Math.NET matrix
182     return One-dimensional list of items in matrix saved column-first.
183     """
184     list = [0] * (matrix.ColumnCount * matrix.RowCount)
185     index = 0
186     for col in range(0, matrix.ColumnCount):
187         for row in range(0, matrix.RowCount):
188             list [index] = matrix[row, col]
189             index += 1
190     return list
191
192
193
194 def onSolve(load, stream):
195     """
196     Adds the required APDL commands to the solver input (ds.dat) file . Activates when
    the user hits "Solve"
197
198     :param load: the load associated to the callback . Interface IUserLoad
199     :param stream: a System.IO.StringWriter object , to which solver commands should be
    appended (represents the ds.dat file )
200     """
201
202     # Collecting user input :
203     nodeSelectionMethod = load . Properties [ "Generation" ] . Properties [ "Geometry" ] . Value
204     method = load . Properties [ "Method" ] . Properties [ "ReductionMethod" ] . Value
205     filename =
        load . Properties [ "Method" ] . Properties [ "ReductionMethod" ] . Properties [ "Name" ] . Value
        # string

```

```

206  nmode =
      load.Properties ["Method"].Properties ["ReductionMethod"].Properties ["Nmode"].Value
      # string
207  isUseLumped =
      load.Properties ["Method"].Properties ["ReductionMethod"].Properties ["LumpedMatrix"].Value
      # string
208  eNodes = []
209  if (nodeSelectionMethod=="Geometry"):
210      eNodes = getENodes(load) # List [ int ]
211  if (nodeSelectionMethod=="RBE2 Import"):
212      eNodes = getRBE2Nodes(load)
213  isExport = load.Properties ["Export"].Properties ["IsExport"].Value
214  if (isExport == "Yes"):
215      isExport = True
216  else :
217      isExport = False
218
219  stiffnessName = "Kmat"
220  massName = "Mmat"
221  matrixFormat = "MMF" # Set this to ".matrix" for lower precision , but more
      readable matrices
222
223  #
224  # Writing solver commands to the ds.dat file
225  #
226  stream.WriteLine(" finish ")
227  stream.WriteLine("/filename, " + filename) # Name of the super element file to
      be generated
228  stream.WriteLine("save")
229  stream.WriteLine("/solu")
230  stream.WriteLine("antype, substr") # Defining a substructure analysis type
      (includes both Guyan and CMS)
231  if (method == "CMS"): # If CMSOPT needs to be set
232      cmsmeth = "fix"
233      nmode = str(nmode)
234      freqb = ""
235      freqe = ""
236      fbddef = ""
237      fbdval = ""
238      iokey = "tcms"
239      if (iokey=="tcms"):
240          stream.WriteLine("outpr, nsol, all") # Print command that must be
      defined if iokey==tcms
241          stream.WriteLine("cmsopt, " + cmsmeth + "," + nmode + "," + freqb + "," + freqe
      + "," + fbddef + "," + fbdval + "," + iokey)
242  sename = filename # name of superelement matrix file
243  sematr = str(2) # generate stiffness and mass matrices
244  sepr = str(0) # Do not print superelement matrices or load vectors
245  sesst = str(0) # Do not save space for stress stiffening

```

```

246 stream.WriteLine("seopt, " + sename + ", " + sematr + ", " + sepr + ", " + sesst +
    ", ,")
247 for nodeID in eNodes:
248     stream.WriteLine("m, " + str(nodeID) + ", all")
249 if (isUseLumped == "Yes"):
250     stream.WriteLine("lumpm, on")           # Using a lumped mass matrix
251     stream.WriteLine("alls ")
252     stream.WriteLine("solve")
253     stream.WriteLine("save")
254     stream.WriteLine(" finish ")
255     stream.WriteLine("! Master node selection finished ")
256
257 if ( isExport ):           # Exporting matrices if selected
258     stream.WriteLine("! Exporting matrices ... ")
259     # Creating H matrix. H = [CST NOR]
260     stream.WriteLine("*dmat, cst, D, import, tcms, " + filename + ".tcms, cst")
261     # Constraint mode data
262     stream.WriteLine("*dmat, nor, D, import, tcms, " + filename + ".tcms, nor")
263     # Fixed–interface normal mode data
264     stream.WriteLine("save")
265     if (matrixFormat == "MMF"):
266         stream.WriteLine("*export, cst, MMF, CST.mmf")           # Exporting
267         constraint modes
268         stream.WriteLine("*export, nor, MMF, NOR.mmf")           # Exporting
269         fixed –interface normal mode data
270
271     stream.WriteLine("*smat, nod2solv, D, import, full , " + filename + ".full ,
272     nod2solv")           # Importing the mapping vector internal –> solver ordering
273     stream.WriteLine("*vec, mapback, I, import, full , " + filename + ".full ,
274     back")           # Importing the BACK nodal mapping vector for external –> internal
275     ordering .
276
277     stream.WriteLine("/aux2")           # Manipulating binary files
278     stream.WriteLine("FILEAUX2, " + filename + ", sub")           # Specifying
279     to dump reduced matrices file
280     stream.WriteLine("HBMAT, M_red, hbmat, , ASCII, MASS, NO")           #
281     Dumping reduced mass matrix to Harwell–Boeing format for 16–decimal precision
282     stream.WriteLine("HBMAT, K_red, hbmat, , ASCII, STIFF, NO")           # Dumping
283     reduced stiffness matrix
284
285     stream.WriteLine("FILEAUX2, " + filename + ", full")           # Specifying to
286     dump full mass matrix
287     stream.WriteLine("HBMAT, M_full, hbmat, , ASCII, MASS, NO, YES")           #
288     Dumping full mass matrix in Harwell–Boeing format
289     stream.WriteLine(" finish ")
290     stream.WriteLine("*smat, M_red, D, import, HBMAT, M_red.hbmat, ASCII") #
291     Creating sparse matrix from Harwell–Boeing format. Preserving 16–decimal precision
292     stream.WriteLine("*smat, M_full, D, import, HBMAT, M_full.hbmat, ASCII")
293     stream.WriteLine("*smat, K_red, D, import, HBMAT, K_red.hbmat, ASCII")
294

```

```

282         stream.WriteLine("export, M_red, MMF, M_red.mmf")           # Exporting
to Matrix Market Format. Preserves 16-decimal precision
283         stream.WriteLine("export, M_full, MMF, M_full.mmf")
284         stream.WriteLine("export, K_red, MMF, K_red.mmf")
285
286         if (matrixFormat == ".matrix"):
287             stream.WriteLine("PRINT, Kmat, " + stiffnessName + ".matrix") #
Printing matrix directly in easy-to-read format
288             stream.WriteLine("PRINT, Mmat, " + massName + ".matrix")
289
290         stream.WriteLine("save")
291
292     stream.WriteLine("/eof")           # Quits correctly
293
294
295
296
297 def generateFMX(G, M_red, K_red, solverData):
298     """
299     This will save FMX files of G, M_red and K_red at the solver directory .
300
301     :param G: One-dimensional list of gravitational vectors saved column-first.
302     :param M_red: One-dimensional list of reduced mass matrix saved column-first.
303     :param K_red: One-dimensional list of reduced stiffness matrix saved column-first.
304     :param solverData: SolverData object .
305
306     """
307     # Converting to double slashes and in bytes representation
308     file = solverData._fedemDir + "\\\" + solverData._filename
309     file = file.replace("\\", "\\\\")
310     file = bytes(file, 'utf-8')
311
312     # Skriver stivhetsmatrise
313     status = writeFMX(file, 1, K_red)
314
315     # Skriver massematrise
316     status = writeFMX(file, 2, M_red)
317
318     # Skriver gravitasjonskrefter
319     status = writeFMX(file, 3, G)
320
321     ExtAPI.Log.WriteMessage("FMX files written to " + solverData._fedemDir)
322
323
324
325 def writeFMX(file, ityp, data):
326     """
327     Writes a square matrix as a binary FMX-file for FEDEM, using the fmxWriter DLL
328
329     arg file : Full path to the fmx-file to be written

```

```

330     arg ityp : 1= stiffness matrix, 2=mass matrix, 3=gravity force vectors
331     arg data : Matrix content, column-wise storage
332     return : Zero on success, otherwise negative
333     """
334     # Loading DLL for writing FMX files:
335     extensionDir = ExtAPI.ExtensionManager.CurrentExtension. InstallDir
336     dll = ctypes.cdll.LoadLibrary(extensionDir + "\\fmxWriter\\fmxWriter.dll")
337     cfil = file
338     ctyp = ctypes.c_int(ityp)
339     cdat = (ctypes.c_double*len(data))()
340     cdat[:] = data
341     clen = ctypes.c_int(len(data))
342     return dll.WRITEFMX(cfil, ctypes.byref(ctyp), cdat, ctypes.byref(clen), len(file))
343
344
345 def getENodes(load):
346     """
347     Returning list with node numbers of selected external nodes
348
349     :param load: The current load object
350     :return List[int] Sorted list with integers representing node numbers of current
351     selected nodes.
352     """
353     selectedIDs =
354         load.Properties["Generation"].Properties["Geometry"].Properties["Geometry"].Value.Ids
355     return sorted(selectedIDs) # List[int]
356
357 def generateFTL(nodes, elements, eNodes, file):
358     """
359     Generates the FTL file for the current model to be used by Fedem.
360     The FTL file is placed in the solverData.fedemDir.
361
362     :param nodes: list [Node] of all nodes in the model. Sorted ascending by id
363     :param elements: list [Element] of all elements in the model. Sorted ascending by id
364     :param eNodes: list [int] of ids of external nodes
365     :param file : FTL file object
366     """
367     #
368     # Writing file header
369     #
370     date = datetime.datetime.now().strftime("%Y-%m-%d %H:%M:%S")
371     file.write("FTLVERSION{4 ASCII}\n")
372     string = "#\n# FTL file generated by ModRed at " + str(date) + "\n#\n#"
373     file.write(string)
374     string = "#\n# Nodal coordinates\n#"
375     file.write(string + "\n")
376
377     #

```

```

378 # Writing NODE part
379 #
380 for node in nodes:
381     nodeString = getNodeString(node, eNodes)
382     file . write(nodeString + "\n")
383
384 #
385 # Writing element definitions part
386 #
387 thicknessId = 1
388 materialId = 1
389 file . write("#\n# Element definitions \n#\n")
390 for element in elements:
391     elString = getElementString(element, thicknessId, materialId)
392     file . write(elString + "\n")
393 #
394 # Writing coordinate system part
395 #
396 file . write("#\n# Local coordinate systems\n#\n")
397 file . write("PCOORDSYS{2 0 0 0 0 1 1 0 0}" + "\n")
398
399 #
400 # Writing material properties part
401 #
402 file . write("#\n# Material properties \n#\n")
403 file . write("PMAT{1 2.07e+11 8.02e+10 0.29 7820}" + "\n") # Dummy material
404
405 #
406 # Writing shell thicknesses part
407 #
408 file . write("#\n# Shell thicknesses \n#\n")
409 file . write("PTHICK{1 0.02}" + "\n")
410
411 #
412 # End of file
413 #
414 file . write("#\n# End of file ")
415
416 file . close() # Closing the created FTL file
417
418
419 def getNodeString(node, eNodes):
420     """
421     Returns node string to write in FTL file .
422
423     :param id: id of node to write to string
424     :param nodes: sorted list [Node] of all nodes in the model. Sorted by node id
425     :param eNodes: list [int] of ids of selected external nodes
426     :return string of NODE part to be written in FTL file
427     """

```

```

428     string = "NODE{"
429     string += str(node._id)
430     string += " "
431     if(node._id in eNodes):
432         string += "1"
433     else :
434         string += "0"
435     string += " "
436     string += str(node._x) + " " + str(node._y) + " " + str(node._z)
437     string += "}"
438     return string
439
440
441 def getElementString(element, thicknessId , materialId):
442     """
443     Returns string of ELEMENT part to be written in FTL file
444
445     :param element: Element object of the element to write
446     :param thicknessID: id of thickness PTHICK
447     :param materialID: id of material PMAT
448     """
449     string = "QUAD4{"
450     string += str(element._id) + " "
451     string += str(element._nodes[0]._id) + " "
452     string += str(element._nodes[1]._id) + " "
453     string += str(element._nodes[2]._id) + " "
454     string += str(element._nodes[3]._id) + " "
455     string += "{PTHICK "
456     string += str(thicknessId) + "}"
457     string += " "
458     string += "{PMAT "
459     string += str(materialId) + "}"
460     string += "}"
461
462     return string
463
464
465
466
467 def isValidNoModes(entity, property):
468     """
469     Returns true if valid input to component modes property. False if not.
470
471     :param entity : IUserObject. The entity containing the property .
472     :param property: ISimProperty. The property to check.
473
474     :return True if property is positive int. False if not.
475     """
476     try:
477         #ExtAPI.Log.WriteMessage(str(int(property.Value) >= 0))

```

```

478         return int(property.Value) >= 0
479     except ValueError:
480         #ExtAPI.Log.WriteMessage(str(False))
481         return False
482
483 def readMappingFileFedem(path):
484     """
485     Reads a Fedem mapping file that maps degree of freedom to equation.
486     The MEQN file is structured
487     column 0: dof      column 1: equation
488
489     1   25
490     2   26
491     3   27 ...
492
493     This means dof 1 equals equation 25.
494
495     :param path: File path of MEQN.res file from Fedem
496     :return Dictionary on the form
497     dict = {
498     "dof": equation
499     }
500     """
501     file = open(path, "r")
502     line = file . readline ()           # Reading header line
503     dict = {}
504     lines = file . readlines ()
505     for line in lines :
506         line = line . split ()
507         dof = int ( line [0])
508         eq = int ( line [1])
509         dict[dof] = eq
510     return dict
511
512 def readMappingFile(path):
513     """
514     Reads an ANSYS .mapping file that maps equation to corresponding node.
515
516     The .mapping file is structured :
517     column 0: equation      column 1: node      column 2: dof
518     Example:
519
520     1   5   UX
521     2   5   UY
522     3   5   UZ
523
524     This means equation 1 is related to node 5, dof UX
525     Relation node -> dof is
526     node 1, UX    => dof 1
527     node 1, UY    => dof 2

```

```

528     node 1, UX      => dof 3
529     node 2, ROTX   => dof 4 ...
530
531     node 2, UX      => dof 7
532
533     :param path: File path of ".mapping" file
534     :param dofDict: Dictionary relating internal dof for a node to its corresponding
                    numbering.
535     Example: "UX" = 1, "UY" = 2, "UZ" = 3, "ROTX" = 4
536     :return Dictionary
537     dict = {
538         "dof": "equation"
539     }
540     where "dof" is in ascending order from node numbers, i.e.:
541     dof = 1 => Node 1, degree of freedom nr 1 (UX)
542     dof = 2 => Node 1, degree of freedom nr 2 (UY)
543     dof = 7 => Node 2, degree of freedom nr 1 (UX)
544     dof = 12 => Node 2, degree of freedom nr 6 (ROTX)
545     """
546     dofDict = {
547         "UX": 1,
548         "UY": 2,
549         "UZ": 3,
550         "ROTX": 4,
551         "ROTY": 5,
552         "ROTZ": 6
553     }
554     dict = {}
555     file = open(path, "r")
556     line = file.readline() # Skipping first line
557     lines = file.readlines()
558     for line in lines:
559         line = line.split()
560         eq = int(line[0])
561         node = int(line[1])
562         dofText = line[2]
563         dof = node2dof(node, dofDict, 6, dofText) # Currently hard-typing in
                    6 dofs
564         dict[dof] = eq
565     return dict
566
567 def node2dof(node, dofDict, dofs, dofText):
568     """
569     Relates node number to dof number.
570
571     :param node: int of node number
572     :param dofs: Number of dofs for the node
573     :param dofText: string of dof in ANSYS .mapping file. Ex.: "UX", "UY",
574     """
575     dof = (node-1)*dofs + dofDict[dofText]

```

```

576     return dof
577
578
579
580
581
582 def nodes2Dofs(nodes, dofs):
583     """
584     # Relation dof -> node is
585     # dof = (node - 1) * dofs + a
586     # where a = [1, dofs]
587     #
588     # Example:
589     # nodes = [1, 5, 10] in elements with 6 dofs
590     # => returnDofs = [1, 2, 3, 4, 5, 6, 25, 26, 27, 28, 29, 30, 55, 56, 57, 58, 59, 60]
591     #
592     # Example 2:
593     # nodes = [1, 3]
594     # dofs = 2
595     # => returnDofs = [1, 2, 5, 6]
596     """
597     returnDofs = []
598     for node in nodes:
599         partDofs = range((node - 1) * dofs + 1, (node - 1) * dofs + (dofs + 1))
600         returnDofs += partDofs
601     return returnDofs
602
603
604 def dofs2Eqs(dofs, dict):
605     """
606     Returns a list with the corresponding equation for a node.
607
608     # Example:
609     # dofs = [1, 2, 3]
610     # dict = {
611         1: 2,
612         2: 3,
613         3: 1
614     }
615     returns [2, 3, 1]
616
617     :param dofs: List of dofs. 1-indexed. Length n
618     :param dict: Dictionary coupling dof to equation like
619         dict[dof] = eq
620     :param return: List of equations for the specified dofs. Length n. 1-indexed.
621     """
622     eqs = []
623     for dof in dofs:
624         eq = dict[dof]
625         eqs.append(eq)

```

```

626     return eqs
627
628
629
630
631 def partitionMatrix (mat, eNodes, iNodes, dict, dofs, sorting="equation"):
632     '''
633     Creates a partitioned matrix on the form
634     M =
635     [[Mee, Mei],
636     [Mii, Mii]]
637
638     :param mat: The Math.NET matrix to partition
639     :param eNodes: list [int] of the selected external nodes. Indexing starting at 1.
640     Sorted.
641     :param iNodes: list [int] of internal nodes. 1-indexed. Sorted.
642     :param dict: Dictionary with the mapping of DOF to equation number in the matrix.
643     Example: dict[dof] = eq
644     NOTE: dof and equation in the dictionary is 1-indexed!
645     :param DOFs: Number of degrees of freedom per node.
646     :param sorting: How to sort the partitioned matrix.
647     sorting="equation" means the partitioned matrix is sorted after ascending equation
648     numbers. This is what Fedem uses.
649     This is the default selection.
650     sorting="dof" means the partitioned matrix is sorted after ascending degree of
651     freedom numbers.
652
653     :return partitioned matrix
654     '''
655
656     # Getting the dof numbering for the external (e) and internal (i) nodes
657     # Relation dof -> node is
658     # dof = (node - 1) * dofs + a
659     # where a = [1, dofs]
660     # Example:
661     # eNodes = [1, 5, 10] in elements with 6 dofs
662     # => eDofs = [1, 2, 3, 4, 5, 6, 31, 32, 33, 34, 35, 36, 61, 62, 63, 64, 65, 65]
663     eDofs = nodes2Dofs(eNodes, dofs)
664     iDofs = nodes2Dofs(iNodes, dofs)
665
666     # Mapping to solver ordering:
667     eEqs = dofs2Eqs(eDofs, dict)           # equations corresponding to external
668     nodes. 1-indexed.
669     iEqs = dofs2Eqs(iDofs, dict)           # equations corresponding to internal
670     nodes. 1-indexed.
671
672     if (sorting=="equation"):                #eEqs and iEqs are already sorted by dof number.
673         eEqs = sorted(eEqs)
674         iEqs = sorted(iEqs)
675
676     # Initializing partitioned matrix

```

```

671 nRows = (len(eNodes) + len(iNodes)) * dofs
672 nCols = nRows
673 matPart = la.Double.SparseMatrix(nRows, nCols) # Initializing with zeroes
674
675 # Moving rows
676 row = 0
677 for e in eEqs:
678     matPart.SetRow(row, mat.Row(e-1))
679     row+=1
680 for i in iEqs:
681     matPart.SetRow(row, mat.Row(i-1))
682     row+=1
683 matCopy = la.Double.SparseMatrix(nRows, nCols) # Initializing with zeroes
684 matPart.CopyTo(matCopy) # Using a copy because values are accessed by reference.
685 # Moving columns
686 col = 0
687 for e in eEqs:
688     matPart.SetColumn(col, matCopy.Column(e-1))
689     col+=1
690 for i in iEqs:
691     matPart.SetColumn(col, matCopy.Column(i-1))
692     col+=1
693 return matPart
694
695
696
697
698
699 def readFedemMatrix(filename):
700     """
701     Reading a Fedem matrix to MathNETmatrix for easier comparison with ANSYS matrices.
702     Written to import the full mass and stiffness matrix from Fedem
703
704     :param filename: Path to matrix.
705     :return a sparse Math.NET matrix with contents of the matrix in path
706     """
707     file = open(filename, "r") # Reading data
708     line = file.readline()
709     line = line.split()
710     nRows = int(line[0]) # Number of columns and rows are stored in the
711     # first line after the comments
712     nCols = int(line[1])
713     mat = la.Double.SparseMatrix(nRows, nCols) # Initializing with zeros
714
715     row = 0
716     col = 0
717     content = file.readlines()
718     for line in range(0, len(content)):
719         splitted = content[line].split()
720         if (len(splitted) != 0):

```

```

720         if (content [ line ]. split () [0]. startswith ("+++")):
721             row = int(content [ line ]. split () [2]) - 1
722             colLine = content [ line + 1]. split ()
723             valLine = content [ line + 2]. split ()
724             col = int(colLine [0]) - 1
725             val = float (valLine [0])
726             mat[row, col] = val
727     file . close ()
728     return mat
729
730 def readFedemVector(path):
731     """
732     Reads vector from fedem_reducer.res file for comparison with vectors from ANSYS
733     Reads gravity vectors , as well as BMAT.res and PHI.res
734
735     :param path: Path to Fedem vector to read.
736     :return Sparse math.NET matrix of the Fedem vector in path
737     """
738     file = open(path, "r")
739     lines = file . readlines ()
740     firstLine = lines [0]. split ()
741     nRows = int( firstLine [0])
742     nCols = int( firstLine [1])
743     mat = la . Double.SparseMatrix(nRows, nCols) # Initializing with zeros
744
745     colIndexes = []
746     nextNumCols = 0
747     currNumCols = 0
748     row = 0
749     col = 0
750     startCol = 0
751     for i in range(0, len( lines )):
752         col = startCol
753         line = lines [i] . split ()
754         if (i != (len( lines ) - 1)): # if not at end of file
755             nextNumCols = len( lines [i + 1]. split ())
756             currNumCols = len( line )
757         if (currNumCols > nextNumCols): # if new columns
758             startCol = int( lines [i + 1]. split () [0]) - 1
759         continue
760         row = int( line [0]) - 1
761         vals = list (map(float, line )) [1:]
762         for val in vals :
763             mat[row, col] = val
764             col += 1
765     file . close ()
766     return mat
767
768
769 def readMMFMatrix(filename, format):

```

```

770     '''
771     Reading a dense or sparse Matrix Market file format into a Math.NET array.
772     NOTE:
773     - The matrices are saved column first !
774     - Matrices created from the .SUB binary ANSYS file are saved with the full (n x n)
       matrix .
775         This is the case for M_red and K_red files
776     - Matrices created from the .FULL binary ANSYS file have only the lower triangular
       part of the matrix saved.
777
778     :param filename: String of file name to the .mmf file . Dense .mmf matrices are
       saved column-first!
779     :param format: Format of the MMF matrix. "sub" for the reduced matrices . "full" for
       the full matrices .
780     :return Dense Math.NET matrix read from filename . Saved row-first .
781     '''
782     isFirstLine = False
783     nRows = 0
784     nCols = 0
785     file = open(filename, "r")      # Reading data
786     #
787     # Reading the lines defining dimensions of the matrix
788     #
789     dimLine = None
790     while ( isFirstLine == False):
791         line = file . readline ()
792         if (not line . startswith ('%')):
793             isFirstLine = True
794             line = line . split ()
795             nRows = int( line [0])      # Number of columns and rows are stored in
       the first line after the comments
796             nCols = int( line [1])
797             dimLine = line             # Saving for use in the next for loop
798
799     # Initializing Math.NET matrix
800     mat = la . Double.SparseMatrix(nRows, nCols)
801     isDenseFull = False
802     isSparseFull = False
803     isSparseSub = False
804
805     if (len(dimLine) == 2 and format=="full"):
806         isDenseFull = True
807     if (len(dimLine) == 3 and format=="full"):
808         isSparseFull = True
809     if (len(dimLine) == 3 and format=="sub"):
810         isSparseSub = True
811
812
813     # Filling Math.NET matrix with values from the file
814     row = 0

```

```

815     col = 0
816     for line in file :
817         if (isDenseFull):           # If we are reading a dense MMF matrix from ANSYS
            SUB or full file
818             val = float ( line )
819             mat[row, col] = val
820
821             if (row != nRows - 1):  # Updating counting variables
822                 row += 1
823             else :
824                 col += 1
825                 row = 0
826         if (isSparseSub or isSparseFull): # We are reading a sparse MMF matrix
            from ANSYS FULL file
827             line = line . split ()
828             i = int ( line [0] ) - 1      # Reading indexes . Is 1-indexed in the
            MMF file
829             j = int ( line [1] ) - 1
830             val = float ( line [2])
831             mat[i , j] = val
832             mat[j , i] = val            # sparse MMF matrix is saved in lower
            triangular form
833         file . close ()
834     return mat
835
836
837
838
839 def getRBE2Nodes(load):
840     """
841     Imports RBE2 nodes from NASTRAN file and returns list node ids of RBE2 nodes.
842     "CompID" is the string from the Workbench site, when pressing on external model,
            under "General" tab. Typically "Setup X"
843     "Identifier" is the string under identifier tab when double-clicking the external
            model. Typically "File1".
844     :param load: The current load object
845
846     :return List[int] Sorted list with integer nodal ids of found RBE2 nodes in
            NASTRAN file.
847     """
848     compID =
            load.Properties ["Generation"]. Properties ["Geometry"]. Properties ["CompID"]. Value
849     identifier =
            load.Properties ["Generation"]. Properties ["Geometry"]. Properties ["Identifier"]. Value
850     commands = ExtAPI.DataModel.Project.Model.GetFECommandsRepository(compID,
            identifier)
851     rbe2Comms = commands.GetCommandsByName("RBE2")
852     rbe2Count = rbe2Comms.Count
853     nodeList = []
854     for i in range (0, rbe2Count):

```

```

855     cmd = rbe2Comms[i+1]
856     nodeList.append(int(cmd.GetArgument[2])) # Node ID is on argument index
      two. Convert to int .
857     return sorted(nodeList)
858
859
860 class Element():
861     """
862     Class used for storing info on elements in a mesh
863     """
864
865     def __init__( self , id , nodes):
866         self ._nodes = nodes # list [Node] connected to the element
867         self ._id = id # Element id
868
869
870     def getNodes( self ):
871         return self ._nodes
872
873     def getId( self ):
874         return self ._id
875
876
877
878
879 class Node:
880     """
881     Data storage class for one node from a mesh
882     """
883
884     def __init__( self , id , dofs , elements , x , y , z):
885         """
886         :param id: Int of nodal id
887         :param dofs: Int of number of dofs for the node
888         :param elements: List [ int ] of element ids to the elements connected to the node
889         :param x: x coordinate
890         :param y: y coordinate
891         :param z: z coordinate
892         """
893         self ._id = id
894         self ._dofs = dofs
895         self ._elements = elements
896         self ._x = x
897         self ._y = y
898         self ._z = z
899
900
901     def getId ( ):
902         return self ._id
903

```

```

904     def getDofs():
905         return self._dofs
906
907     def getElements():
908         return self._elements
909
910
911
912 class SolverData:
913     #
914     # Class used to store data from the solver and ModRed application.
915     #
916
917     def __init__(self, analysis, load):
918         #
919         # Collecting user input
920         #
921         self._dofs = 6          # Currently hard-typing in number of dofs
922         self._filename =
load.Properties["Method"].Properties["ReductionMethod"].Properties["Name"].Value
# string
923         self._workingDir = analysis.WorkingDir
924         self._fedemDir =
load.Properties["Export"].Properties["IsExport"].Properties["FileDir"].Value
925         self._testResourcesDir = ExtAPI.ExtensionManager.CurrentExtension.InstallDir +
"\test-resources"
926         self._eNodes = getENodes(load)          # List[int] of selected external nodes.
Sorted
927         self._cModes =
int(load.Properties["Method"].Properties["ReductionMethod"].Properties["Nmode"].Value)
# Number of component modes
928
929         self._elements = self.getElementsFromMesh(analysis.MeshData)
#List[Element]
930         self._nodes = self.getNodesFromElements(self._elements)    # list [Node]
931
932         self._nRows = len(self._nodes) * self._dofs    # Number of rows in the full
mass matrix
933         self._nCols = self._nRows    # Quadratic matrix
934
935
936
937     def saveObject(self, obj, fname):
938         """
939         Saves the current object to _fedemDir
940         """
941         with open(self._testResourcesDir + "\\ " + fname + ".p", "wb") as file :
942             pickle.dump(obj, file , pickle.HIGHEST_PROTOCOL)
943
944     def getNodes(self):

```

```

945     """
946     Return sort list [Node] of nodes in the model. Sorted ascending by id
947     """
948     return sorted( self ._nodes, key=lambda x: x._id)
949
950
951 def getElements( self ):
952     """
953     Return sorted list [Element] of elements in the model. Sorted ascending by id
954     """
955     return sorted( self ._elements, key=lambda x: x._id)
956
957
958 def getNodesFromElements(self, elements):
959     """
960     Creates list [Node] of nodes connected to the element.
961
962     :param elements: List [Element]
963     :return List [Node] with all nodes in the model
964     """
965     nodes = []
966     for e in elements:
967         for n in e.getNodes():
968             if not any(x._id == n._id for x in nodes):
969                 nodes.append(n)
970     return nodes
971
972 def getNodeIds( self ):
973     """
974     :return Sorted list [int] of all node ids in the model
975     """
976     ids = []
977     for node in self ._nodes:
978         ids.append(node._id)
979     return sorted(ids)
980
981
982 def getElementsFromMesh(self, mesh):
983     """
984     Returns list of custom Element objects containing all elements in the model.
985
986     :param mesh: ANSYS MeshData object
987     :return List [Element]
988     """
989     returnElements = []
990     ansysElements = mesh.Elements
991     for e in ansysElements:
992         id = e.Id
993
994         nodeList = []

```

```

995         for node in e.Nodes:
996             nodeId = node.Id
997             elements = node.ConnectedElementIds           #List[int] of ids of
the connected elements
998             node = Node(nodeId, 6, elements, node.X, node.Y, node.Z)           #
Currently hard-typing in dofs per node. Should be changed to be read from ANSYS
API
999             nodeList.append(node)
1000
1001             element = Element(id, nodeList)
1002             returnElements.append(element)
1003         return returnElements
1004
1005
1006
1007 class Results :
1008     """
1009     Class for calculating results from the model reduction extension
1010     """
1011     def __init__( self ):
1012         self._resourcesFolder = os.path.dirname(os.path.realpath( __file__ )) +
"""test-resources"""
1013
1014
1015
1016     def quadraticSumDifference( self , mat, refMat ):
1017         """
1018         Quadratic sum of the difference of all elements in n x n matrix
1019
1020         :param mat: Matrix that will be compared
1021         :param refMat: Matrix used as reference for comparison
1022         :return percentage difference on quadratic sum of mat, compared to refMat
1023         """
1024         matSum = self.matrixSum(mat)
1025         refMatSum = self.matrixSum(refMat)
1026         squared = (matSum - refMatSum) * (matSum - refMatSum)
1027         diff = math.sqrt(squared)
1028         percentage = diff / refMatSum
1029         return percentage
1030
1031
1032     def quadraticSumDifferenceOfVector( self , vec, refVec ):
1033         """
1034         Quadratic sum of the difference of two vectors of size (n x 1)
1035
1036         :param : Vector that will be compared
1037         :param refVec: Vector used as reference for comparison
1038         :return percentage difference on quadratic sum of vec, compared to refVec
1039         """
1040         vecSum = vec.Sum()

```

```

1041     refVecSum = refVec.Sum()
1042     squared = (vecSum - refVecSum) * (vecSum - refVecSum)
1043     diff = math.sqrt(squared)
1044     percentage = diff / refVecSum
1045     return percentage
1046
1047
1048 def matrixSum(self, mat):
1049     """
1050     Calculating the matrix sum of a mat.NET matrix
1051     :param mat: The matrix to calculate the sum of
1052     :return float Sum of all elements in the matrix
1053     """
1054     rowSums = mat.RowSums()
1055     return sum(rowSums)
1056
1057
1058 def calculateMassFromMassMatrix(self, mat, reduced=True, cmodes=2):
1059     """
1060     Calculates the mass of a mass matrix mat by applying unit translation :
1061     m_tot = u^T * mat * u
1062
1063     :param mat: The mass matrix
1064     :param reduced: Indicating if it is a reduced mass matrix. Defaults to True.
1065     :param cmodes: Number of component modes in the reduced mass matrix. Defaults
1066     to 2
1067     """
1068     if (reduced==True):
1069         nRows = mat.RowCount
1070         nNodes = int((nRows - cmodes) / 6)          # Currently assuming 6 DOFs per
1071         node
1072         u_x = ModRed.createUnitVector(1, nNodes, cmodes)
1073         u_y = ModRed.createUnitVector(2, nNodes, cmodes)
1074         u_z = ModRed.createUnitVector(3, nNodes, cmodes)
1075         mass_x = u_x.Transpose().Multiply(mat).Multiply(u_x)[0,0]
1076         mass_y = u_y.Transpose().Multiply(mat).Multiply(u_y)[0,0]
1077         mass_z = u_z.Transpose().Multiply(mat).Multiply(u_z)[0,0]
1078         return [mass_x, mass_y, mass_z]
1079     else :
1080         nRows = mat.RowCount
1081         nNodes = int(nRows/6)          # currently assuming 6 dofs per node
1082         u_x = ModRed.createUnitVector(1, nNodes)
1083         u_y = ModRed.createUnitVector(2, nNodes)
1084         u_z = ModRed.createUnitVector(3, nNodes)
1085         mass_x = u_x.Transpose().Multiply(mat).Multiply(u_x)[0,0]
1086         mass_y = u_y.Transpose().Multiply(mat).Multiply(u_y)[0,0]
1087         mass_z = u_z.Transpose().Multiply(mat).Multiply(u_z)[0,0]
1088         return [mass_x, mass_y, mass_z]

```

A.2 ModRed.xml

ModRed.xml

```
1 <extension version="1" name="ModRed">
2   <!-- This is the GUID for the ModRed extension. It must remain the same independently
   ↪ of
3   the extension version or name. It is used to uniquely identify this extension. The
   ↪ shortId
4   attribute is needed for compatibility with old projects, it is the name of the
   ↪ extension
5   before adding this GUID. -->
6   <guid shortid="modred">D0B4EDE5-4151-4EC1-96B4-BDA4410E0CDE</guid>
7   <script src="ModRed.py" compiled="true"/>
8   <interface context="Mechanical">
9     <images>images</images>
10    <toolbar name="modred" caption="Fedem Reduction">
11      <entry name="Fedem Reduction" icon="fedem-transp">
12        <callbacks>
13          <onclick>onCreateModRed</onclick>
14        </callbacks>
15      </entry>
16    <separator />
17    <entry name="Export" caption="Generate Fedem Data" icon="fedem-export">
18      <callbacks>
19        <onclick>onExportData</onclick>
20      </callbacks>
21    </entry >
22  </toolbar>
23 </interface >
24
25  <simdata context="Mechanical">
26    <!-- defining the object that is inserted under "Model" in the project
   ↪ tree -->
27    <load name="modred" version="1" caption="Fedem Reduction"
   ↪ icon="fedem-transp"
28          isload="true" color="#0000FF" contextual="true">
29      <callbacks>
30        <!-- Allowing the user to right-click Fedem icon in the tree and select "Export
   ↪ Fedem Data" -->
31        <action name="onExportData" caption="Export Fedem Data"
   ↪ icon="fedem-export">onExportData</action>
32        <getsolvecommands>onSolve</getsolvecommands><!--
   ↪ Running the specified function when the user hits "solve" -->
33      </callbacks >
34
35    <propertygroup name="Generation"
36      caption="Selection of Master Nodes"
37      display="caption">
```

```

38         <propertygroup name="Geometry" caption="Pick Master
↪ Nodes By..." control="select" display="Property" default="Geometry">
39             < attributes options="Geometry,RBE2 Import" />
40             <property name="Geometry" control="scoping"
41                 caption="Geometry Selection"
↪ visibleon="Geometry">
42                 < attributes selection_filter ="node" />
43                 </property>
44                 <property name="CompID" control="String"
↪ caption="Component ID" visibleon="RBE2 Import" />
45                 <property name="Identifier" control="String"
↪ caption=" Identifier " visibleon="RBE2 Import" />
46                 </propertygroup>
47             </propertygroup>
48
49         <propertygroup name="Method" caption="Model Reduction Method"
50             display="Caption">
51             <propertygroup name="ReductionMethod"
↪ caption="Reduction Method" control="select" display="Property" default="CMS">
52                 < attributes options="Guyan,CMS" />
53                 <property name="Nmode" caption="Num. of
↪ Component Modes to Extract"
54                     control="int" default="2"
↪ visibleon="CMS">
55                     <callbacks>
56
↪ <isvalid>isValidNoModes</isvalid>
57                     </callbacks>
58                 </property>
59                 <property name="Name" caption="File Name"
↪ control="string"
60                     default="master"
↪ visibleon="Guyan|CMS" />
61                 <property name="LumpedMatrix" caption="Use lumped matrices?" control="select"
↪ default="No" >
62                     < attributes options="Yes,No" />
63                 </property>
64             </propertygroup>
65         </propertygroup>
66
67         <propertygroup name="Export"
68             caption="Export Options"
69             display="Caption">
70             <propertygroup name="IsExport" control="select"
↪ caption="Export Matrices to Fedem?" display="Property"
71                 default="Yes" >
72                 < attributes options="No,Yes" />
73                 <property name="FileDir" caption="Fedem File
↪ Directory"
74                     control="folderopen"

```

```

75                                     default=""
76                                     visibleon="Yes">
77                                     </property>
78                                 </propertygroup>
79                             </propertygroup>
80                         </load>
81                     </simdata>
82 </extension>

```

A.3 test_ModRed.py

test_ModRed.py

```

1
2
3 #
4 # Unit tests for ModRed
5 #
6
7 import clr
8 clr.AddReferenceToFileAndPath(r"C:\Program Files\ANSYS
    Inc\v192\Addins\ACT\bin\Win64\MathNet.Numerics.dll") # Using the Math.NET dll.
    Can not use NumPy, as this is not supported by IronPython.
9 import MathNet.Numerics.LinearAlgebra as la
10 import System
11 import unittest
12 import os
13 import pickle
14 from System import Array as sys_array
15
16 import sys
17 sys.path.insert(0,
    r"C:\Users\adrian\Documents\Dokumenter\NTNU\Master\masters\ModRed\ModRed')
18 import ModRed
19 from ModRed import Node
20 from ModRed import Element
21 from ModRed import SolverData
22
23
24
25 class TestModRed(unittest.TestCase):
26
27     def setUp(self):
28         self._resourcesFolder = os.path.dirname(os.path.realpath( __file__ )) +
    "\\test-resources\\"
29
30 #

```

```

31     # Defining the twoQUAD4 model
32     #
33     self ._node1 = Node(1, 6, [1], 0.0, 0.0, 0.0)
34     self ._node2 = Node(2, 6, [1, 2], 1.0, 0.0, 0.0)
35     self ._node3 = Node(3, 6, [2], 2.0, 0.0, 0.0)
36     self ._node4 = Node(4, 6, [1], 0.0, -1.0, 0.0)
37     self ._node5 = Node(5, 6, [1, 2], 1.0, -1.0, 0.0)
38     self ._node6 = Node(6, 6, [2], 2.0, -1.0, 0.0)
39
40     self ._element1 = Element(1, [ self ._node1, self ._node2, self ._node5,
self ._node4]) # clockwise
41     self ._element2 = Element(2, [ self ._node2, self ._node3, self ._node6,
self ._node5])
42
43     self ._elements = [ self ._element1, self ._element2]
44     self ._nodes = [ self ._node1, self ._node2, self ._node3, self ._node4, self ._node5,
self ._node6]
45     self ._unsortedNodes = [ self ._node2, self ._node1, self ._node3, self ._node6,
self ._node5, self ._node4]
46
47
48
49
50
51     def array(*x): return sys_array [ float ](x) # Helper function to create custom
Math.NET matrices
52     self ._mat_ = la .Double.Matrix.Build.DenseOfRowArrays(
53         array (1, 2, 3, 4, 5, 6, 7, 8, 9),
54         array (10, 11, 12, 13, 14, 15, 16, 17, 18),
55         array (19, 20, 21, 22, 23, 24, 25, 26, 27),
56         array (28, 29, 30, 31, 32, 33, 34, 35, 36),
57         array (37, 38, 39, 40, 41, 42, 43, 44, 45),
58         array (46, 47, 48, 49, 50, 51, 52, 53, 54),
59         array (55, 56, 57, 58, 59, 60, 61, 62, 63),
60         array (64, 65, 66, 67, 68, 69, 70, 71, 72),
61         array (73, 74, 75, 76, 77, 78, 79, 80, 81)
62     )
63     self ._indexMat_ = la .Double.Matrix.Build.DenseOfRowArrays(
64         array (11, 12, 13, 14, 15, 16, 17, 18, 19),
65         array (21, 22, 23, 24, 25, 26, 27, 28, 29),
66         array (31, 32, 33, 34, 35, 36, 37, 38, 39),
67         array (41, 42, 43, 44, 45, 46, 47, 48, 49),
68         array (51, 52, 53, 54, 55, 56, 57, 58, 59),
69         array (61, 62, 63, 64, 65, 66, 67, 68, 69),
70         array (71, 72, 73, 74, 75, 76, 77, 78, 79),
71         array (81, 82, 83, 84, 85, 86, 87, 88, 89),
72         array (91, 92, 93, 94, 95, 96, 97, 98, 99)
73     )
74
75     self ._indexMatSmall_ = la .Double.Matrix.Build.DenseOfRowArrays(

```

```

76         array (11, 12, 13),
77         array (21, 22, 23),
78         array (31, 32, 33)
79     )
80
81     self ._indexMat5x5_ = la .Double.Matrix.Build.DenseOfRowArrays(
82         array (11, 12, 13, 14, 15),
83         array (21, 22, 23, 24, 25),
84         array (31, 32, 33, 34, 35),
85         array (41, 42, 43, 44, 45),
86         array (51, 52, 53, 54, 55)
87     )
88
89     def tearDown(self):
90         if os.path.exists ( self ._resourcesFolder + "\\testFTL.ftl "):
91             os.remove(self ._resourcesFolder + "\\testFTL.ftl ")
92
93
94
95     def saveObject( self , obj , fname):
96         """
97         Saves the current object to _fedemDir
98         """
99         with open(self ._resourcesFolder + "\\ " + fname + ".p", "wb") as input:
100             pickle.dump(obj, input , pickle.HIGHEST_PROTOCOL)
101
102
103
104
105     def test_generateFTL ( self ):
106         file = open(self ._resourcesFolder + "\\testFTL.ftl ", "w")
107         eNodes = [1, 3]
108         ModRed.generateFTL(self._nodes, self._elements, eNodes, file )
109
110         # Need to open the closed file again
111         with open(self ._resourcesFolder + "\\testFTL.ftl ", "r") as file:
112             fileLines = file .readlines ()
113             file .close ()
114         with open(self ._resourcesFolder + "\\demoFTL.ftl", "r") as file:
115             correctFileLines = file .readlines ()
116             file .close ()
117         print ( fileLines )
118         print ( correctFileLines )
119         self .assertEqual ( len( fileLines ) , len( correctFileLines )) # Should be the
same length
120         self .assertEqual ( fileLines [3: ] , correctFileLines [3: ]) # Content is equal
(except for the "generated at ..." part
121
122
123

```

```

124     def test_getNodeString ( self ):
125         node = self ._node1
126         eNodes = [1, 3]
127
128         string = ModRed.getNodeString(node, eNodes)
129         correctString = "NODE{1 1 0.0 0.0 0.0}"
130         self . assertEquals ( string , correctString )
131
132
133
134     def test_getNodeString_notENode( self ):
135         """
136         If the node id is not in eNodes, it should be marked with 0
137         """
138         node = self ._node2
139         eNodes = [1, 3]
140
141         string = ModRed.getNodeString(node, eNodes)
142         correctString = "NODE{2 0 1.0 0.0 0.0}"
143         self . assertEquals ( string , correctString )
144
145     def test_getElementString ( self ):
146         element = self ._element1
147         thicknessId = 1
148         matId = 1
149         string = ModRed.getElementString(element, thicknessId , matId)
150         correctString = "QUAD4{1 1 2 5 4 {PTHICK 1} {PMAT 1}}"
151
152         self . assertEquals ( string , correctString )
153
154     def test_getElementString_element2 ( self ):
155         element = self ._element2
156         thicknessId = 1
157         matId = 1
158         string = ModRed.getElementString(element, thicknessId , matId)
159         correctString = "QUAD4{2 2 3 6 5 {PTHICK 1} {PMAT 1}}"
160
161         self . assertEquals ( string , correctString )
162
163
164
165     def test_createUnitVector ( self ):
166         """
167         Should return matrix of size ((nnodes*6)+cmodes x 1)
168         """
169
170         nnodes = 10
171         cmodes = 2
172
173         u = ModRed.createUnitVector(1, nnodes, cmodes)

```

```

174     self . assertEquals ( u [0,0], 1)
175     self . assertEquals ( u [1,0], 0)
176     self . assertEquals ( u [6,0], 1)
177     self . assertEquals ( u . ColumnCount, 1)
178     self . assertEquals ( u . RowCount, (nnodes*6) + cmodes)
179     # Checking if component mode positions are 0:
180     self . assertEquals ( u [nnodes*6, 0], 0)
181     self . assertEquals ( u [nnodes*6 + 1, 0], 0)
182
183     u = ModRed.createUnitVector(2, nnodes, cmodes)
184     self . assertEquals ( u [0,0], 0)
185     self . assertEquals ( u [1,0], 1)
186     self . assertEquals ( u [7,0], 1)
187     self . assertEquals ( u . ColumnCount, 1)
188     self . assertEquals ( u . RowCount, (nnodes*6) + cmodes)
189     # Checking if component mode positions are 0:
190     self . assertEquals ( u [nnodes*6, 0], 0)
191     self . assertEquals ( u [nnodes*6 + 1, 0], 0)
192
193 def test_extractPartitions ( self ):
194     def array(*x): return sys.array [ float ](x) # Helper function to create custom
Math.NET matrices
195
196     mat = self . _indexMat_
197     nENodes = 1
198     dofs = 3
199     parts = ModRed.extractPartitions ( mat, nENodes, dofs)
200     parts_ee = parts [0]
201     parts_ei = parts [1]
202     parts_ie = parts [2]
203     parts_ii = parts [3]
204
205     corr_ee = la . Double . Matrix . Build . DenseOfRowArrays(
206         array (11, 12, 13),
207         array (21, 22, 23),
208         array (31, 32, 33)
209     )
210     self . assertEquals ( parts_ee , corr_ee )
211
212     corr_ei = la . Double . Matrix . Build . DenseOfRowArrays(
213         array (14, 15, 16, 17, 18, 19),
214         array (24, 25, 26, 27, 28, 29),
215         array (34, 35, 36, 37, 38, 39)
216     )
217     self . assertEquals ( parts_ei , corr_ei )
218
219     corr_ie = la . Double . Matrix . Build . DenseOfRowArrays(
220         array (41, 42, 43),
221         array (51, 52, 53),
222         array (61, 62, 63),

```

```

223         array (71, 72, 73),
224         array (81, 82, 83),
225         array (91, 92, 93),
226     )
227     self.assertEqual ( parts_ie , corr_ie )
228
229     corr_ii = la.Double.Matrix.Build.DenseOfRowArrays(
230         array (44, 45, 46, 47, 48, 49),
231         array (54, 55, 56, 57, 58, 59),
232         array (64, 65, 66, 67, 68, 69),
233         array (74, 75, 76, 77, 78, 79),
234         array (84, 85, 86, 87, 88, 89),
235         array (94, 95, 96, 97, 98, 99)
236     )
237     self.assertEqual ( parts_ii , corr_ii )
238
239     def test_extractPartitions_FedemMatrix ( self ):
240         with open( self . _resourcesFolder + "\\beam.nas" + "\\Fedem" +
241             "\\M_full_partitioned .p" ) as file:
242             M_full = pickle.load( file )
243             mat = ModRed.extractPartitions ( M_full , 4, 6)
244
245             M_ee = mat[0]
246             M_ei = mat[1]
247             M_ie = mat[2]
248             M_ii = mat[3]
249
250             self.assertEqual ( M_ee.RowCount, 24)
251             self.assertEqual ( M_ee.ColumnCount, 24)
252             self.assertEqual ( M_ee[0,0], 9.775000E-002)
253             self.assertEqual ( M_ee[0,1], 0.0)
254             self.assertEqual ( M_ee[1,0], 0.0)
255             self.assertEqual ( M_ee[23,0], 0.0)
256             self.assertEqual ( M_ee[0,23], 0.0)
257             self.assertAlmostEqual ( M_ee[23,23], 4.072916E-006)
258
259             self.assertEqual ( M_ei.RowCount, 24)
260             self.assertEqual ( M_ei.ColumnCount, 606)
261             self.assertEqual ( M_ei[0,0], M_full[0, 24])
262             self.assertEqual ( M_ei[0, 605], M_full[0, 629])
263             self.assertEqual ( M_ei[23, 0], M_full[23, 24])
264             self.assertEqual ( M_ei[23, 605], M_full[23, 629])
265
266             self.assertEqual ( M_ie.RowCount, 606)
267             self.assertEqual ( M_ie.ColumnCount, 24)
268             self.assertEqual ( M_ie[0,0], M_full[24, 0])
269             self.assertEqual ( M_ie[0, 23], M_full[24, 23])
270             self.assertEqual ( M_ie[605, 0], M_full[629, 0])
271             self.assertEqual ( M_ie[605, 23], M_full[629, 23])

```

```

272     self.assertEqual(M.ii.RowCount, 606)
273     self.assertEqual(M.ii.ColumnCount, 606)
274     self.assertEqual(M.ii[0,0], M.full[24, 24])
275     self.assertEqual(M.ii[0, 605], M.full[24, 629])
276     self.assertEqual(M.ii[605, 0], M.full[629, 24])
277     self.assertEqual(M.ii[605, 605], M.full[629, 629])
278
279
280
281 def test_massMatrix_correct_mass_beam ( self ):
282     """
283     Testing if the mass matrix is correct by applying
284     u_trans ^T * M * u_trans = m_tot
285     """
286     # Validating M_full from Fedem
287     M_full = ModRed.readFedemMatrix(self._resourcesFolder + "\\beam.nas" +
    "\\Fedem" + "\\M_full.res")
288     correct_mass = 3.12800E+01 # Taken from fedem_reducer.res . In kg
289     for i in [1, 2, 3]:
290         u = ModRed.createUnitVector(i, 105)
291         m = u.Transpose().Multiply(M_full).Multiply(u)[0,0]
292         self.assertEqual(correct_mass, m)
293
294     # Testing mass matrix from ANSYS
295     M_full = ModRed.readMMFMMatrix(self._resourcesFolder + "\\beam.nas" + "\\Ansys"
    + "\\M_full.mmf", "full")
296     for i in [1, 2, 3]:
297         u = ModRed.createUnitVector(i, 105)
298         m = u.Transpose().Multiply(M_full).Multiply(u)[0,0]
299         self.assertEqual(correct_mass, m)
300
301     # Testing the lumped mass matrix from ANSYS
302     M_full = ModRed.readMMFMMatrix(self._resourcesFolder + "\\beam.nas" + "\\Ansys"
    + "\\M_full_lumped.mmf", "full")
303     for i in [1, 2, 3]:
304         u = ModRed.createUnitVector(i, 105)
305         m = u.Transpose().Multiply(M_full).Multiply(u)[0,0]
306         print(m)
307         self.assertEqual(correct_mass, m)
308
309
310
311 def test_partitionMatrix_correct_mass_beam ( self ):
312     """
313     Testing if the mass is unchanged after partitionMatrix method
314     """
315     M_full = ModRed.readMMFMMatrix(self._resourcesFolder + "\\beam.nas" + "\\Ansys"
    + "\\M_full.mmf", "full")
316     eNodes = [1, 2, 3, 4]
317     iNodes = [i for i in range(5, 106)]

```

```

318     dict = ModRed.readMappingFile(self._resourcesFolder + "\\beam.nas" + "\\Ansys"
+ "\\M_full.mapping")
319     M_full = ModRed.partitionMatrix(M_full, eNodes, iNodes, dict, 6)
320
321     correct_mass = 3.12800E+01 # Taken from fedem_reducer.res. In kg
322     for i in [1, 2, 3]:
323         u = ModRed.createUnitVector(i, 105)
324         m = u.Transpose().Multiply(M_full).Multiply(u)[0,0]
325         print(m)
326         self.assertAlmostEqual(correct_mass, m)
327
328
329
330
331     def test_partitionMatrix_smallMatrices (self):
332         """
333         Testing the partitionMatrix method with small matrices
334         """
335         def array(*x): return sys_array [ float ](x) # Helper function to create custom
Math.NET matrices
336         dict = {
337             1: 2,
338             2: 3,
339             3: 1
340         }
341         # Remember: 3 DOFs per node
342         eNodes = [1]
343         iNodes = [2, 3]
344         partitioned = ModRed.partitionMatrix( self._indexMatSmall_, eNodes, iNodes,
dict, 1, sorting="dof")
345         correct = la.Double.Matrix.Build.DenseOfRowArrays(
346             array(22, 23, 21),
347             array(32, 33, 31),
348             array(12, 13, 11)
349         )
350         self.assertEqual( correct, partitioned )
351         for row in range(0, correct.RowCount):
352             for col in range(0, correct.ColumnCount):
353                 self.assertEqual( partitioned [row, col], correct [row, col] )
354
355
356
357     def test_partitionMatrix_5x5 (self):
358         #
359         # Testing of partitioning a custom 5x5 matrix
360         #
361         def array(*x): return sys_array [ float ](x) # Helper function to create custom
Math.NET matrices
362         dict = {
363             1: 3,

```

```

364         2: 4,
365         3: 5,
366         4: 1,
367         5: 2
368     }
369     eNodes = [2, 5]
370     iNodes = [1, 3, 4]
371     partitioned = ModRed.partitionMatrix( self ._indexMat5x5_, eNodes, iNodes, dict ,
1, sorting="equation")
372     print ( partitioned )
373
374
375 def test_partitionMatrix_9x9 ( self ):
376     #
377     # Testing with 9x9 matrix
378     #
379     def array(*x): return sys_array [ float ](x) # Helper function to create custom
Math.NET matrices
380     dict = {
381         1: 4,
382         2: 5,
383         3: 6,
384         4: 7,
385         5: 8,
386         6: 9,
387         7: 1,
388         8: 2,
389         9: 3
390     }
391     eNodes = [1]
392     iNodes = [2, 3]
393     partitioned = ModRed.partitionMatrix( self ._indexMat_, eNodes, iNodes, dict , 3,
sorting="dof")
394     """
395     tmp = la .Double.Matrix.Build.DenseOfRowArrays( # Moving rows
396         array (41, 42, 43, 44, 45, 46, 47, 48, 49),
397         array (51, 52, 53, 54, 55, 56, 57, 58, 59),
398         array (61, 62, 63, 64, 65, 66, 67, 68, 69),
399         array (71, 72, 73, 74, 75, 76, 77, 78, 79),
400         array (81, 82, 83, 84, 85, 86, 87, 88, 89),
401         array (91, 92, 93, 94, 95, 96, 97, 98, 99).
402         array (11, 12, 13, 14, 15, 16, 17, 18, 19),
403         array (21, 22, 23, 24, 25, 26, 27, 28, 29),
404         array (31, 32, 33, 34, 35, 36, 37, 38, 39)
405     )
406     """
407     correct = la .Double.Matrix.Build.DenseOfRowArrays( # Moving cols
408         array (44, 45, 46, 47, 48, 49, 41, 42, 43),
409         array (54, 55, 56, 57, 58, 59, 51, 52, 53),
410         array (64, 65, 66, 67, 68, 69, 61, 62, 63),

```

```

411         array (74, 75, 76, 77, 78, 79, 71, 72, 73),
412         array (84, 85, 86, 87, 88, 89, 81, 82, 83),
413         array (94, 95, 96, 97, 98, 99, 91, 92, 93),
414         array (14, 15, 16, 17, 18, 19, 11, 12, 13),
415         array (24, 25, 26, 27, 28, 29, 21, 22, 23),
416         array (34, 35, 36, 37, 38, 39, 31, 32, 33)
417     )
418     for row in range(0, correct.RowCount):
419         for col in range(0, correct.ColumnCount):
420             self.assertEqual( partitioned [row, col], correct [row, col])
421
422
423     def test_dofs2Eqs ( self ):
424         dofs = [1, 2, 3]
425         dict = {
426             1: 2,
427             2: 3,
428             3: 1
429         }
430         correct = [2, 3, 1]
431         eqs = ModRed.dofs2Eqs(dofs, dict)
432         self.assertEqual( eqs, correct )
433         self.assertEqual( len(dofs), len(eqs))
434
435
436         with open( self._resourcesFolder + "\\beam.nas" + "\\Ansys" +
437                 "\\M_full.mapping.p") as file:
438             dict = pickle.load( file )
439             dofs = range(1, 631)
440             eqs = ModRed.dofs2Eqs(dofs, dict)
441
442             self.assertEqual( len(dofs), len(eqs))
443             self.assertEqual( eqs [0], 625)          # dof 1 coupled to equation 624
444             self.assertEqual( eqs [1], 626)
445             self.assertEqual( eqs [282], 1)
446
447     def test_nodes2Dofs ( self ):
448         """
449         # Example:
450         # eNodes = [1, 5, 10] in elements with 6 dofs
451         # => eDofs = [1, 2, 3, 4, 5, 6, 25, 26, 27, 28, 29, 30, 55, 56, 57, 58, 59, 60]
452         """
453         eNodes = [1]
454         dofs = 6
455         correct = [1, 2, 3, 4, 5, 6]
456
457         eDofs = ModRed.nodes2Dofs(eNodes, dofs)
458         self.assertEqual( eDofs, correct )
459

```

```

460     eNodes = [1, 5, 10]
461     correct = [1, 2, 3, 4, 5, 6, 25, 26, 27, 28, 29, 30, 55, 56, 57, 58, 59, 60]
462     eDofs = ModRed.nodes2Dofs(eNodes, dofs)
463     self.assertEqual(eDofs, correct)
464
465
466     eNodes = [1, 3]
467     dofs = 2
468     correct = [1, 2, 5, 6]
469     eDofs = ModRed.nodes2Dofs(eNodes, dofs)
470     self.assertEqual(eDofs, correct)
471
472
473
474
475 def test_partitionMatrix_difference_twoQUAD4 ( self ):
476     """
477     Testing if Fedem and ANSYS matrices are partitioned correctly .
478     Using the partitioned Fedem matrix as reference , as this is proven to yield
479     the identical gravity vector as Fedem calculates .
480     """
481     dict_fedem = ModRed.readMappingFileFedem(self._resourcesFolder +
482     "\\twoQUAD4" + "\\Fedem" + "\\MEQN.res")
483     dict_ansys = ModRed.readMappingFile(self._resourcesFolder + "\\twoQUAD4" +
484     "\\Ansys" + "\\M_full_lumped.mapping")
485     M_full_fedem = ModRed.readFedemMatrix(self._resourcesFolder + "\\twoQUAD4" +
486     "\\Fedem" + "\\M_full.res")
487     M_full_ansys = ModRed.readMMFMatrix(self._resourcesFolder + "\\twoQUAD4" +
488     "\\Ansys" + "\\M_full_lumped.mmf", "full")
489
490     eNodes = [1, 3]
491     iNodes = [2, 4, 5, 6]
492     fedem = ModRed.partitionMatrix(M_full_fedem, eNodes, iNodes, dict_fedem, 6,
493     sorting="equation")
494     ansys = ModRed.partitionMatrix(M_full_ansys, eNodes, iNodes, dict_ansys, 6,
495     sorting="equation")
496
497     for row in range(0, ansys.RowCount):
498         for col in range(0, ansys.ColumnCount):
499             if row==col:
500                 print("r: " + str(row) + ", tc: " + str(col) + ",\tfedem: " +
501                 str(fedem[row, col]) + ",\t\tansys: " + str(ansys[row, col]) )
502                 self.assertEqual(fedem[row, col], ansys[row, col], places=1)
503                 self.fail("Not correct")
504
505
506
507 def test_massMatrix_diagonality ( self ):
508     #

```

```

503     # Verifying that a lumped matrix is diagonal
504     #
505     mat = ModRed.readMMFMatrix(self._resourcesFolder + "\\beam.nas" + "\\Ansys" +
    "\\M_full_lumped.mmf", "full")
506     for row in range(0, mat.RowCount):
507         for col in range(0, mat.ColumnCount):
508             if row == col:
509                 self.assertTrue(mat[row, col] != 0.0 )
510             else:
511                 self.assertTrue(mat[row, col] == 0.0)
512
513
514
515     def test_partitionMatrix_twoQUAD4_Ansys(self):
516         #
517         # Testing with mass matrix from twoQUAD4.
518         #
519         dict = ModRed.readMappingFileFedem(self._resourcesFolder + "\\twoQUAD4" +
    "\\Fedem" + "\\MEQN.res")
520         M_full = ModRed.readFedemMatrix(self._resourcesFolder + "\\twoQUAD4" +
    "\\Fedem" + "\\M_full.res")
521
522         eNodes = [1, 3]
523         iNodes = [2, 4, 5, 6]
524         mat = ModRed.partitionMatrix(M_full, eNodes, iNodes, dict, 6)
525         # Checking if diagonal:
526         for row in range(0, mat.RowCount):
527             for col in range(0, mat.ColumnCount):
528                 if row == col:
529                     self.assertTrue(mat[row, col] != 0.0 )
530                 else:
531                     self.assertTrue(mat[row, col] == 0.0)
532
533         # It seems that some of the precision is lost in Fedem when comparing M_full
    and M_ee from Fedem.
534         # Therefore the assertAlmostEqual
535         self.assertAlmostEqual(mat[0,0], 3.910000E+001)
536         self.assertAlmostEqual(mat[1,1], 3.910000E+001)
537         self.assertAlmostEqual(mat[2,2], 3.910000E+001)
538         self.assertAlmostEqual(mat[3,3], 2.782139E-001, places=6)
539         self.assertAlmostEqual(mat[5,5], 6.516665E-001, places=6)
540         self.assertAlmostEqual(mat[11,11], 6.516665E-001, places=6)
541
542
543     def test_partitionMatrix_twoQUAD4_Fedem(self):
544         #
545         # Testing with mass matrix + dictionary from ANSYS. Compared to M_ee from
    Fedem
546         #

```

```

547     dict = ModRed.readMappingFile(self._resourcesFolder + "\\twoQUAD4" +
    "\\Ansys" + "\\M_full.mapping")
548     M_full = ModRed.readMMFMatrix(self._resourcesFolder + "\\twoQUAD4" +
    "\\Ansys" + "\\M_full_lumped.mmf", "full")
549
550     eNodes = [1, 3]
551     iNodes = [2, 4, 5, 6]
552     mat = ModRed.partitionMatrix(M_full, eNodes, iNodes, dict, 6)
553     correct = ModRed.readFedemVector(self._resourcesFolder + "\\twoQUAD4" +
    "\\Fedem" + "\\M_ee.res")
554
555     # Checking if diagonal:
556     for row in range(0, mat.RowCount):
557         for col in range(0, mat.ColumnCount):
558             if row == col:
559                 self.assertTrue(mat[row, col] != 0.0 )
560             else:
561                 self.assertTrue(mat[row, col] == 0.0)
562
563     # It seems that some of the precision is lost in Fedem when comparing M_full
    and M_ee from Fedem.
564     # Therefore the assertAlmostEqual
565     self.assertAlmostEqual(mat [0,0], 3.910000E+001, places=2)
566     self.assertAlmostEqual(mat [1,1], 3.910000E+001, places=2)
567     self.assertAlmostEqual(mat [2,2], 3.910000E+001, places=2)
568     self.assertAlmostEqual(mat [3,3], 2.782139E-001, places=2)
569     self.assertAlmostEqual(mat [5,5], 6.516665E-001, places=2)
570     self.assertAlmostEqual(mat [9,9], 2.782139E-001, places=5)
571     self.assertAlmostEqual(mat [11,11], 6.516665E-001, places=2)
572
573
574 def test_partitionMatrix_Fedem_beam ( self ):
575     #
576     # Testing with mass matrix from beam.nas. Using the Fedem matrices,
577     # because here we know the resulting matrix
578     #
579     dict = ModRed.readMappingFileFedem(self._resourcesFolder + "\\beam.nas" +
    "\\Fedem" + "\\MEQN.res")
580     M_full = ModRed.readFedemMatrix(self._resourcesFolder + "\\beam.nas" +
    "\\Fedem" + "\\M_full.res")
581
582     eNodes = [1, 2, 3, 4]
583     iNodes = [i for i in range(5, 106)]
584     mat = ModRed.partitionMatrix(M_full, eNodes, iNodes, dict, 6)
585     # Checking if diagonal:
586     for row in range(0, mat.RowCount):
587         for col in range(0, mat.ColumnCount):
588             if row == col:
589                 self.assertTrue(mat[row, col] != 0.0 )
590             else:

```

```

591         self . assertTrue (mat[row, col] == 0.0)
592
593     # It seems that some of the precision is lost in Fedem when comparing M_full
and M_ee from Fedem.
594     # Therefore the assertAlmostEqual
595     self . assertAlmostEqual(mat [0,0], 9.775000E-002)
596     self . assertEquals (mat [1,0], 0.0)
597     self . assertEquals (mat [0,1], 0.0)
598     self . assertEquals (mat [1,1], 9.775000E-002)
599     self . assertAlmostEqual(mat [3,3], 1.982601E-006)
600     self . assertAlmostEqual(mat [4,4], 3.048196E-006)
601     self . assertAlmostEqual(mat [5,5], 4.07292E-06)
602
603     self . assertAlmostEqual(mat [11,11], 4.072916E-006)
604     self . assertAlmostEqual(mat [15, 15], 1.982601E-006)
605     self . assertAlmostEqual(mat [22, 22], 3.048196E-006)
606     self . assertAlmostEqual(mat [23, 23], 4.072916E-006)
607
608     self . assertEquals (mat [24, 24], 1.95500E-01)
609     self . assertEquals (mat [629, 629], 1.62917E-05)
610
611
612 def test_partitionMatrix_Fedem_unsrtENodes ( self ):
613
614     # beam.nas model from Fedem, with selected eNodes = [22, 28, 6, 44]
615
616     dict = ModRed.readMappingFileFedem(self._resourcesFolder + "\\beam.nas" +
"\\Fedem" + "\\unsrtENodes" + "\\MEQN.res")
617     M_full = ModRed.readFedemMatrix(self._resourcesFolder + "\\beam.nas" +
"\\Fedem" + "\\unsrtENodes" + "\\M_full.res") # NOTE: The full mass matrix is
not the same
618
619     eNodes = [6, 22, 28, 44]
620     iNodes = [i for i in range(1, 106)]
621     for e in eNodes:
622         iNodes.remove(e)
623
624     mat = ModRed.partitionMatrix(M_full, eNodes, iNodes, dict, 6)
625     # Checking if diagonal:
626     for row in range(0, mat.RowCount):
627         for col in range(0, mat.ColumnCount):
628             if row == col:
629                 self . assertTrue (mat[row, col] != 0.0 )
630             else:
631                 self . assertTrue (mat[row, col] == 0.0)
632
633     # Checking entries in M_ee:
634     self . assertAlmostEqual(mat [0,0], 1.955000E-001)
635     self . assertAlmostEqual(mat [3,3], 3.965202E-006)
636     self . assertAlmostEqual(mat [6,6], 1.955000E-001)

```

```

637         self.assertAlmostEqual(mat[16,16], 6.096392E-006)
638
639
640
641     def test_partitionMatrix_beam_Mee ( self ):
642         # Testing with M_full from beam.nas (medium model)
643         mapping = ModRed.readMappingFileFedem(self._resourcesFolder + "\\beam.nas" +
644         "\\Fedem" + "\\MEQN.res")
645         M_full = ModRed.readFedemMatrix(self._resourcesFolder + "\\beam.nas" +
646         "\\Fedem" + "\\M_full.res")
647         eNodes = [1, 2, 3, 4]
648         iNodes = [i for i in range(5, 106)]
649         M_full_partitioned = ModRed.partitionMatrix(M_full, eNodes, iNodes, mapping, 6)
650         # Using M_ee from Fedem as reference
651         M_ee = ModRed.readFedemVector(self._resourcesFolder + "\\beam.nas" +
652         "\\Fedem" + "\\M_ee.res")
653
654         for row in range(0, M_ee.RowCount):
655             for col in range(0, M_ee.ColumnCount):
656                 self.assertAlmostEqual( M_full_partitioned [row, col ], M_ee[row, col ])
657
658
659     def test_partitionMatrix_noDict ( self ):
660         """
661         Testing the partitionMatrix method with dictionary dof = equation
662         """
663         def array(*x): return sys.array [ float ](x) # Helper function to create custom
664         Math.NET matrices
665         # Dictionary coupling dof to equation:
666         # dict [dof] = eq
667         noDict = { # dof = equation
668             1: 1,
669             2: 2,
670             3: 3,
671             4: 4,
672             5: 5,
673             6: 6,
674             7: 7,
675             8: 8,
676             9: 9
677         }
678         # Remember: 3 DOFs per node
679         eNodes = [1]
680         iNodes = [2, 3]
681
682         partitioned = ModRed.partitionMatrix( self._mat_, eNodes, iNodes, noDict, 3,
683         sorting="dof")
684         # Dictionary with dof = equation and eNodes, iNodes in ascending order

```

```

682         # should return the same matrix
683         self.assertEqual( self._mat_, partitioned )
684
685         partitioned = ModRed.partitionMatrix( self._mat_, eNodes, iNodes, noDict, 3,
sorting="dof")
686         # Testing first if partitionMatrix works with no dictionary mapping
687
688         correct = la.Double.Matrix.Build.DenseOfRowArrays( # Moving rows
689             array(31, 32, 33, 28, 29, 30, 34, 35, 36),
690             array(40, 41, 42, 37, 38, 39, 43, 44, 45),
691             array(49, 50, 51, 46, 47, 48, 52, 53, 54),
692             array(4, 5, 6, 1, 2, 3, 7, 8, 9),
693             array(13, 14, 15, 10, 11, 12, 16, 17, 18),
694             array(22, 23, 24, 19, 20, 21, 25, 26, 27),
695             array(58, 59, 60, 55, 56, 57, 61, 62, 63),
696             array(67, 68, 69, 64, 65, 66, 70, 71, 72),
697             array(76, 77, 78, 73, 74, 75, 79, 80, 81)
698         )
699
700         eNodes = [2]
701         iNodes = [1, 3]
702         partitioned = ModRed.partitionMatrix( self._mat_, eNodes, iNodes, noDict, 3,
sorting="dof")
703         self.assertEqual( partitioned , correct )
704
705
706
707
708     def test_mathNET2list( self ):
709         list = ModRed.mathNET2list(self._mat_)
710         correct_list = [1.0, 10.0, 19.0, 28.0]
711         wrong_list = [1.0, 2.0, 3.0, 4.0] # Should be returned column-first
712
713         print("my list: " + str( list [0:4] ))
714         self.assertEqual( list [0:4], correct_list )
715         self.assertNotEqual( list [0:4], wrong_list )
716         self.assertEqual( len( list ), 81) # 9x9 matrix
717
718
719     def test_readMMFMatrix_red( self ):
720         M_red_path = self._resourcesFolder + "\\beam.nas" + "\\Ansys" + "\\M_red.mmf"
721         mat = ModRed.readMMFMatrix(M_red_path, "sub")
722
723         self.assertEqual( mat.RowCount, 26)
724         self.assertEqual( mat.ColumnCount, 26)
725
726         self.assertEqual( mat [0,0], 9.037402289784589E+00)
727         self.assertEqual( mat [0,2], 0)
728         self.assertEqual( mat [2,0], 0)
729

```

```

730     self . assertEquals (mat [0,5], 4.528581487170890E-05)
731     self . assertEquals (mat [5,0], 4.528581487170890E-05)
732
733     self . assertEquals (mat [25,25], 1.0)
734
735     M_red_path = self . _resourcesFolder + "\\plate.nas" + "\\Ansys" + "\\M_red.mmf"
736     mat = ModRed.readMMFMatrix(M_red_path, "sub")
737
738     self . assertEquals (mat.RowCount, 26)
739     self . assertEquals (mat.ColumnCount, 26)
740
741     self . assertEquals (mat [0,0], 5.871600155722680E-01)
742     self . assertEquals (mat [0,2], 4.381258188504480E-01)
743     self . assertEquals (mat[24, 20], 2.027417791055110E-17)
744
745
746
747
748
749
750 def test_readMMFMatrix_full( self ):
751     path = self . _resourcesFolder + "\\beam.nas" + "\\Ansys" + "\\M_full.mmf"
752     mat = ModRed.readMMFMatrix(path, "full")
753
754     # Checking if dimensions are correct
755     self . assertEquals (mat.RowCount, 630)
756     self . assertEquals (mat.ColumnCount, 630)
757
758     # Checking some values at the boundaries:
759     self . assertEquals (mat [0,0], 8.688888888888170E-02)
760     self . assertEquals (mat[0, 629], 0)
761     self . assertEquals (mat[629, 0], 0)
762     self . assertEquals (mat[629, 629], 1.448148148148030E-16)
763
764     # Testing random values:
765     self . assertEquals (mat [23,23], 1.448148148148030E-16)
766     self . assertEquals (mat [285,243], 3.620370370370130E-07)
767     self . assertEquals (mat [291,291], 5.792592592592110E-06)
768
769
770
771
772 def test_readMMFMatrix_full_isSymmetric( self ):
773     path = self . _resourcesFolder + "\\beam.nas" + "\\Ansys" + "\\M_full.mmf"
774     mat = ModRed.readMMFMatrix(path, "full")
775
776     # Checking if matrix is symmetric:
777     for row in range(0, mat.RowCount):
778         for col in range(0, mat.ColumnCount):
779             self . assertEquals (mat[row, col ], mat[col, row])

```

```

780
781
782 def test_readMMFMatrix_full_correctMass ( self ):
783     path = self . _resourcesFolder + "\\beam.nas" + "\\Ansys" + "\\M_full.mmf"
784     mat = ModRed.readMMFMatrix(path, "full")
785     correct_mass = 3.12800E+01      # Taken from Fedem_reducer
786
787     for i in [1, 2, 3]:
788         u = ModRed.createUnitVector(i, 105)
789         m = u.Transpose() . Multiply (mat) . Multiply (u) [0,0]
790         print (m)
791         self . assertAlmostEqual ( correct_mass , m)
792
793
794 def test_readMMFMatrix_red_isSymmetric(self):
795     path = self . _resourcesFolder + "\\beam.nas" + "\\Ansys" + "\\M_red.mmf"
796     mat = ModRed.readMMFMatrix(path, "sub")
797
798     # Checking if matrix is symmetric:
799     for row in range(0, mat.RowCount):
800         for col in range(0, mat.ColumnCount):
801             self . assertEquals (mat[row, col ], mat[col , row])
802
803
804 def test_readFedemVector_red_correctMass ( self ):
805     path = self . _resourcesFolder + "\\beam.nas" + "\\Fedem" + "\\M_red.res"
806     mat = ModRed.readFedemVector(path)
807     print (mat)
808     correct_mass = 3.12800E+01
809
810     for i in [1, 2, 3]:
811         u = ModRed.createUnitVector(i, 4, 2)
812         m = u.Transpose() . Multiply (mat) . Multiply (u) [0,0]
813         self . assertAlmostEqual ( correct_mass , m, places=5)
814
815
816 def test_readMMFMatrix_red_correctMass( self ):
817     """
818     Applying unit translation in x, y, z direction and verifying it is the same
819     mass
820     as computed in Fedem
821     """
822     path = self . _resourcesFolder + "\\beam.nas" + "\\Ansys" + "\\M_red.mmf"
823     mat = ModRed.readMMFMatrix(path, "sub")
824     correct_mass = 3.12800E+01
825
826     for i in [1, 2, 3]:
827         u = ModRed.createUnitVector(i, 4, 2)
828         m = u.Transpose() . Multiply (mat) . Multiply (u) [0,0]
829         self . assertAlmostEqual ( correct_mass , m, places=6)

```

```

829
830
831
832
833
834 """
835 # Commented out for being able to run all tests faster
836 def test_readMMFMatrix_largeFiles( self ):
837     M_full_path = self . _resourcesFolder + "\\lca.nas" + "\\M_full.mmf" # Large
      file (230 000 lines )
838     M_full = ModRed.readMMFMatrix(M_full_path, "full")
839     self . assertEquals ( M_full . RowCount, 35553)
840     self . assertEquals ( M_full . ColumnCount, 35553)
841     self . assertEquals ( M_full [0, 0], 6.837606621530309E-05) # start
842     self . assertEquals ( M_full [165, 12], 2.229360838501640E-05) # random place
843     self . assertEquals ( M_full [35552, 35552], 2.580374027760560E-07) # end
844
845
846 def test_readMMFMatrix_largeFiles2( self ):
847     M_full_path = self . _resourcesFolder + "\\reactor" + "\\M_full.mmf" # Large
      file (450 000 lines )
848     M_full = ModRed.readMMFMatrix(M_full_path, "full")
849     self . assertEquals ( M_full . RowCount, 69438)
850     self . assertEquals ( M_full . ColumnCount, 69438)
851     self . assertEquals ( M_full [0, 0], 9.966062445579789E-05) # start
852     self . assertEquals ( M_full [26236, 26236], 7.549508902862550E-04) # random place
853     self . assertEquals ( M_full [69437, 69437], 8.306723475513540E-04) # end
854 """
855
856 def test_readFedemMatrix( self ):
857     path = self . _resourcesFolder + "\\beam.nas" + "\\Fedem" + "\\M_full.res"
858     mat = ModRed.readFedemMatrix(path)
859     # Checking some entries
860     self . assertEquals (0.1955, mat [0,0])
861     self . assertEquals (mat [0,1], 0.000)
862     self . assertEquals (mat [1,1], 0.1955)
863     self . assertEquals (mat [202, 202], 6.09639E-06)
864     self . assertEquals (mat [605, 605], 1.62917E-05)
865     self . assertEquals (mat [606, 606], 9.77500E-02)
866     self . assertEquals (mat [629, 629], 4.07292E-06)
867
868 def test_readFedemVector_twoQUAD4(self):
869     path = self . _resourcesFolder + "\\twoQUAD4" + "\\Fedem" + "\\gravVec.res"
870     mat = ModRed.readFedemVector(path)
871
872     self . assertEquals (mat [0,0], 1.564000E+002)
873     self . assertEquals (mat [0,1], -3.556089E+001)
874     self . assertEquals (mat [13,2], -9.710526E-015)
875     self . saveObject (mat, "gravVec")
876

```

```

877
878 def test_readFedemVector( self ):
879     path = self . _resourcesFolder + "\\beam.nas" + "\\Fedem" + "\\Bmat.res"
880     mat = ModRed.readFedemVector(path)
881
882     self . assertEquals( mat [0,0], 2.270180E-001)
883     self . assertEquals( mat [0,1], 8.015493E-002)
884     self . assertEquals( mat [0,9], 0.0)
885     self . assertEquals( mat [0,17], -3.023801E-004)
886     self . assertEquals( mat [0,23], -7.732552E-003)
887     self . assertEquals( mat [601,0], 1.497216E-002)
888     self . assertEquals( mat [604,0], 0.0)
889     self . assertEquals( mat [605, 23], -2.384053E-002)
890     self . assertEquals( mat [605,23], -2.384053E-002)
891     self . assertEquals( mat [604,9], -2.839266E-004)
892     self . assertEquals( mat [605,9], 0.0)
893
894     self . assertEquals( mat [605,10], 4.054158E-003)
895     self . assertEquals( mat [605,11], -3.068604E-002)
896     self . assertEquals( mat [605,5], 4.054158E-003)
897     self . assertEquals( mat [605,1], 1.572045E-001)
898     self . assertEquals( mat [605,0], 5.364874E-001)
899
900     mat = ModRed.readFedemVector(self . _resourcesFolder + "\\beam.nas" + "\\Fedem"
+ "\\phi_fedem.res")
901     self . assertEquals( 0.0, mat [0,0])
902     self . assertEquals( 6.392561E-016, mat[0,1])
903     self . assertEquals( -8.829041E-002, mat[2,0])
904     self . assertEquals( 3.321254E-016, mat[605,0])
905     self . assertEquals( -5.843641E-014, mat[605,1])
906
907     vec = ModRed.readFedemVector(self . _resourcesFolder + "\\beam.nas" + "\\Fedem"
+ "\\gravVec.res")
908     self . assertEquals( vec [0,0], 7.820000E+000)
909     self . assertEquals( vec [0,1], 1.297520E+001)
910     self . assertEquals( vec [25,0], 1.826227E-014)
911     self . assertEquals( vec [25,1], 4.104385E-014)
912     self . assertEquals( vec [25,2], -1.456509E-013)
913
914
915 def test_readMappingFileFedem( self ):
916     dict = ModRed.readMappingFileFedem(self . _resourcesFolder + "\\beam.nas" +
"\\Fedem" + "\\MEQN.res")
917
918     dof = 1
919     eq = 607
920     self . assertEquals( eq, dict [dof])
921
922     dof = 2
923     eq = 608

```

```

924         self.assertEqual(eq, dict[dof])
925
926         dof = 630
927         eq = 258
928         self.assertEqual(eq, dict[dof])
929
930
931
932     def test_calculateGravityVector_Fedem_beam ( self ):
933
934         with open(self._resourcesFolder + "\\beam.nas" + "\\Fedem" + "\\gravVec.p") as
file:
935             correct = pickle.load( file )
936             with open(self._resourcesFolder + "\\beam.nas" + "\\Fedem" + "\\NOR.p") as file:
937                 NOR = pickle.load( file )
938                 with open(self._resourcesFolder + "\\beam.nas" + "\\Fedem" +
"\\identityMatrix.p") as file:
939                     ident = pickle.load( file )
940                     B = ModRed.readFedemVector(self._resourcesFolder + "\\beam.nas" + "\\Fedem" +
"\\Bmat.res")
941                     CST = la.Double.SparseMatrix(630, 24)
942                     CST.SetSubMatrix(0, 24, 0, 24, ident )
943                     CST.SetSubMatrix(24, 606, 0, 24, B)
944                     H = CST.Append(NOR)          # H = [CST NOR]
945
946                     # Building M_full:
947                     M_full = ModRed.readFedemMatrix(self._resourcesFolder + "\\beam.nas" +
"\\Fedem" + "\\M_full.res")
948                     dict = ModRed.readMappingFileFedem(self._resourcesFolder + "\\beam.nas" +
"\\Fedem" + "\\MEQN.res")
949                     eNodes = [1, 2, 3, 4]
950                     iNodes = [i for i in range(5, 106)]
951                     M_full = ModRed.partitionMatrix(M_full, eNodes, iNodes, dict , 6)
952
953                     gravVec = ModRed.calculateGravityVector(M_full, H, 4, 101, 6)
954                     print (gravVec)
955                     for row in range(0, gravVec.RowCount):
956                         for col in range(0, gravVec.ColumnCount):
957                             self.assertAlmostEqual(gravVec[row, col ], correct [row, col ], places=5)
958
959
960
961
962
963     def test_calculateGravityVectorReduced_beam ( self ):
964         with open(self._resourcesFolder + "\\beam.nas" + "\\Fedem" + "\\gravVec.p") as
file:
965             correct = pickle.load( file )
966

```

```

967     M_red = ModRed.readMMFMatrix(self._resourcesFolder + "\\beam.nas" + "\\Ansys" +
    "\\M_red.mmf", "sub")
968
969     gravVec = ModRed.calculateGravityVectorReduced(M_red, 4, 2)
970     diffs = []
971     for row in range(0, gravVec.RowCount):
972         for col in range(0, gravVec.ColumnCount):
973             diff = correct [row, col] - gravVec[row, col]
974             diffs .append(diff)
975             if ( diff > 1.0):
976                 self . fail ("Too large diff")
977
978
979 def test_calculateGravityVectorReduced_twoQUAD4(self):
980     with open(self . _resourcesFolder + "\\twoQUAD4" + "\\Fedem" + "\\gravVec.p")
    as file:
981         correct = pickle .load( file )
982
983     CST = ModRed.readMMFMatrix(self._resourcesFolder + "\\twoQUAD4" + "\\Ansys"
    + "\\CST.mmf", "full")
984     NOR = ModRed.readMMFMatrix(self._resourcesFolder + "\\twoQUAD4" + "\\Ansys"
    + "\\NOR.mmf", "full")
985     H = CST.Append(NOR)          # H = [CST NOR]
986
987     M_red = ModRed.readMMFMatrix(self._resourcesFolder + "\\twoQUAD4" +
    "\\Ansys" + "\\M_red.mmf", "sub")
988
989     gravVec = ModRed.calculateGravityVectorReduced(M_red, 2, 2)
990     print (gravVec)
991     diffs = []
992     for row in range(0, gravVec.RowCount):
993         for col in range(0, gravVec.ColumnCount):
994             diff = correct [row, col] - gravVec[row, col]
995             diffs .append(diff)
996     print ( diffs )
997     if (max(diffs)) > 20:
998         self . fail ("Too large diff")
999
1000
1001 def test_calculateGravityVectorReduced_plate ( self ):
1002     correct = ModRed.readFedemVector(self._resourcesFolder + "\\plate .nas" +
    "\\Fedem" + "\\gravVec.res")
1003
1004     M_red = ModRed.readMMFMatrix(self._resourcesFolder + "\\plate .nas" + "\\Ansys"
    + "\\M_red.mmf", "sub")
1005
1006     gravVec = ModRed.calculateGravityVectorReduced(M_red, 4, 2)
1007     print (gravVec)
1008     diffs = []
1009     for row in range(0, gravVec.RowCount):

```

```

1010         for col in range(0, gravVec.ColumnCount):
1011             diff = correct [row, col] - gravVec[row, col]
1012             diffs .append(diff)
1013
1014
1015         print ( diffs )
1016         print (max(diffs))
1017         if (max(diffs)) > 20:
1018             self . fail ("Too large diff")
1019
1020     def test_calculateGravityVector_Ansys_twoQUAD4( self ):
1021
1022         with open( self . _resourcesFolder + "\\twoQUAD4" + "\\Fedem" + "\\gravVec.p" )
1023         as file:
1024             correct = pickle . load( file )
1025             M_full = ModRed.readMMFMatrix(self._resourcesFolder + "\\twoQUAD4" +
1026             "\\Ansys" + "\\M_full.lumped.mmf", "full")
1027             CST = ModRed.readMMFMatrix(self._resourcesFolder + "\\twoQUAD4" + "\\Ansys"
1028             + "\\CST.mmf", "full")
1029             NOR = ModRed.readMMFMatrix(self._resourcesFolder + "\\twoQUAD4" + "\\Ansys"
1030             + "\\NOR.mmf", "full")
1031             H = CST.Append(NOR)          # H = [CST NOR]
1032
1033             dict = ModRed.readMappingFile(self._resourcesFolder + "\\twoQUAD4" +
1034             "\\Ansys" + "\\M_full.lumped.mapping")
1035             eNodes = [1, 3]
1036             iNodes = [2, 4, 5, 6]
1037             M_full = ModRed.partitionMatrix(M_full, eNodes, iNodes, dict , 6)
1038
1039             gravVec = ModRed.calculateGravityVector(M_full, H, 2, 4, 6)
1040             print (" calculated: " + str (gravVec))
1041             print (" correct: " + str ( correct ))
1042             for row in range(0, gravVec.RowCount):
1043                 for col in range(0, gravVec.ColumnCount):
1044                     self . assertAlmostEqual(gravVec[row, col], correct [row, col], places=1)
1045
1046
1047     def test_node2dof ( self ):
1048         dofDict = {
1049             "UX": 1,
1050             "UY": 2,
1051             "UZ": 3,
1052             "ROTX": 4,
1053             "ROTY": 5,
1054             "ROTZ": 6
1055         }
1056
1057         node = 1
1058         dofText = "UX"
1059         correctDof = 1

```

```

1055     dof = ModRed.node2dof(node, dofDict, 6, dofText)
1056     self.assertEqual(dof, correctDof)
1057
1058     node = 1
1059     dofText = "UY"
1060     correctDof = 2
1061     dof = ModRed.node2dof(node, dofDict, 6, dofText)
1062     self.assertEqual(dof, correctDof)
1063
1064     node = 1
1065     dofText = "UZ"
1066     correctDof = 3
1067     dof = ModRed.node2dof(node, dofDict, 6, dofText)
1068     self.assertEqual(dof, correctDof)
1069
1070     node = 1
1071     dofText = "ROTX"
1072     correctDof = 4
1073     dof = ModRed.node2dof(node, dofDict, 6, dofText)
1074     self.assertEqual(dof, correctDof)
1075
1076     node = 2
1077     dofText = "UY"
1078     correctDof = 8
1079     dof = ModRed.node2dof(node, dofDict, 6, dofText)
1080     self.assertEqual(dof, correctDof)
1081
1082     node = 3
1083     dofText = "UZ"
1084     correctDof = 15
1085     dof = ModRed.node2dof(node, dofDict, 6, dofText)
1086     self.assertEqual(dof, correctDof)
1087
1088
1089
1090 def test_calculateGravityVector_Fedem_twoQUAD4(self):
1091     with open(self._resourcesFolder + "\\twoQUAD4" + "\\Fedem" + "\\gravVec.p")
1092     as file:
1093         correct = pickle.load(file)
1094
1095         M_full = ModRed.readFedemMatrix(self._resourcesFolder + "\\twoQUAD4" +
1096         "\\Fedem" + "\\M_full.res")
1097         ident = la.Double.SparseMatrix.CreateIdentity(12)
1098         B = ModRed.readFedemVector(self._resourcesFolder + "\\twoQUAD4" + "\\Fedem"
1099         + "\\BMAT.res")
1100         nullMat = la.Double.SparseMatrix(12, 2)
1101         phi = ModRed.readFedemVector(self._resourcesFolder + "\\twoQUAD4" +
1102         "\\Fedem" + "\\phi.res")
1103         CST = la.Double.SparseMatrix(36, 12)
1104         CST.SetSubMatrix(0, 12, 0, 12, ident)
1105         CST.SetSubMatrix(12, 24, 0, 12, B)

```

```

1101     NOR = la.Double.SparseMatrix(36, 2)
1102     NOR.SetSubMatrix(0, 12, 0, 2, nullMat)
1103     NOR.SetSubMatrix(12, 24, 0, 2, phi)
1104     H = CST.Append(NOR)           # H = [CST NOR]
1105
1106     dict = ModRed.readMappingFileFedem(self._resourcesFolder + "\\twoQUAD4" +
1107     "\\Fedem" + "\\MEQN.res")
1108     eNodes = [1, 3]
1109     iNodes = [2, 4, 5, 6]
1110     M_full = ModRed.partitionMatrix(M_full, eNodes, iNodes, dict, 6)
1111
1112     gravVec = ModRed.calculateGravityVector(M_full, H, 2, 4, 6)
1113     print(gravVec)
1114     for row in range(0, gravVec.RowCount):
1115         for col in range(0, gravVec.ColumnCount):
1116             self.assertAlmostEqual(gravVec[row, col], correct[row, col], places=4)
1117
1118
1119
1120
1121 def test_calculateGravityVector_Fedem_unsortedENodes ( self ):
1122     """
1123     Gravity vector with eNodes = [22, 28, 6, 44]
1124     Using the beam.nas model
1125     """
1126     correct = ModRed.readFedemVector(self._resourcesFolder + "\\beam.nas" +
1127     "\\Fedem" + "\\unsrtENodes" + "\\gravVec.res")
1128
1129     B = ModRed.readFedemVector(self._resourcesFolder + "\\beam.nas" + "\\Fedem" +
1130     "\\unsrtENodes" + "\\Bmat.res")
1131     ident = la.Double.SparseMatrix(24). CreateIdentity (24)
1132     CST = la.Double.SparseMatrix(630, 24)
1133     CST.SetSubMatrix(0, 24, 0, 24, ident)
1134     CST.SetSubMatrix(24, 606, 0, 24, B)
1135
1136     phi = ModRed.readFedemVector(self._resourcesFolder + "\\beam.nas" + "\\Fedem"
1137     + "\\unsrtENodes" + "\\phi.res")
1138     nullMat = la.Double.SparseMatrix(24, 2)
1139     NOR = la.Double.SparseMatrix(630, 2)
1140     NOR.SetSubMatrix(0, 24, 0, 2, nullMat)
1141     NOR.SetSubMatrix(24, 606, 0, 2, phi)
1142     H = CST.Append(NOR)           # H = [CST NOR]
1143
1144     # Building M_full:
1145     M_full = ModRed.readFedemMatrix(self._resourcesFolder + "\\beam.nas" +
1146     "\\Fedem" + "\\unsrtENodes" + "\\M_full.res")
1147     dict = ModRed.readMappingFileFedem(self._resourcesFolder + "\\beam.nas" +
1148     "\\Fedem" + "\\unsrtENodes" + "\\MEQN.res")
1149     eNodes = [22, 28, 6, 44]

```

```

1145     iNodes = [i for i in range(1, 106)]
1146     for e in eNodes:
1147         iNodes.remove(e)
1148     M_full = ModRed.partitionMatrix(M_full, eNodes, iNodes, dict, 6)
1149
1150     gravVec = ModRed.calculateGravityVector(M_full, H, 4, 101, 6)
1151     print (gravVec)
1152
1153     diffs = []
1154     for row in range(0, gravVec.RowCount):
1155         for col in range(0, gravVec.ColumnCount):
1156             diff = correct [row, col] - gravVec[row, col]
1157             diffs.append(diff)
1158     print (diffs)
1159     print ("max:" + str (max(diffs)))
1160     for row in range(0, gravVec.RowCount):
1161         for col in range(0, gravVec.ColumnCount):
1162             self.assertAlmostEqual(gravVec[row, col], correct [row, col], places=5)
1163
1164
1165
1166 def test_calculateGravityVector_Ansys_beam ( self ):
1167     with open( self._resourcesFolder + "\\beam.nas" + "\\Fedem" + "\\gravVec.p") as
file:
1168         correct = pickle.load( file )
1169
1170         CST = ModRed.readMMFMatrix(self._resourcesFolder + "\\beam.nas" + "\\Ansys" +
"\\CST.mmf", "full")
1171         NOR = ModRed.readMMFMatrix(self._resourcesFolder + "\\beam.nas" + "\\Ansys" +
"\\NOR.mmf", "full")
1172         H = CST.Append(NOR)           # H = [CST NOR]
1173
1174         # Building M_full:
1175         M_full = ModRed.readMMFMatrix(self._resourcesFolder + "\\beam.nas" + "\\Ansys"
+ "\\M_full.lumped.mmf", "full")
1176         dict = ModRed.readMappingFile(self._resourcesFolder + "\\beam.nas" + "\\Ansys"
+ "\\M_full.lumped.mapping")
1177         eNodes = [1, 2, 3, 4]
1178         iNodes = [i for i in range(5, 106)]
1179         M_full = ModRed.partitionMatrix(M_full, eNodes, iNodes, dict, 6)
1180
1181         gravVec = ModRed.calculateGravityVector(M_full, H, 4, 101, 6)
1182
1183         diffs = []
1184         for row in range(0, gravVec.RowCount):
1185             for col in range(0, gravVec.ColumnCount):
1186                 diff = correct [row, col] - gravVec[row, col]
1187                 diffs.append(diff)
1188         print (diffs)
1189         print ("max:" + str (max(diffs)))

```

```
1190         for row in range(0, gravVec.RowCount):
1191             for col in range(0, gravVec.ColumnCount):
1192                 self.assertAlmostEqual(gravVec[row, col], correct[row, col], places=1)
1193
1194
1195
1196     def test_readMappingFile ( self ):
1197
1198         path = self._resourcesFolder + "\\twoQUAD4" + "\\Ansys" + "\\M.full.mapping"
1199         dict = ModRed.readMappingFile(path) # dict[dof] = equation
1200
1201         self.assertEqual ( dict [1], 13)
1202         self.assertEqual ( dict [2], 14)
1203         self.assertEqual ( dict [7], 19)
1204         self.assertEqual ( dict [13], 31)
1205         self.assertEqual ( dict [14], 32)
1206         self.assertEqual ( dict [18], 36)
1207
1208
1209
1210 if __name__ == "__main__":
1211     unittest.main()
```

